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Electrical Engineering

5th Year

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Control Systems (2)

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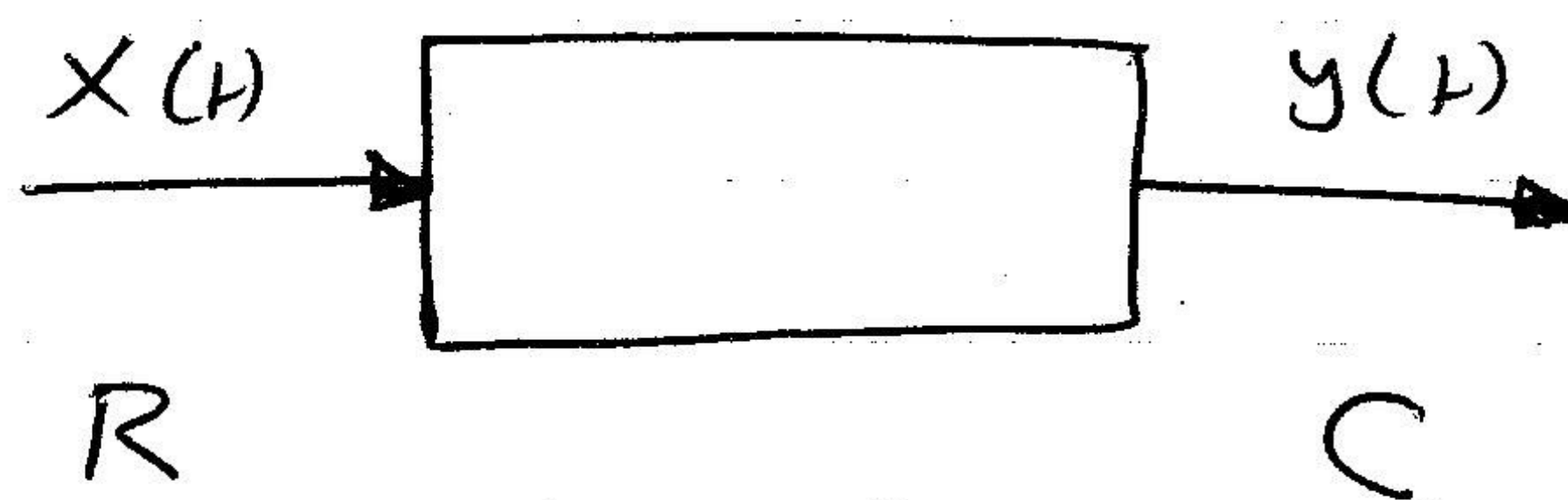
1. 1st Order & 2nd Order Systems

First order systems: - the system that its response varies with the change of its response.

and described by the following derivative equation:-

$$\tau y' + y = kx$$

$$y' = \frac{dy}{dt}$$



$$\tau \dot{y} + y = kx$$

τ : delay time (Time constant).

$y(t)$: output-response.

$x(t)$: input

k : static gain (DC gain)

$$\tau Y(s) s + Y(s) = k X(s)$$

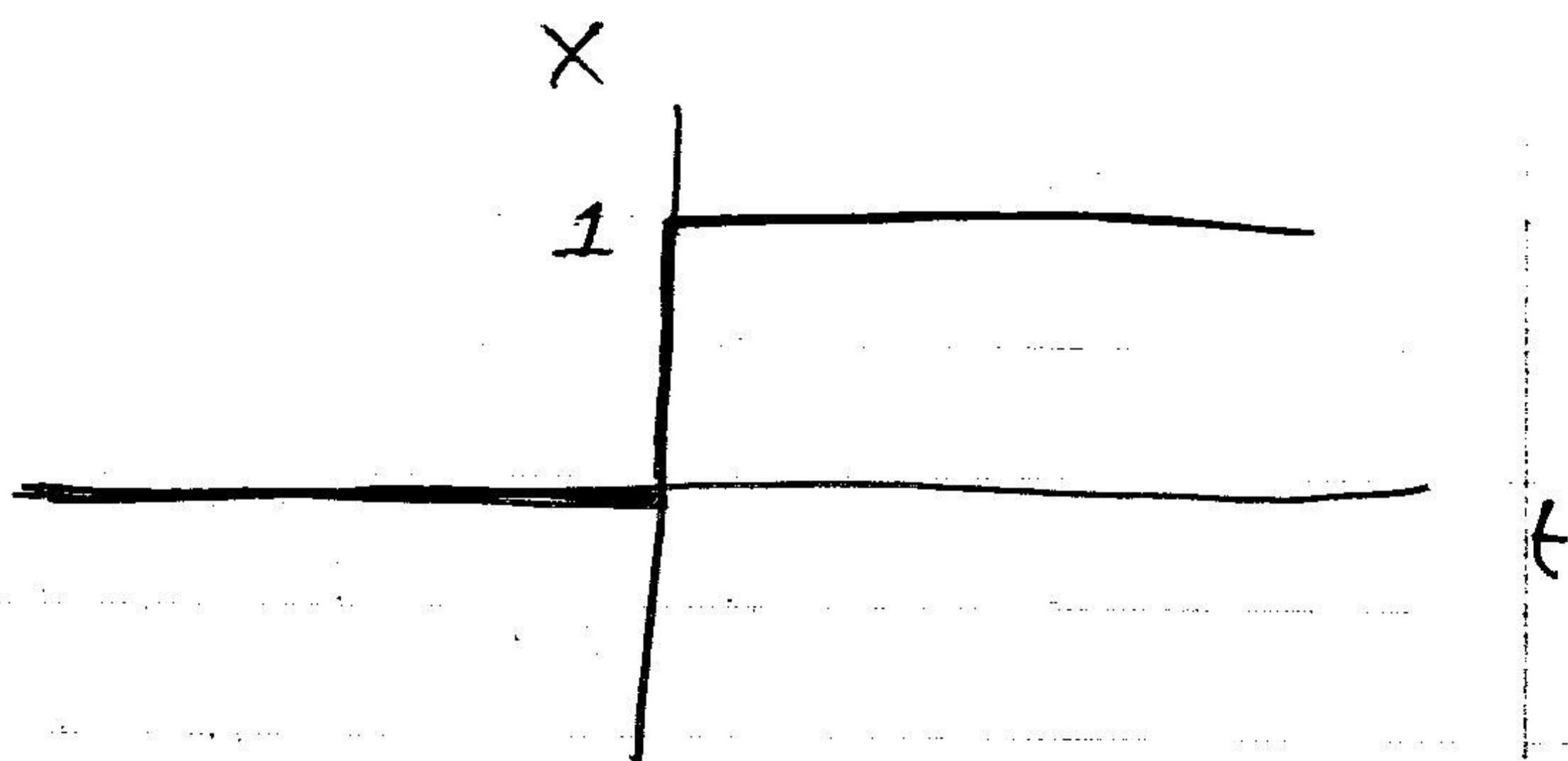
$$Y(s) [\tau s + 1] = k X(s)$$

$$\text{Transfer Function} = \frac{\text{output}}{\text{input}} = \frac{Y(s)}{X(s)} = \frac{k}{\tau s + 1}$$

$$Y(s) = \frac{k}{\tau s + 1} X(s)$$

$$X(t) = \begin{cases} 1 & t > 0 \\ 0 & t < 0 \end{cases}$$

Step input



* If $x(t) = 1$ for $t > 0$

$$X(s) = \frac{1}{s}$$

$$Y(s) = \frac{k}{\tau s' + 1} * \frac{1}{s'}$$

$$\frac{k}{\tau s' + 1} * \frac{1}{s'} = \left(\frac{a}{\tau s' + 1} + \frac{b}{s} \right) * s$$

Multiply by (s') then Made $s=0$

$$\frac{as}{\tau s' + 1} + b \Rightarrow \frac{k}{\tau s' + 1} = b \Rightarrow \boxed{b = k}$$

$$\frac{k}{\tau s' + 1} \Rightarrow \frac{k}{s} = a \Rightarrow a = -k\tau$$

$$\downarrow$$

$$\boxed{s = -\frac{1}{\tau}}$$

$$\therefore Y(s) = \frac{-k\tau}{\tau s' + 1} + \frac{k}{s}$$

$$Y(s) = k \left[\frac{1}{s} + \frac{\tau}{\tau s' + 1} \right]$$

$$\mathcal{L}^{-1} Y(s) = k [1 - e^{-t/\tau}]$$

$$\frac{\tau}{Ts+1} = \frac{1}{s + \frac{1}{\tau}} = e^{-t/\tau}$$

\therefore Response equation for unit step input:

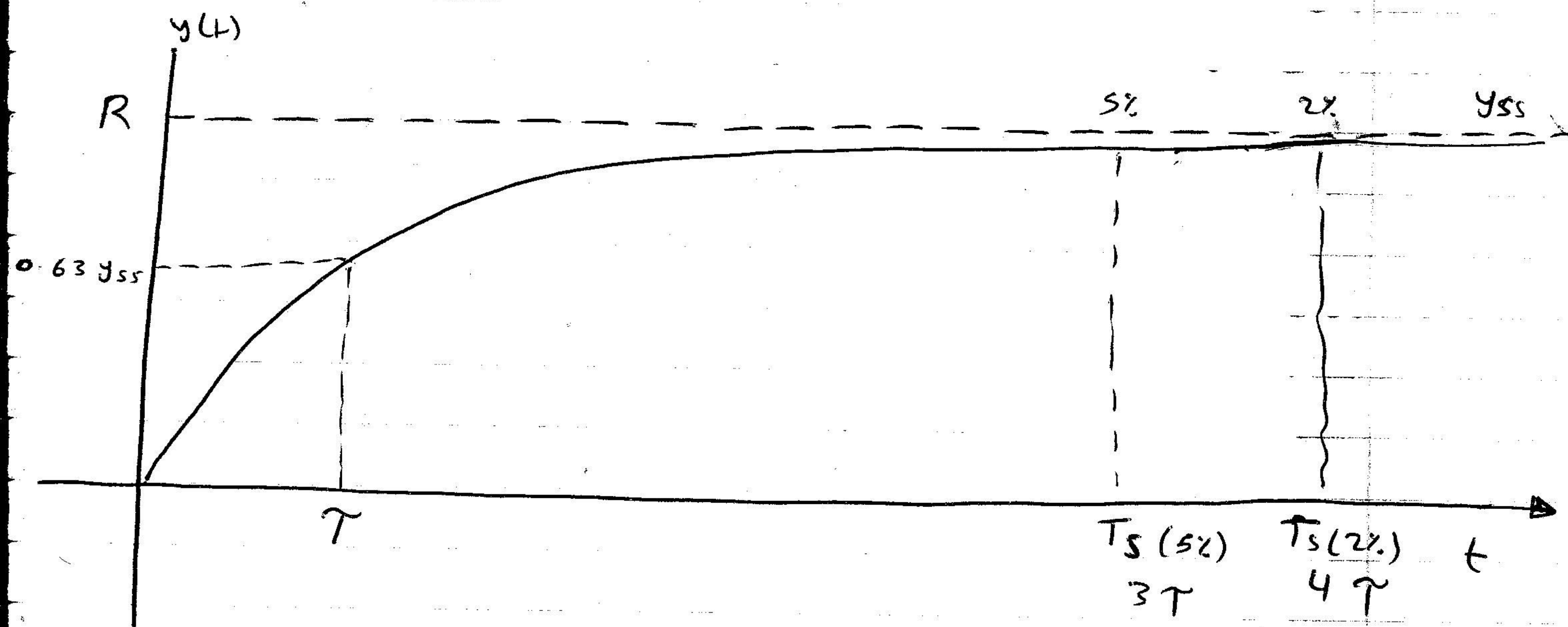
$$Y(t) = k [1 - e^{-t/\tau}]$$

$$e^{at} = \frac{1}{s-a}$$

$$e^{-at} = \frac{1}{s+a}$$

$$Y(t) = KR [1 - e^{-t/\tau}]$$

For unit step input $X(t) = R$



$$\text{Static error} = R(t) - y(t)$$

$t = \infty$

y_{ss} Final Value of response given at $t = \infty$

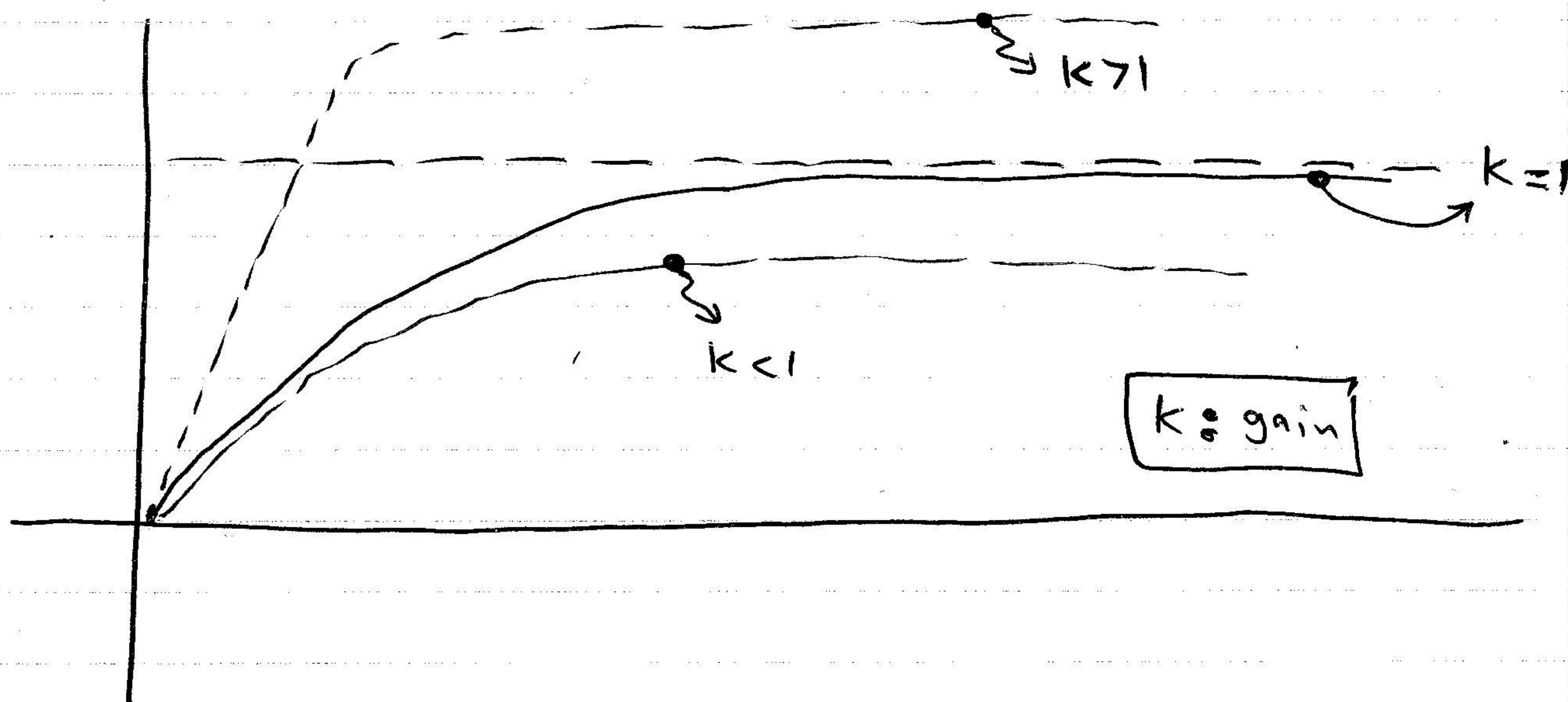
$$y(t) = KR(1 - e^{-t/\tau})$$

$$y(\infty) = KR$$

$$Y(s) = \frac{k}{\tau s + 1} * \frac{R}{s}$$

$$Y(\infty) = \lim_{s \rightarrow 0} s Y(s)$$

$$= \lim_{s \rightarrow 0} s \frac{kR}{\tau s + 1} * \frac{1}{s} = \boxed{kR}$$



$$y(\tau) = kR(1 - e^{-\frac{\tau}{\tau}}) = kR(1 - 0.37) \\ = \boxed{0.63 kR}$$

2%
5% } → Criteria

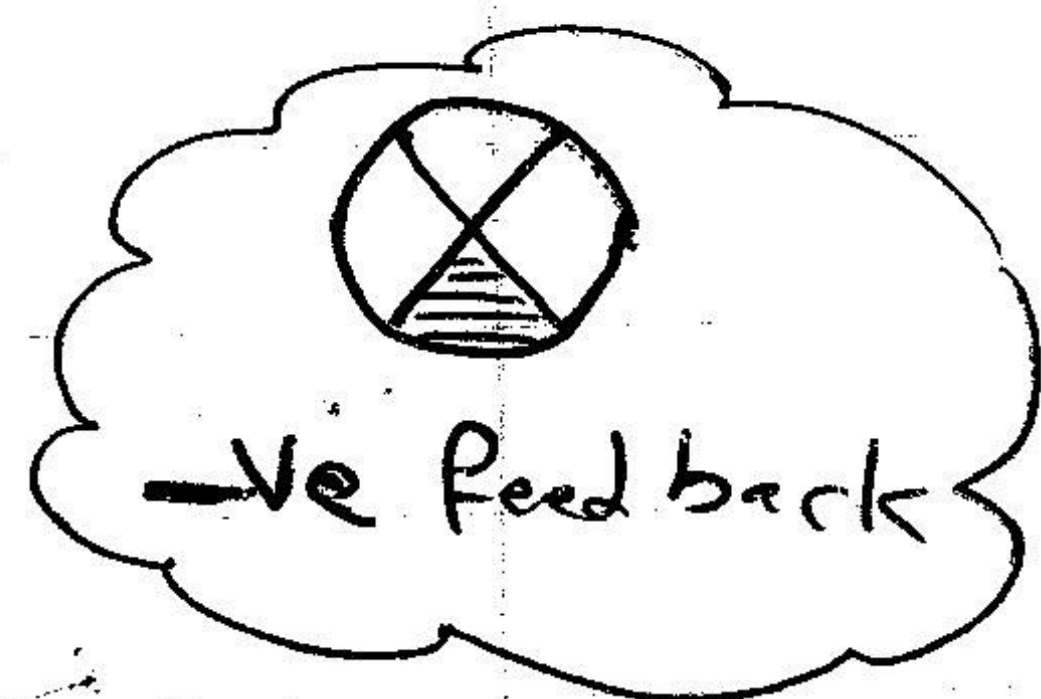
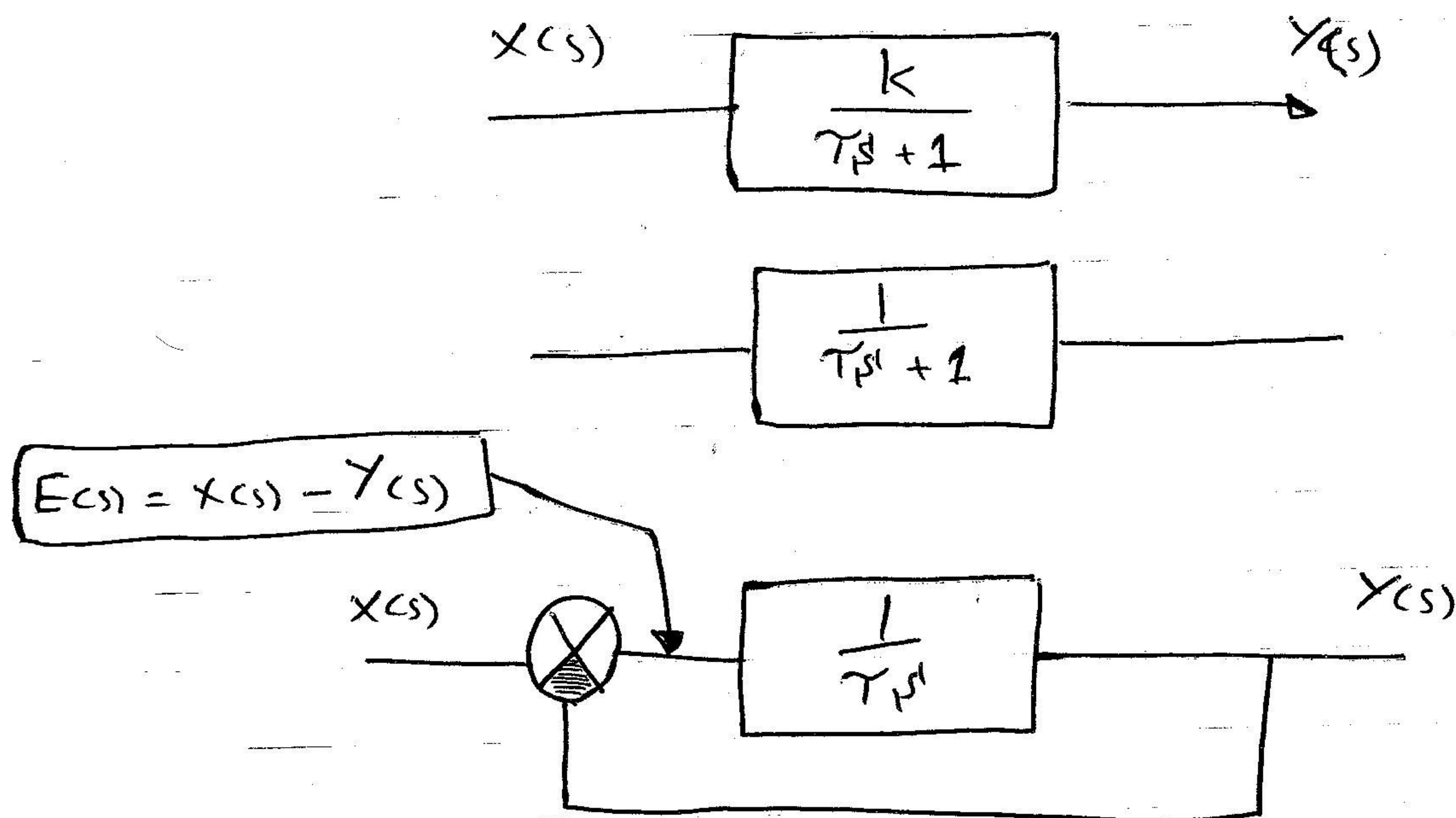
Settling time (T_s)

at which the system is said to be stable, and use two criterias 2%, 5%

$$T_s \Big|_{5\%} = 3\tau$$

$$T_s \Big|_{2\%} = 4\tau$$

1st Order system



$$e(t) = x(t) - y(t)$$

$$e(t) \% = \frac{x(t) - y(t)}{x(t)} \times 100 \%$$

$$y(t) \% = \frac{y(t)}{x(t)} \times 100 \%$$

$$e(t) \% = 1 - y(t) \%$$

$$y(t) \% = 1 - e(t) \%$$

$$\begin{aligned} e(t) &= x(t) - y(t) \\ e(t) &= R - kR(1 - e^{-t/\tau}) \quad , k=1 \\ e(t) &= R e^{-t/\tau} \end{aligned}$$

t	0	τ	2τ	3τ	4τ	∞
y %	0%	63%	86%	95%	98%	100%
y	0	$0.63 y_{ss}$	$0.86 y_{ss}$	$0.95 y_{ss}$	$0.98 y_{ss}$	$y_{ss} = 1$
e %	100%	37%	14%	5%	2%	0%
e	$y_{ss} = 1$	$0.37 y_{ss}$	$0.14 y_{ss}$	$0.05 y_{ss}$	$0.02 y_{ss}$	0

$$(*) e = R e^{-t/\tau}$$

$$\therefore e\% = \frac{e(t)}{R} \times 100\% = \boxed{e^{-t/\tau}}$$

$$e(3\tau) = e^{-\frac{3\tau}{\tau}} = e^{-3} = 0.0497 \approx \boxed{0.05}$$

$$e(4\tau) = e^{-4} = 0.0183 \approx \boxed{0.02}$$

$$e(2\tau) = e^{-2} \approx \boxed{0.14}$$

and so on --

(*) Finding τ

$$① G(s) = \frac{k}{\tau s + 1}$$

or

$$\tau \dot{y} + y = X(t)$$

$$\underline{\text{ex}} \quad G(s) = \frac{7}{5s + 1} \quad \therefore k = 7$$

$$\tau = 5$$

$$\underline{\text{ex}} \quad \frac{6}{s+3} \Rightarrow \text{divide on } \underline{3} \Rightarrow \frac{\frac{6}{3}}{\frac{s}{3} + 1} = \frac{2}{\frac{1}{3}s + 1}$$

$$\therefore k = 2$$

$$\tau = \frac{1}{3}$$

$$\underline{\text{ex}} \quad y(t) = kR(1 - e^{-t/\tau})$$

$$y(t) = 7(1 - e^{-5t})$$

$$k = 7$$

$$\tau = \frac{1}{5} = \boxed{0.2}$$

⑥

hw $y(t) = (3 - 2e^{-t/6})$

ex $2y + 7\dot{y} = 3$

divide on 2

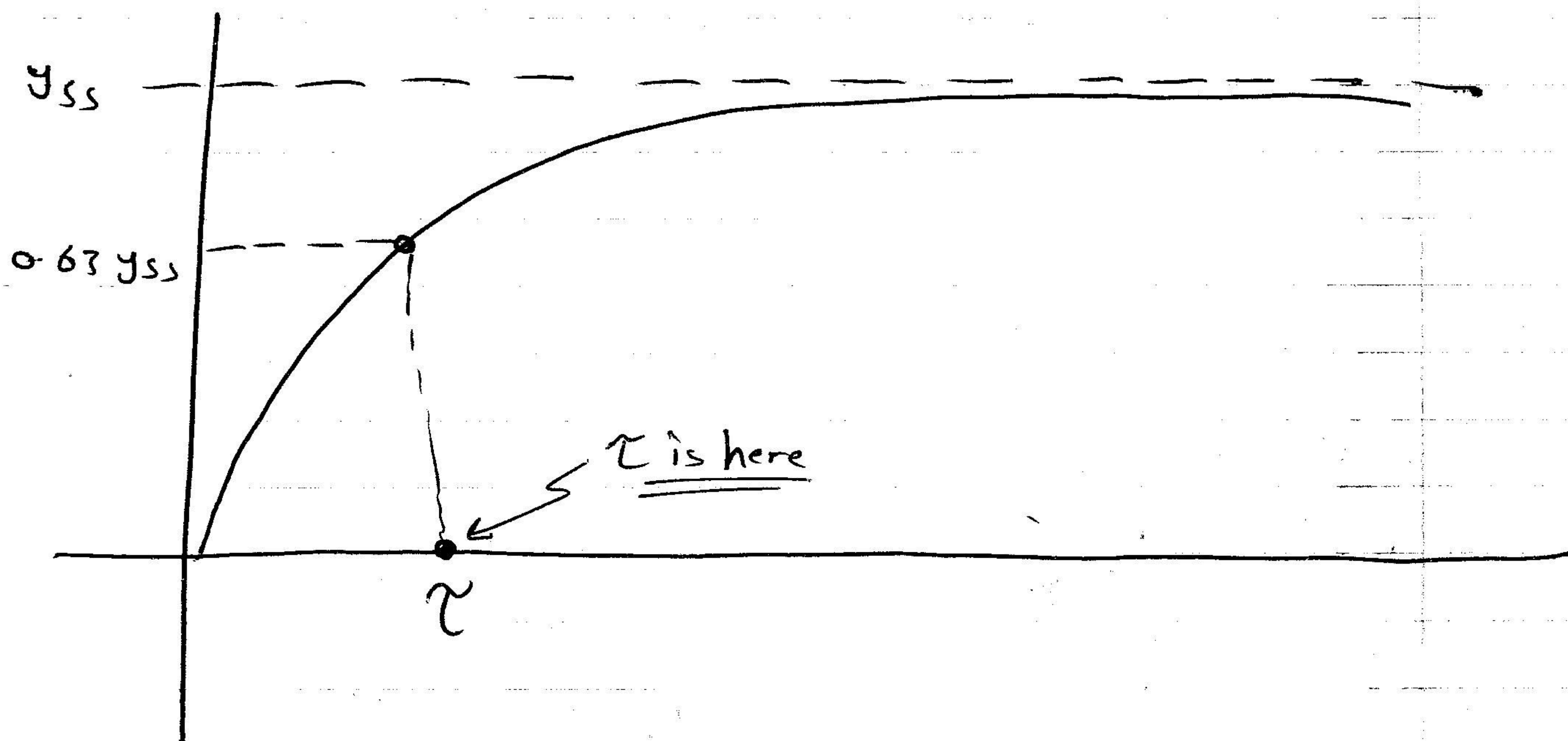
$$y + 3.5\dot{y} = \frac{3}{2}$$

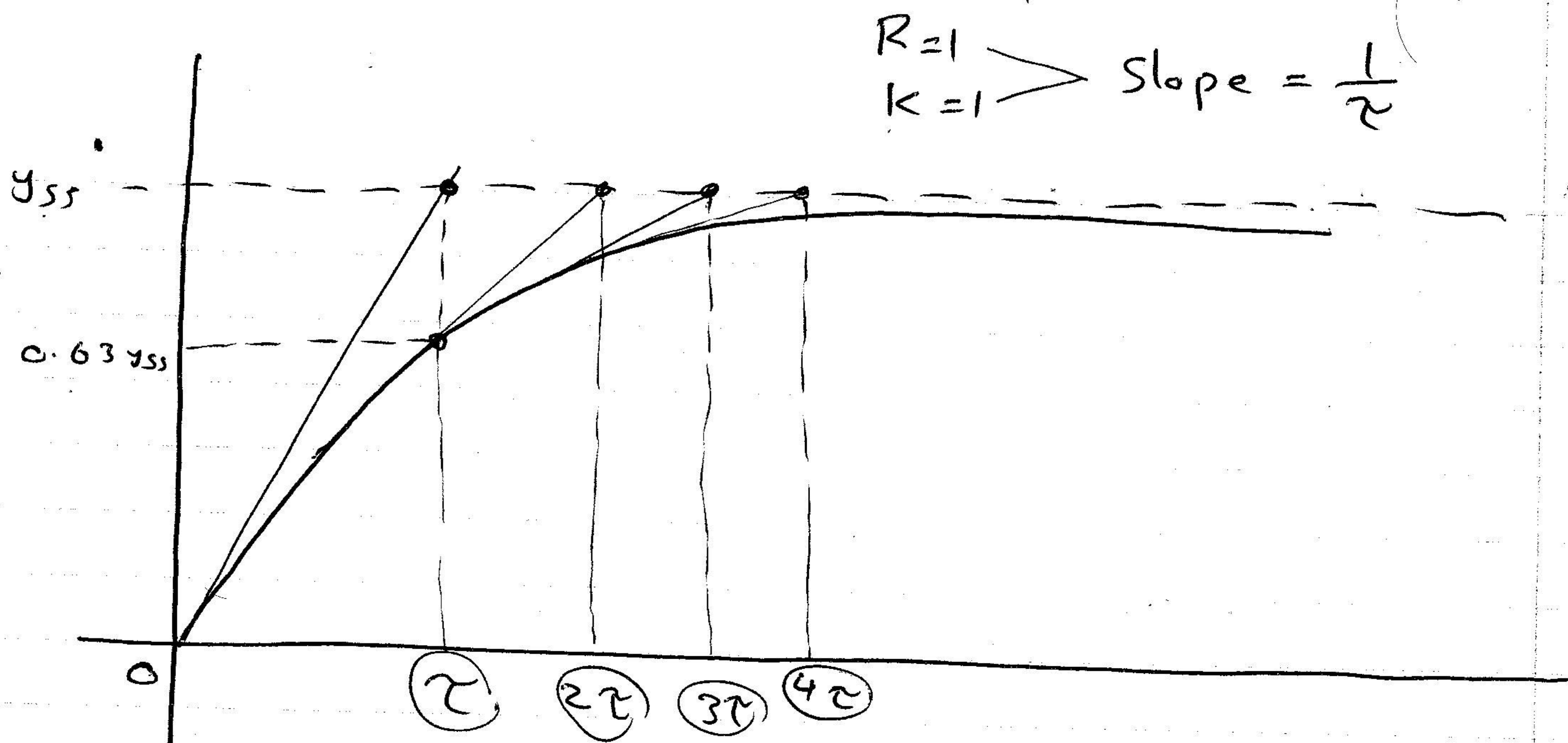
$$\therefore \tau = \boxed{3.5}$$

$$\therefore \text{If input} = 1 \quad \therefore K = \frac{3}{2}$$

not important

2 By graph





Plot tangent for the system near (0), when the tangent intersects with y_{ss} , the corresponding point on the x-axis, will be τ (Slope = $\frac{1}{\tau}$)

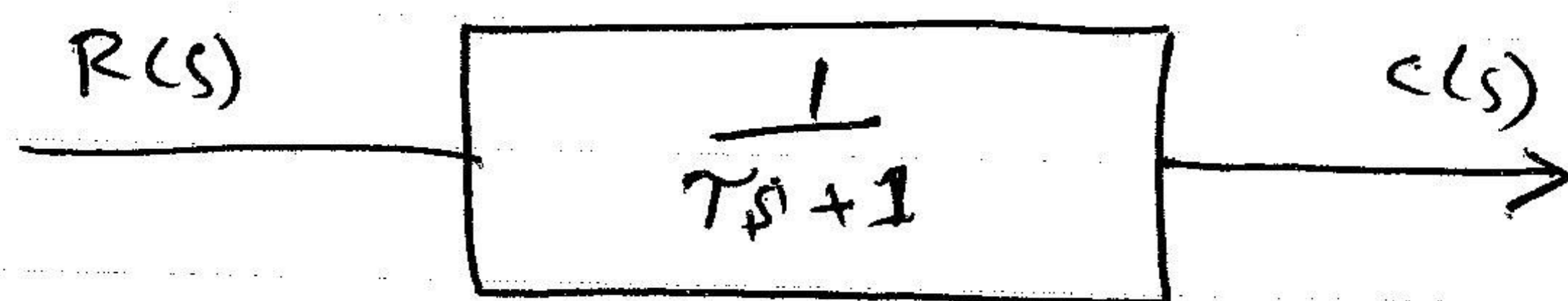
$$y = y_{ss} (1 - e^{-t/\tau})$$

$$\dot{y} = y_{ss} \left(\frac{1}{\tau} * e^{-t/\tau} \right) = \frac{y_{ss}}{\tau} e^{-t/\tau} = \boxed{\frac{y_{ss}}{\tau} * t}$$

For ramp input

$$r(t) = t$$

$$R(s) = \frac{1}{s^2}$$



$$C(s) = \frac{1}{\tau s + 1} * \frac{1}{s^2} = \frac{a}{\tau s + 1} + \frac{b}{s} + \frac{c}{s^2}$$

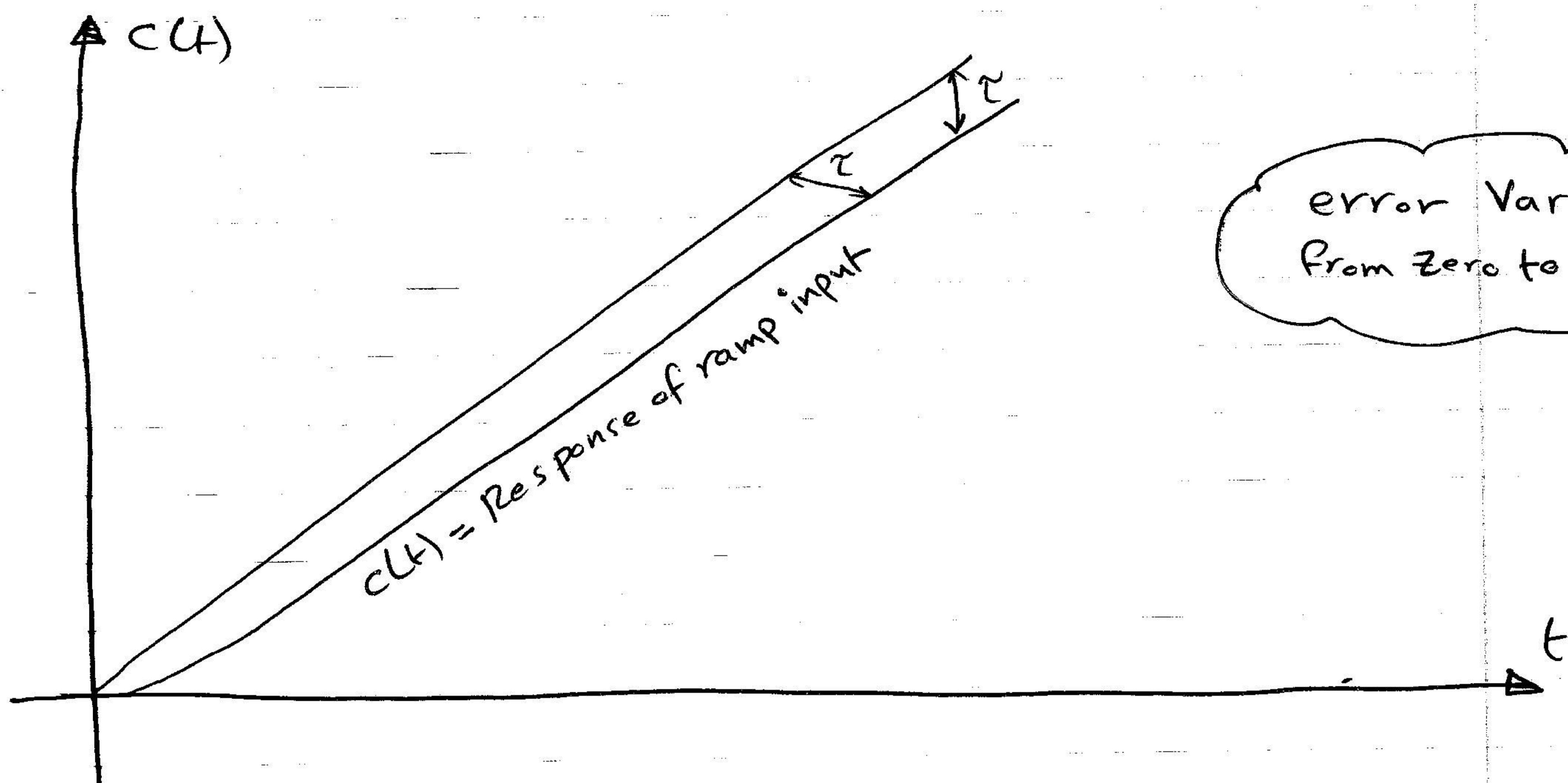
$$= \frac{\tau^2}{\tau s + 1} - \frac{\tau}{s} + \frac{1}{s^2}$$

$$\therefore C(t) = t - \tau + \tau e^{-\frac{t}{\tau}}$$

$$\begin{aligned} e(t) &= R(t) - C(t) = t - [t - \tau + \tau e^{-\frac{t}{\tau}}] \\ &= t - t + \tau - \tau e^{-\frac{t}{\tau}} \\ &= \tau - \tau e^{-\frac{t}{\tau}} \end{aligned}$$

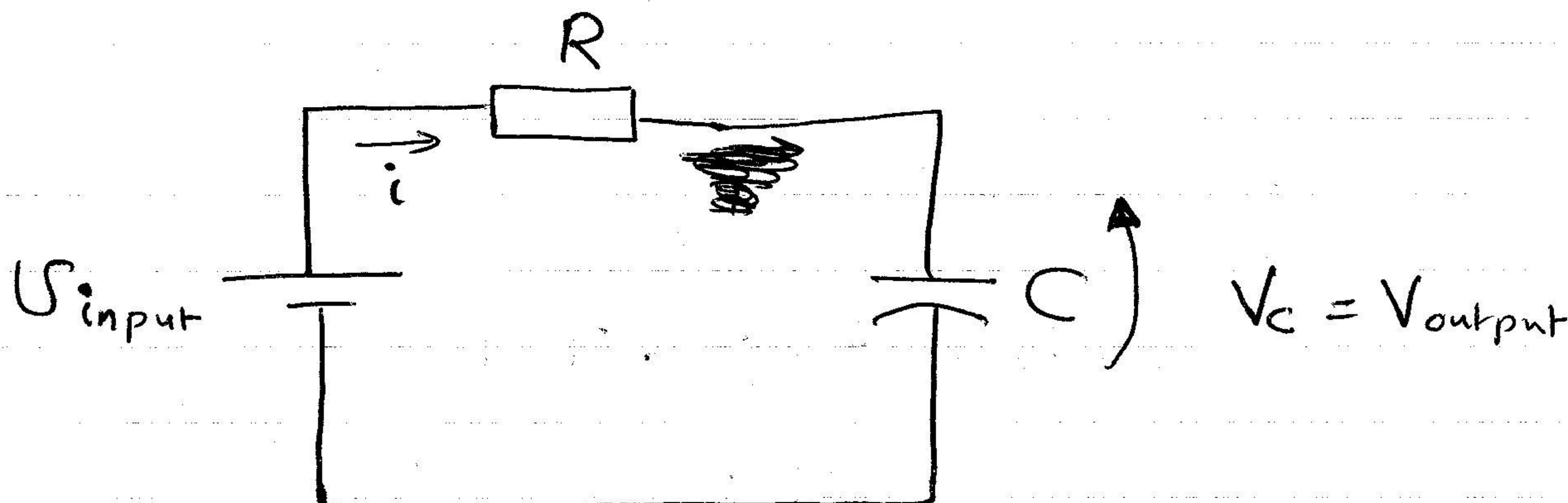
$$\therefore e(t) = \tau [1 - e^{-\frac{t}{\tau}}]$$

$$e_{ss} = \tau$$



Variable = small letter (i)
Constant = Capital letter (I)

Ex1: Charging Capacitor



$$V_i = V_R + V_C$$

$$V_i = IR + \frac{1}{C} \int i dt$$

$$V_o = \frac{1}{C} \int i dt$$

$$\dot{V}_o = \frac{i}{C} \Rightarrow i = C \dot{V}_o$$

$$V_i = RC \dot{V}_o + V_o$$

$$IR = C \dot{V}_o R$$

but standard equation = $\tau \dot{y} + y = kx$

$$\rightarrow \tau \dot{y} + y = kx$$

$$\rightarrow RC \dot{V}_o + V_o = V_i$$

$$\therefore \tau = RC$$

$$k = 1$$

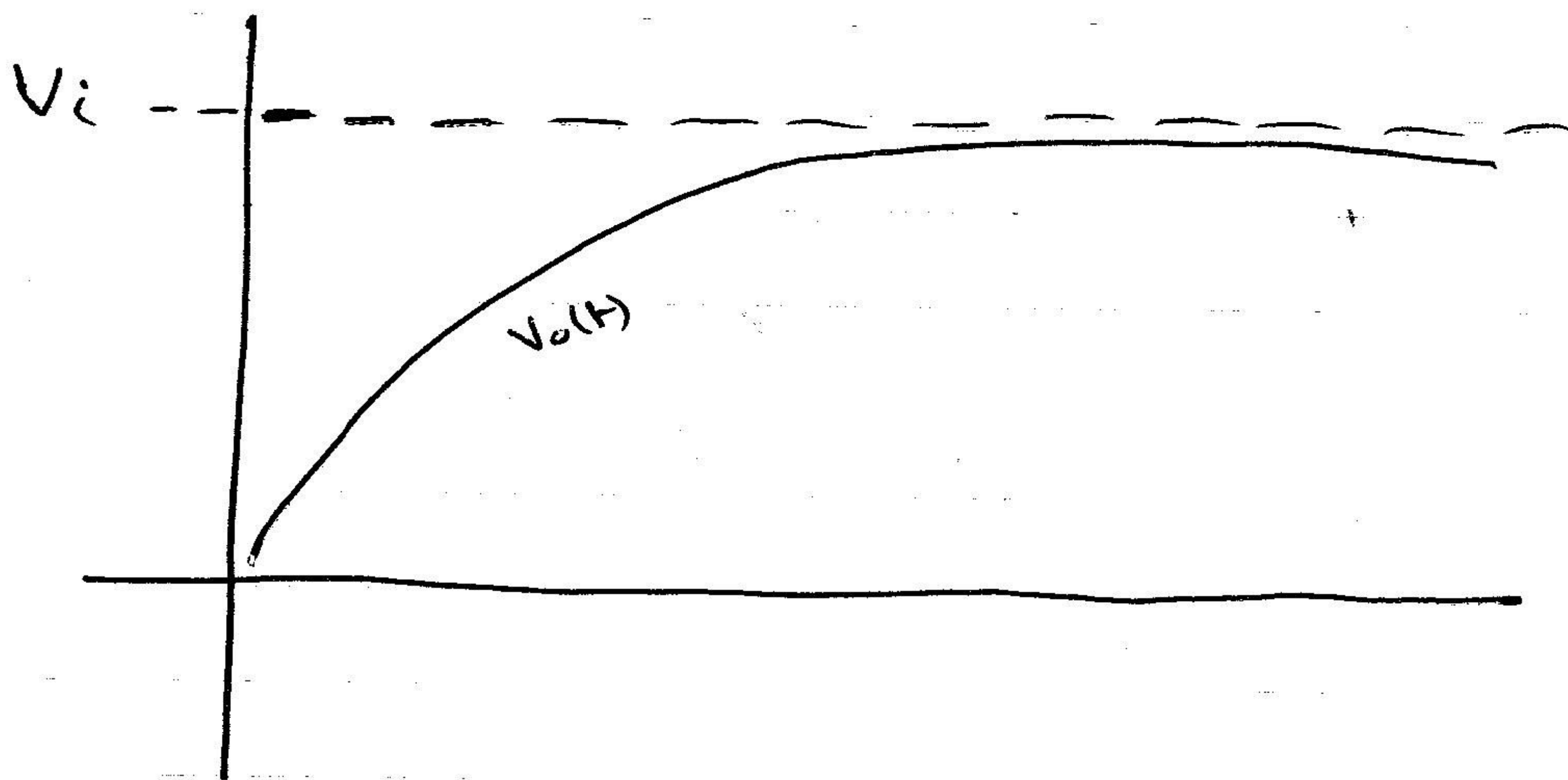
$$V_I(s) = RCS V_o(s) + V_o(s)$$

$$V_I(s) = V_o(s) [RCS + 1]$$

$$\frac{V_o(s)}{V_i(s)} = \frac{1}{RCs + 1}$$

⇒ ⊗ for step input

$$V_o(t) = V_i (1 - e^{-t/RC})$$



Ex 2 : Tank filling

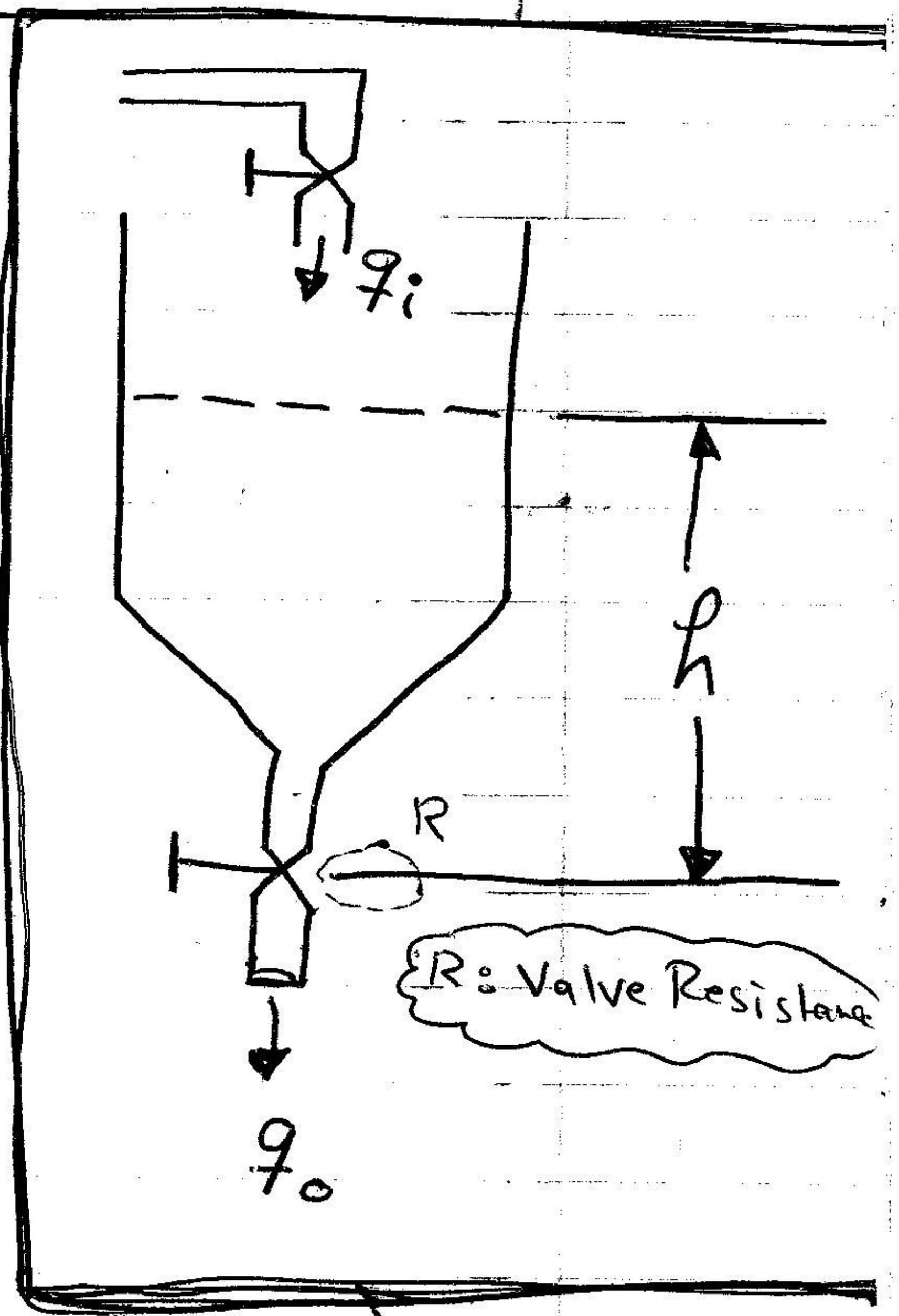
$$\text{Volumetric flow rate} = q_i - q_o$$

↑
(Volume/s)
↓

$$h = \frac{1}{A} \int q_i - q_o \cdot dt \Rightarrow$$

$$\dot{h} = \frac{1}{A} (q_i - q_o)$$

but $q_o = \frac{P}{R} = \frac{\rho g h}{R}$



standard equation
 $\tau \dot{y} + y = kx$

$$\therefore \frac{1}{A} \left(q_i - \frac{\rho g}{R} h \right) = \dot{h}$$

$$\frac{1}{A} q_i - \frac{\rho g}{RA} h = \dot{h}$$

$$\left(\dot{h} + \frac{\rho g}{RA} h = \frac{1}{A} q_i \right) \times \frac{RA}{\rho g}$$

$$\begin{array}{ccc} \frac{RA}{\rho g} \dot{h} + h = \frac{R}{\rho g} q_i \\ \downarrow \quad \uparrow \quad \uparrow \quad \downarrow \\ \tau \dot{y} + y = kx \end{array}$$

$$\therefore \tau = \frac{RA}{\rho g}$$

$$k = \frac{R}{\rho g}$$

Standard

$$\therefore \frac{\text{Output}}{\text{Input}} = \frac{h}{q_i} = \frac{k}{\tau s + 1} = \frac{\frac{R}{\rho g}}{\frac{RA}{\rho g} s + 1}$$

$$\therefore h(t) = \frac{R}{\rho g} q_i \left(1 - e^{-\frac{\rho g}{RA} t} \right)$$

2nd Order Systems

2nd order system is the system that its output rates with the change of its derivative.

and can be expressed by the following derivative equations

~~scribbled out~~

$$\ddot{y} + 2\zeta\omega_n\dot{y} + \omega_n^2 y = k\omega_n^2 x$$

⊗ k : DC gain.

⊗ $\omega_n = \omega_0$: Natural Frequency.

⊗ ζ : damping ratio

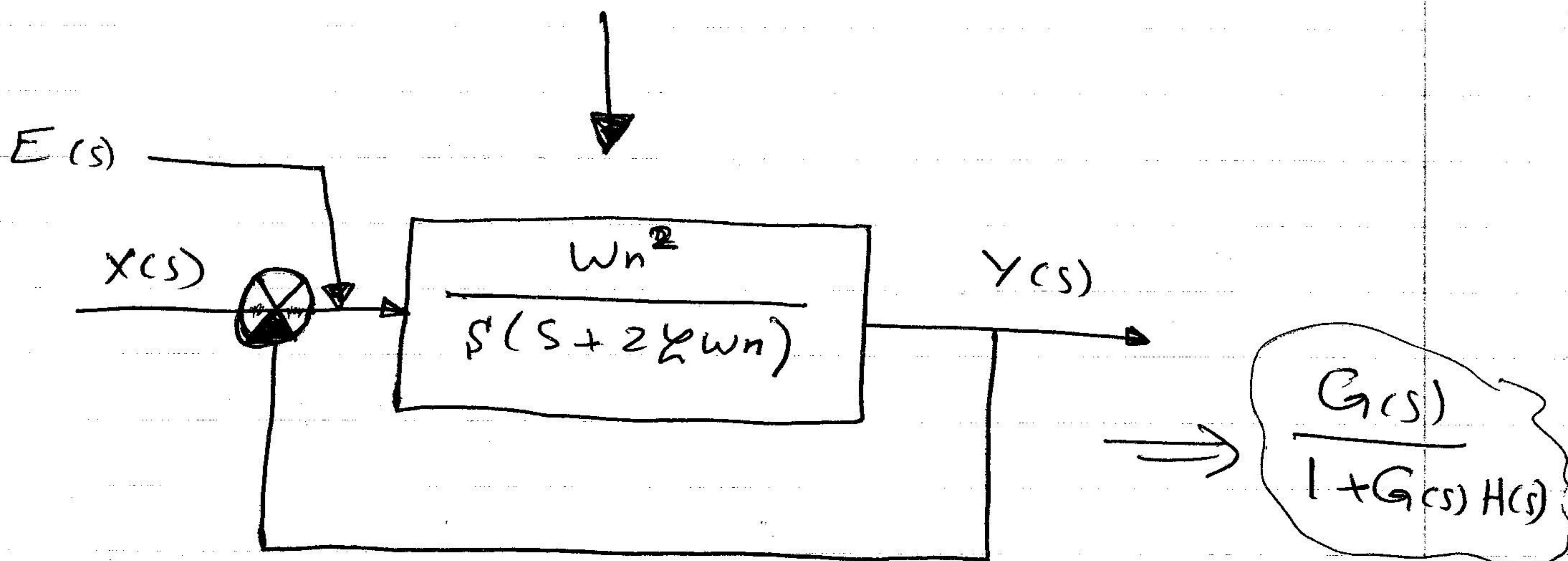
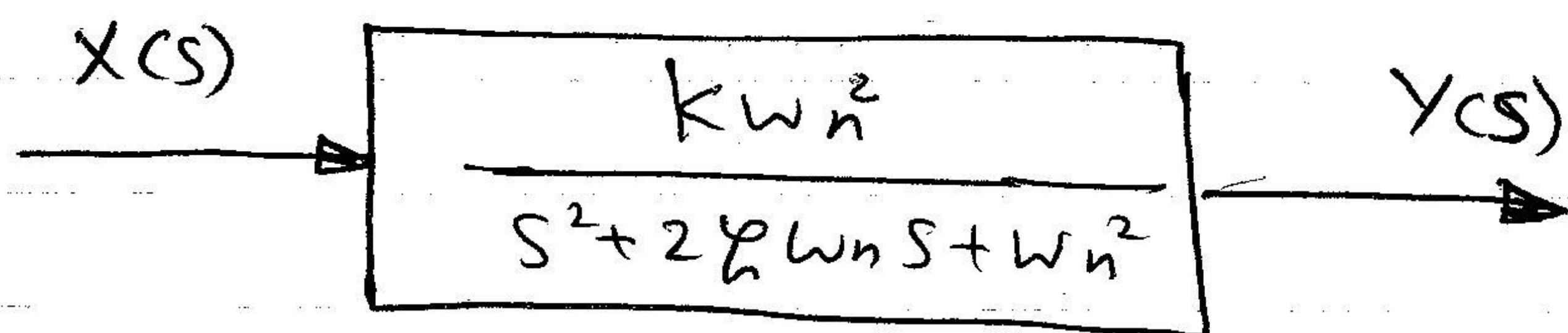
⊗ $\alpha = 2\zeta$: damping factor

ω = omega
 ζ = Zeta

$$s^2 Y(s) + 2\zeta\omega_n s Y(s) + \omega_n^2 Y(s) = k\omega_n^2 X(s)$$

$$Y(s) [s^2 + 2\zeta\omega_n s + \omega_n^2] = k\omega_n^2 X(s)$$

$$\frac{Y(s)}{X(s)} = \frac{k\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$



$$\frac{\omega_n^2}{s(s + 2\zeta\omega_n)} \quad \begin{matrix} \nearrow * \\ \searrow \end{matrix} \quad \frac{s(s + 2\zeta\omega_n)}{s(s + 2\zeta\omega_n)}$$

$$1 + \frac{\omega_n^2}{s(s + 2\zeta\omega_n)}$$

$$= \frac{\omega_n^2}{s(s + 2\zeta\omega_n) + \omega_n^2}$$

$$= \boxed{\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}}$$

~~Unit~~ $x(t) = \text{Unit step input} = \begin{cases} 1 & t > 0 \\ 0 & t < 0 \end{cases}$

$$Y(s) = \frac{K\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} X(s)$$

$$\Rightarrow \text{For } X(s) = \frac{1}{s}$$

$$\Rightarrow Y(s) = \frac{K\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} * \frac{1}{s}$$

CASE 1 $\zeta > 1$ The system is called **OVER DAMPED**

There is two poles for $\frac{Y(s)}{X(s)}$, the poles are -ve and real

$$s_{1,2} = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a}$$

$$= \frac{-2\zeta\omega_n \pm \sqrt{(2\zeta\omega_n)^2 - 4\omega_n^2}}{2}$$

$$= -\zeta\omega_n \pm \sqrt{\zeta^2\omega_n^2 - \omega_n^2}$$

$$= -\zeta\omega_n \pm \omega_n \sqrt{\zeta^2 - 1}$$

الجزء الحقيقي سالب
negative Real

⊛ Real because $\zeta > 1 \Rightarrow \sqrt{\zeta^2 - 1} > 1 \therefore$ No imaginary parts

$$\zeta\omega_n \square \omega_n \sqrt{\zeta^2 - 1}$$

$$\zeta^2\omega_n^2 \square \omega_n^2\zeta^2 - \omega_n^2$$

⊛ \therefore negative

$$\Rightarrow y(t) = 1 + \frac{\omega_n}{2\sqrt{\zeta^2 - 1}} \left(\frac{e^{-s_1 t}}{s_1} - \frac{e^{-s_2 t}}{s_2} \right)$$

جواب
best

Time domain Response

In Case S_2 is too small and very close to zero

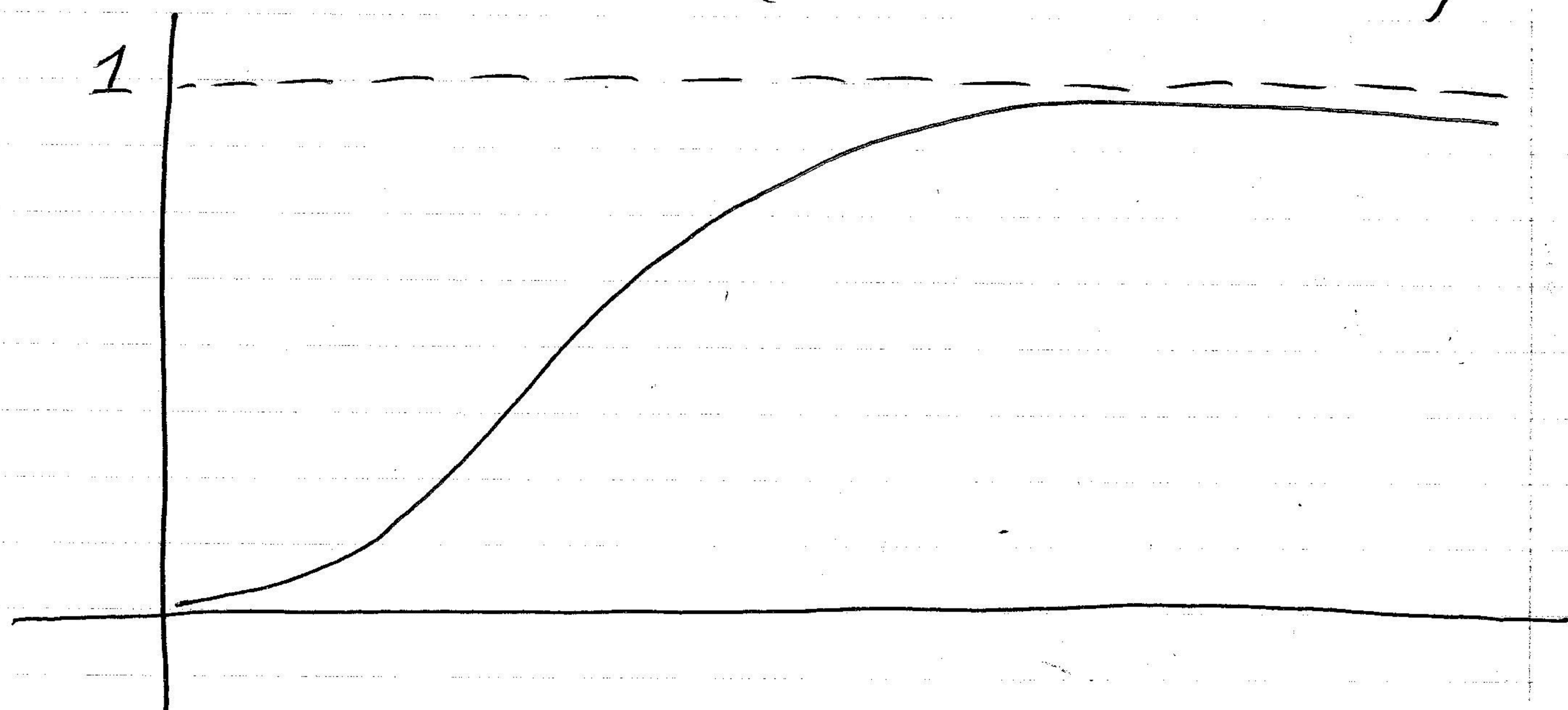
⊛ $C(t)$ can be written as

$$C(t) = 1 - e^{-(\zeta - \sqrt{\zeta^2 - 1})\omega_n t}$$

⊛ For input $X(s) = R$ & Gain = k

$$C_n(t) = kRC(t)$$

$$= kR \left(1 - e^{-(\zeta - \sqrt{\zeta^2 - 1})\omega_n t} \right)$$



CASE 2 & **CASE 3** will be discussed
after **EID AL-ADHA**

Regards

Sun 20/10/2013

CASE 2 Critically damped oscillation ($\zeta = 1$)

$$s_{1,2} = (\zeta \pm \sqrt{\zeta^2 - 1}) \omega_n \Rightarrow \boxed{\zeta = 1}$$

$$s_{1,2} = -\omega_n$$

$$C(s) = \frac{\omega_n^2}{(s + \omega_n)^2 s} = \frac{\omega_n^2}{s^2 + 2\omega_n s + \omega_n^2} * \frac{1}{s}$$

~~Partial fraction expansion~~

$$\mathcal{L}^{-1} C(s) = \frac{\omega_n^2}{(s + \omega_n)^2 s} = \frac{\omega_n^2}{s^2 + 2\omega_n s + \omega_n^2} * \frac{1}{s}$$

~~$C(t) = \frac{1}{\omega_n} e^{-\omega_n t} \sin(\omega_n t + \phi)$~~

~~$\phi = \tan^{-1} \frac{\omega_n}{\omega_n}$~~

$$\Rightarrow C(t) = 1 - e^{-\omega_n t} (1 + \omega_n t)$$

⇒ CASE 2 equation

CASE 3 Underdamped Oscillation

$$\zeta < 1$$

There are two-pole Conjugate

$$C(t) = 1 - \frac{e^{-\zeta \omega_n t}}{\sqrt{1-\zeta^2}} \sin(\omega_d t + \theta)$$

$$\theta = \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta}$$

$$\sin t + \cos t = \sqrt{2} \sin(t + \pi/4)$$

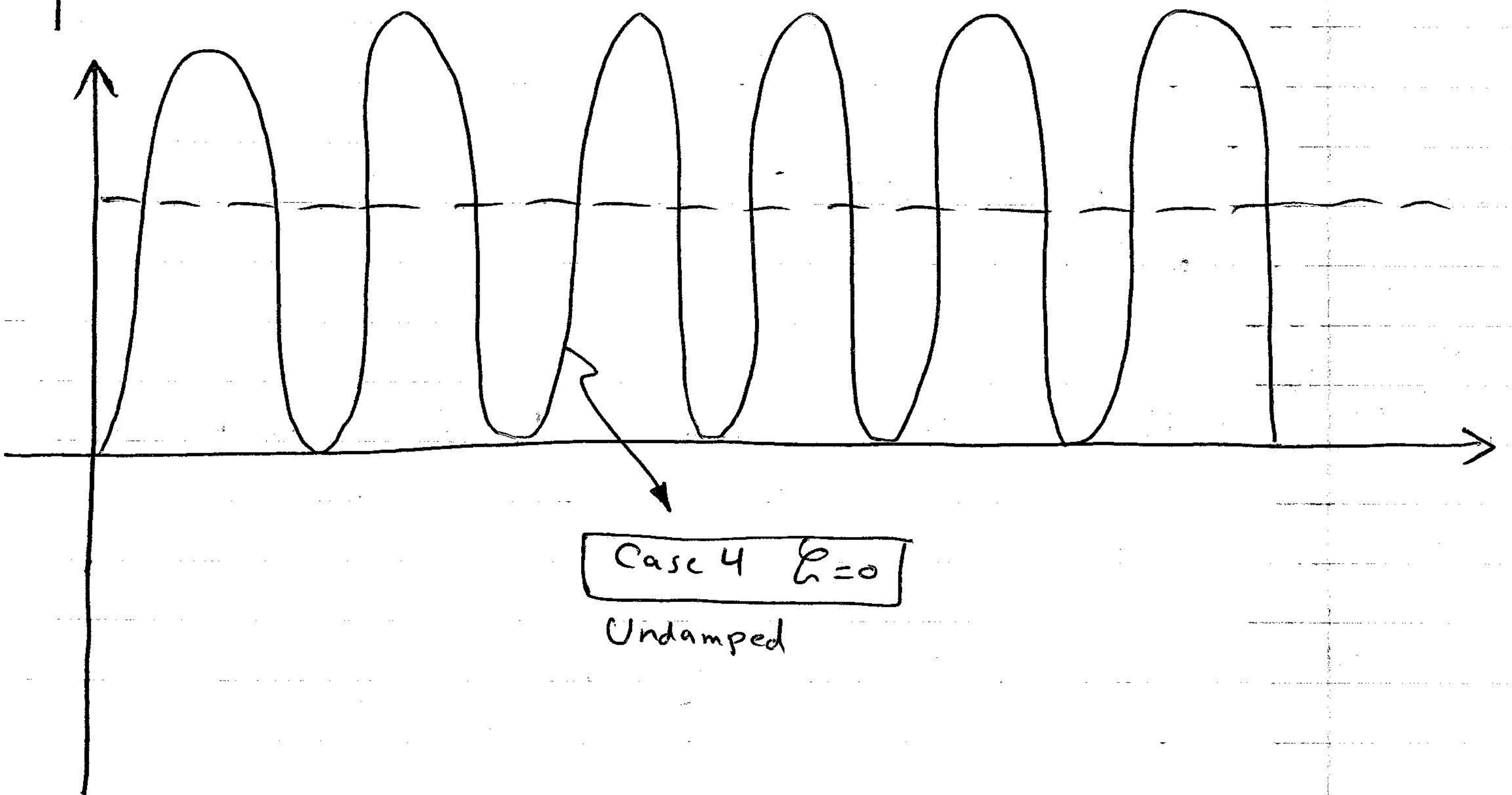
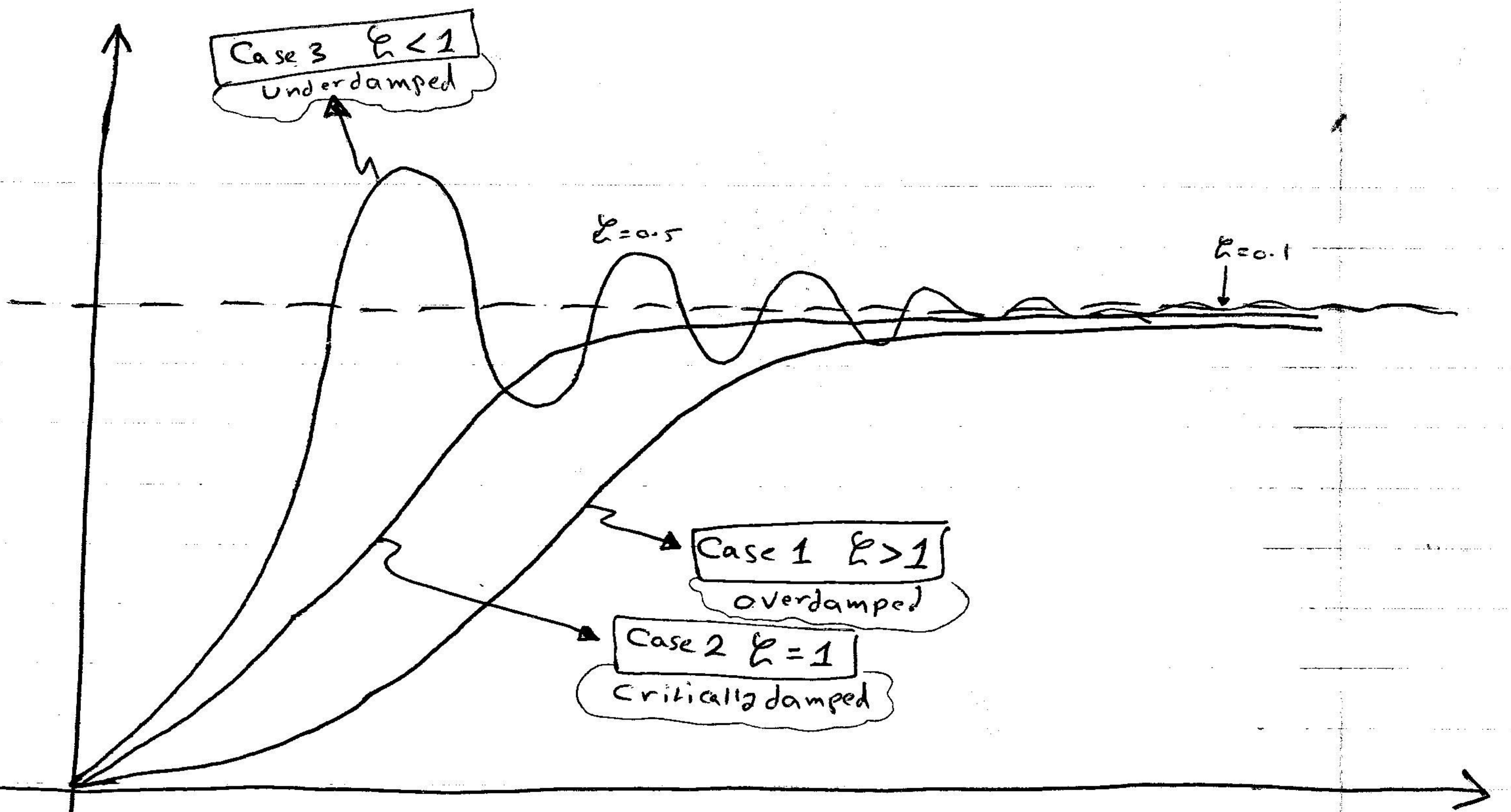
Another form of $C(t)$ [less used] (أقل استخداماً)

$$C(t) = 1 - e^{-\zeta \omega_n t} \left[\cos \omega_d t + \frac{\zeta}{\sqrt{1-\zeta^2}} \sin \omega_d t \right]$$

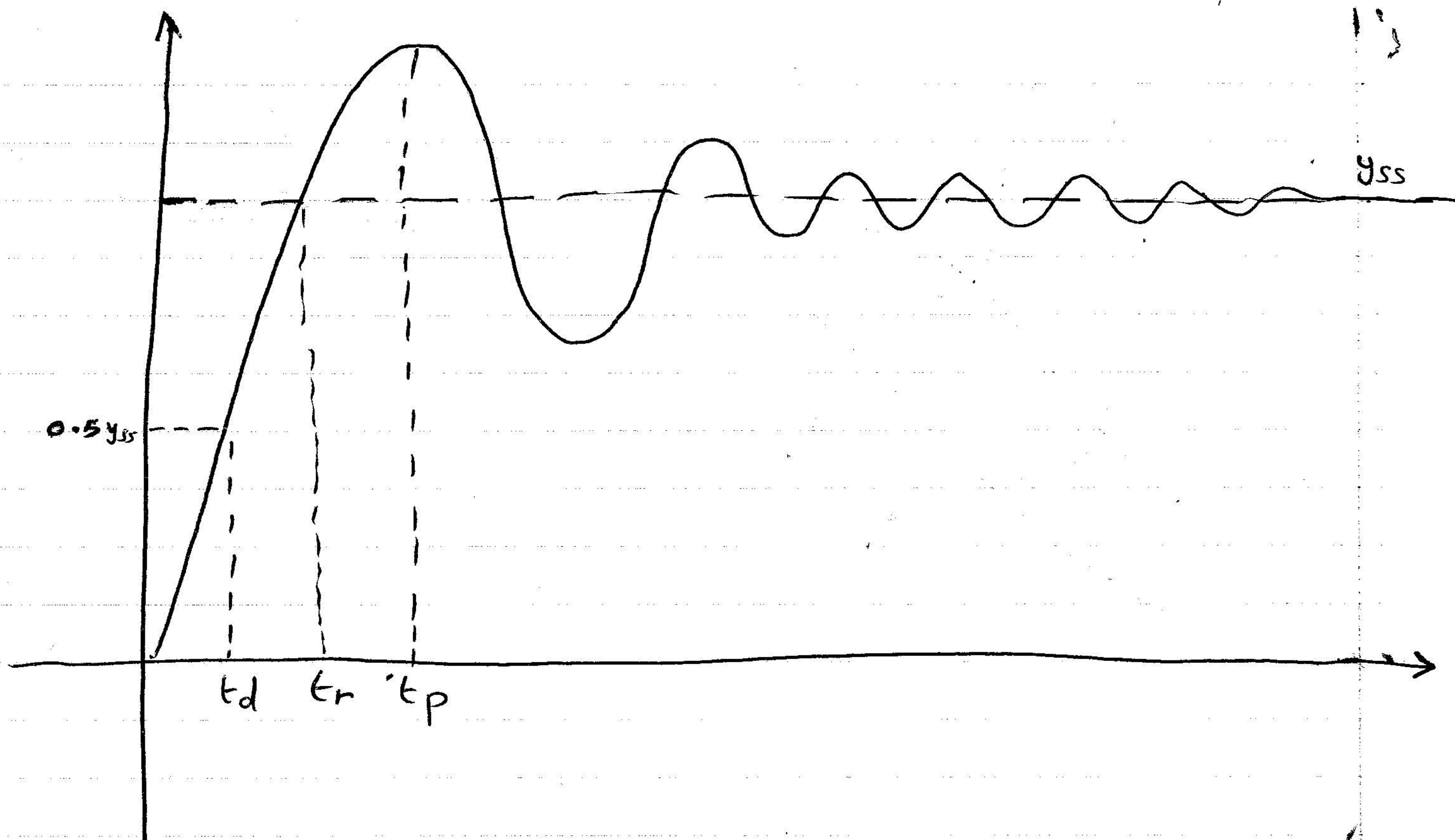
CASE 4 Undamped Oscillation $\zeta = 0$

$$C(t) = 1 - \cos \omega_n t$$

$$\omega_d = \omega_n \sqrt{1-\zeta^2}$$



Tue; 22/10/2013

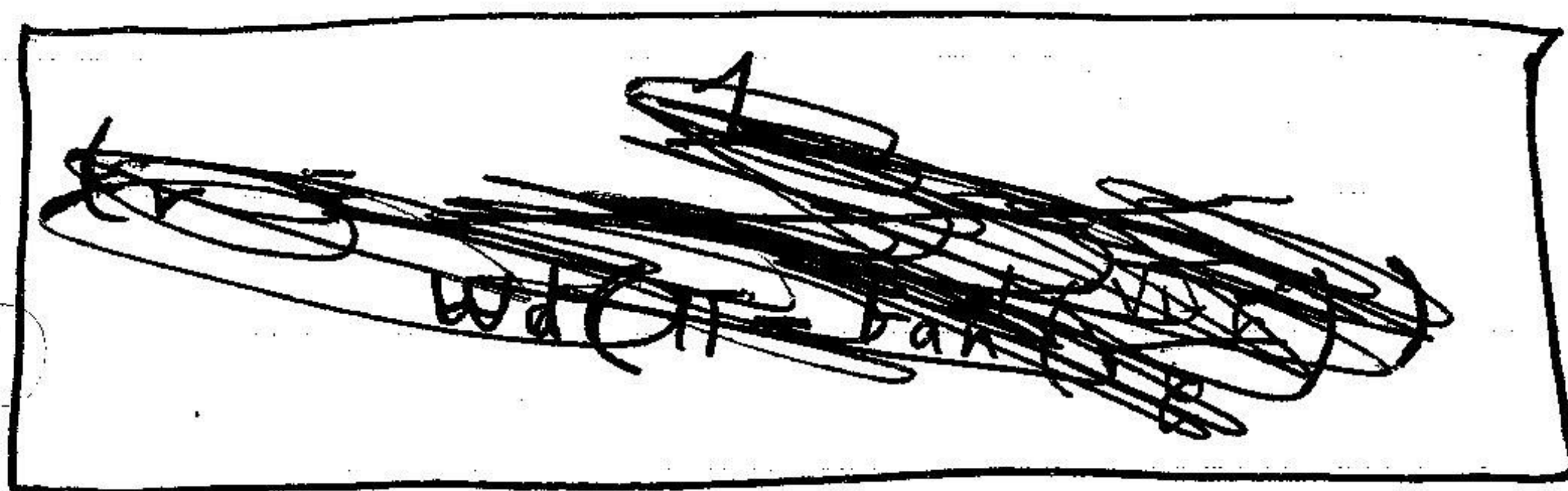


1. y_{ss} : is the final value at (∞) reached by the system
 $= KR$

2. Delay time : is the time required by the system to reach 50% of the final value. (t_d)

3. Rising time (t_r) : is the time taken by the signal to reach
 $[0 \rightarrow 100\%]$ for underdamped Response.
 $[5 \rightarrow 95\%]$ for Critically damped Response.
 $[10 \rightarrow 90\%]$ for overdamped Response.

for underdamped :



$$t_r = \frac{1}{\omega_d} \left(\pi - \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta} \right)$$

40 Peak time (t_p) : is the time required for the response to reach the first peak

by taking the 1st derivative of response equation :

$$\frac{dc(t)}{dt} = 0$$

$$\underbrace{[\sin \omega_d t_p]}_{\substack{\downarrow \\ \text{Can be} \\ \text{Zero!}}} \underbrace{\frac{\omega_n}{\sqrt{1-\zeta^2}}}_{\substack{\downarrow \\ \text{impossible to} \\ \text{be zero}}} \underbrace{e^{-\zeta \omega_n t_p}}_{\substack{\downarrow \\ \text{Zero when} \\ t_p = \infty \\ \times}} = 0$$

$$\therefore \Rightarrow \sin \omega_d t_p = 0$$

$$\therefore \omega_d t_p = 0, \pi, 2\pi, 3\pi, 4\pi, \dots$$

$$\therefore \omega_d t_p = \pi$$

$$\boxed{\therefore t_p = \frac{\pi}{\omega_d}} \quad \text{1st Peak time}$$

$$\boxed{t_{p2} = \frac{3\pi}{\omega_d}} \quad \text{2nd Peak time,}$$

and so on \Rightarrow

$$\text{⊗ Remember : } \omega_d = \omega_n \sqrt{1-\zeta^2}$$

5. y_p = maximum response

$$y(t_p) = 1 + e^{-\zeta\pi/\sqrt{1-\zeta^2}}$$

6. Maximum overshoot M_p

$$M_p = e^{-\zeta\pi/\sqrt{1-\zeta^2}} = y_p - 1$$

7. Overshoot Percentage

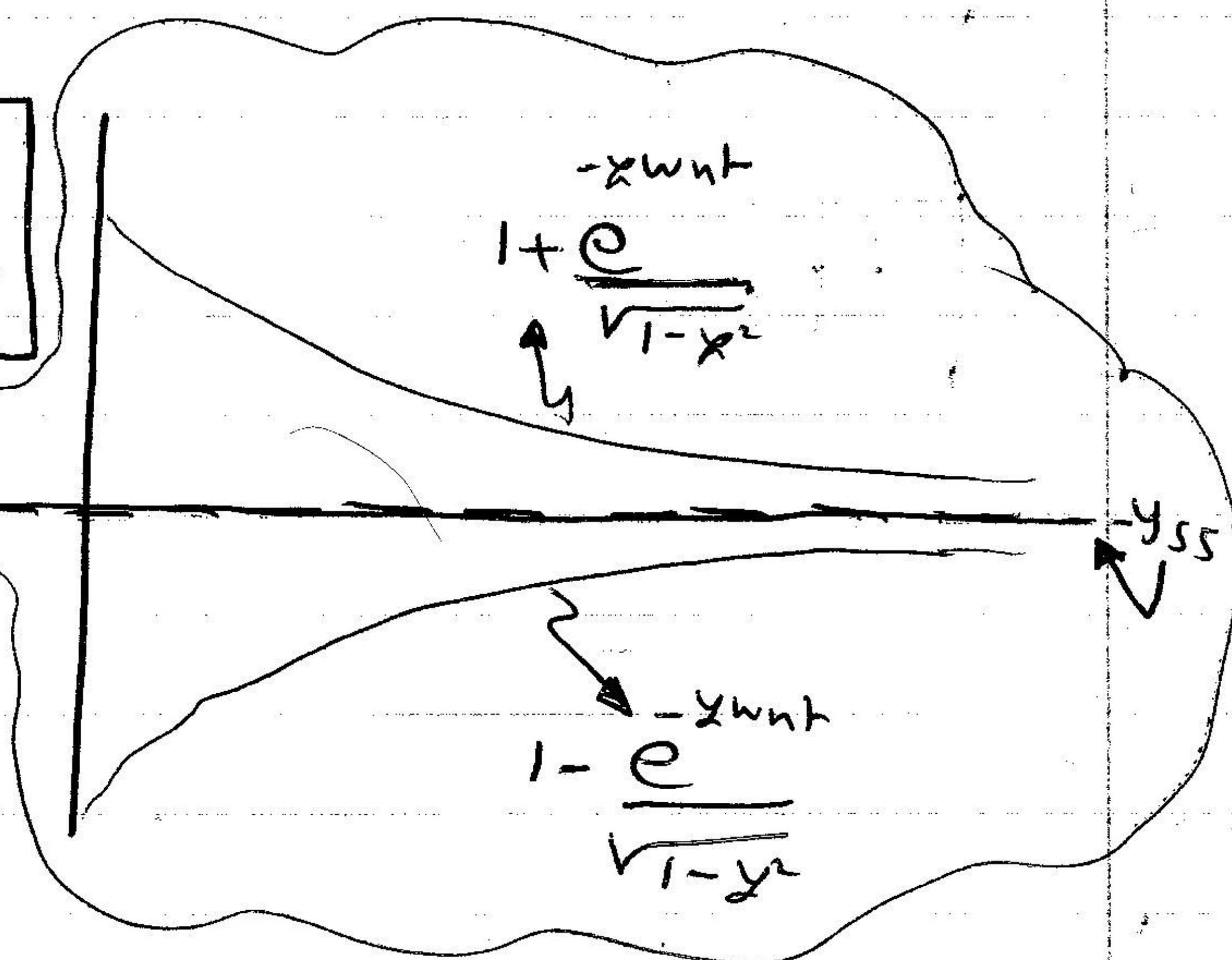
$$M_p\% = \frac{M_p}{y_{ss}} * 100\% = e^{-\zeta\pi/\sqrt{1-\zeta^2}} * 100\%$$

in the case $y_{ss} = 1$

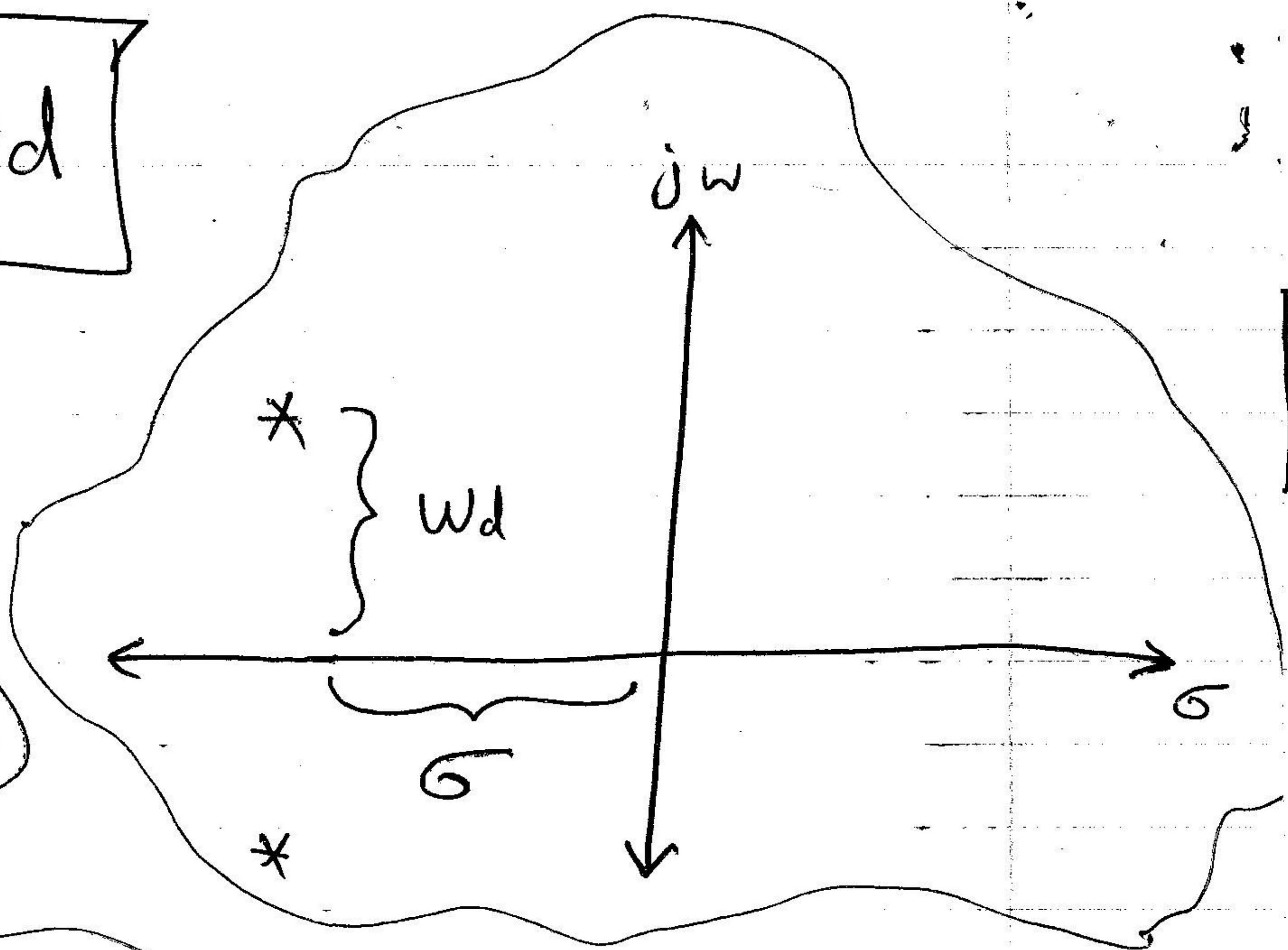
8. Settling time (t_s)

$$t_s \Big|_{5\%} = 3\tau = \frac{3}{\zeta\omega_n}$$

$$t_s \Big|_{2\%} = 4\tau = \frac{4}{\zeta\omega_n}$$



$$s = \sigma + j\omega d$$

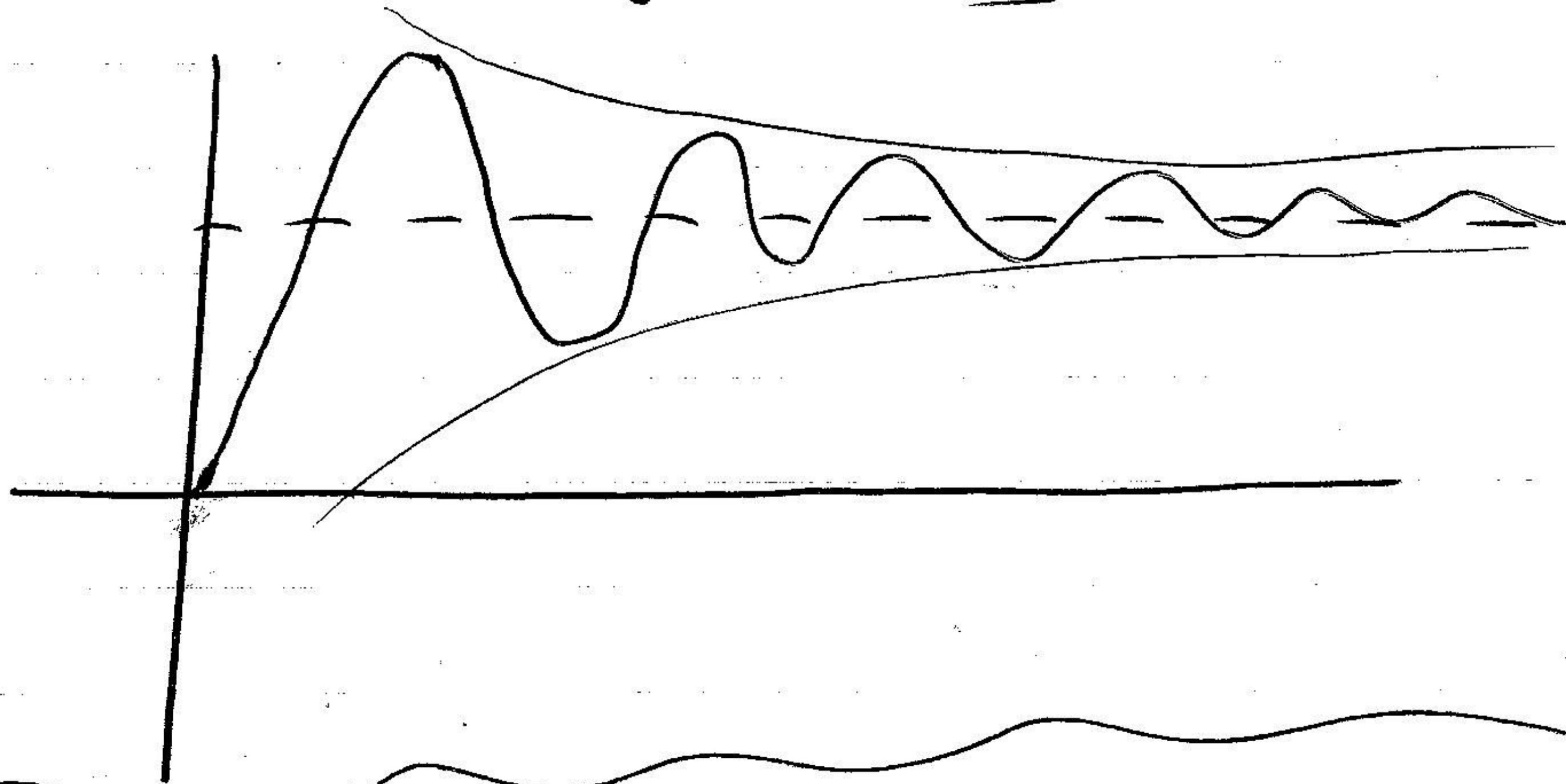


• σ goes bigger
 \Rightarrow the system is faster to be stable

• ωd goes bigger
 \Rightarrow the frequency of the system will be larger

9. Exponential Decay ratio

is measured by dividing the 2nd peak by the 1st peak

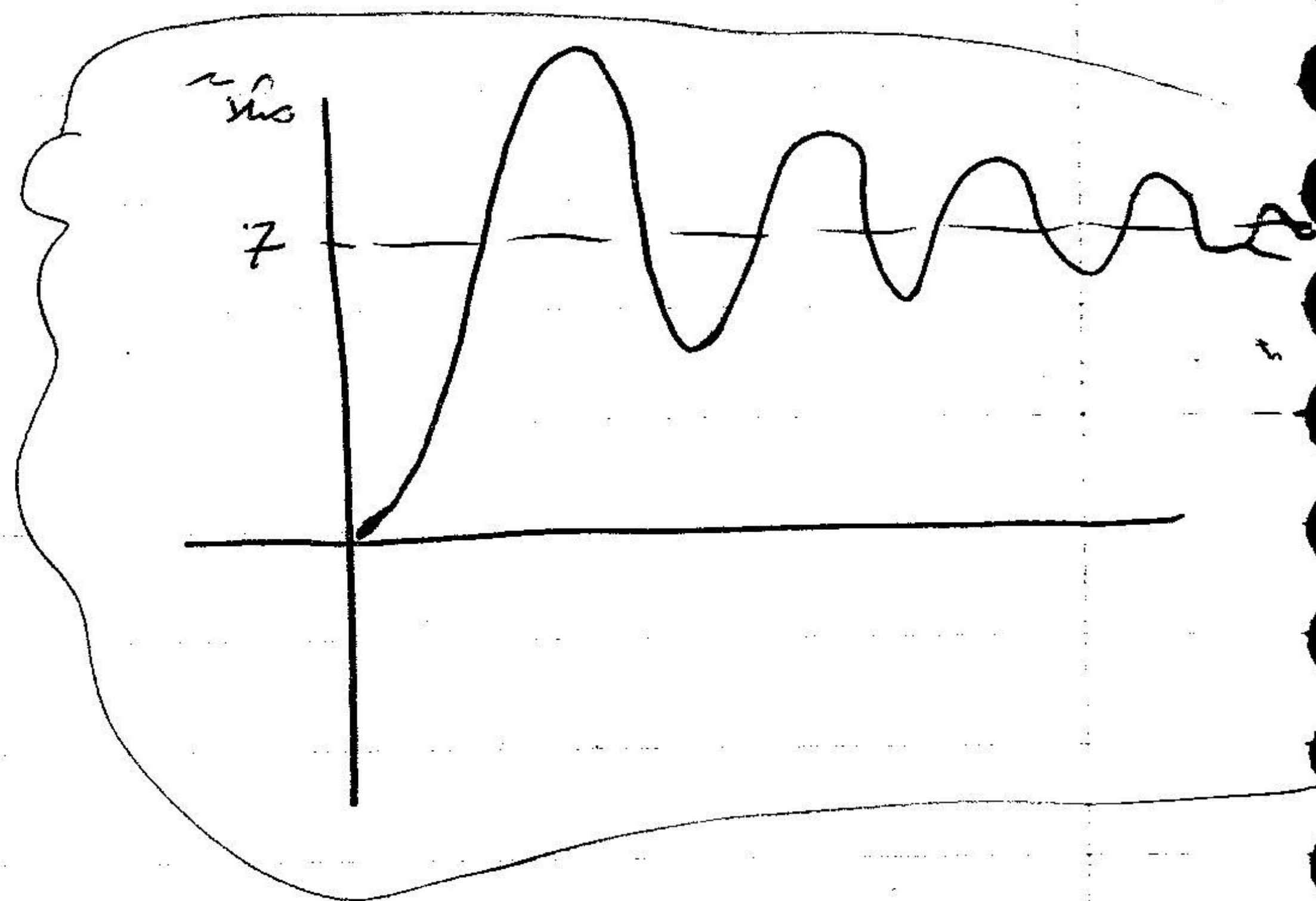


* exponential decay ratio = $\frac{MP_2}{MP_1} = \frac{MP_3}{MP_2}$

• exponential decay ratio = $\frac{MP_2}{MP_1} = \frac{e^{-3\zeta\pi/\sqrt{1-\zeta^2}}}{e^{-\pi/\sqrt{1-\zeta^2}}}$

$$= e^{-2\zeta\pi/\sqrt{1-\zeta^2}}$$

Ex Find ζ if the $E.D.R = 50\%$



~~Ex~~

$$E.D.R = 0.5 = e^{-2\zeta\pi/\sqrt{1-\zeta^2}}$$

$$\therefore \ln 0.5 = \frac{-2\zeta\pi}{\sqrt{1-\zeta^2}}$$

$$\therefore -0.7 = \frac{-2\zeta\pi}{\sqrt{1-\zeta^2}} \Rightarrow 0.7 = \frac{2\zeta\pi}{\sqrt{1-\zeta^2}}$$

$$\Rightarrow 0.7\sqrt{1-\zeta^2} = 2\pi\zeta$$

$$\Rightarrow 0.7\sqrt{1-\zeta^2} = 6.28\zeta$$

$$\Rightarrow \sqrt{1-\zeta^2} = 9\zeta \Rightarrow 1-\zeta^2 = 81\zeta^2$$

$$\Rightarrow 82\zeta^2 = 1 \Rightarrow \zeta = \sqrt{\frac{1}{82}} = \boxed{0.11}$$

Example:- For a 2nd order system, $\zeta = 0.6$

$\omega_n = 5$ rad/sec, Subjected to a unit step input.

Find :-

- ① ω_d ② ζ ③ t_r ④ t_p ⑤ M_p ⑥ t_s
⑦ E.D.R. • then draw the response.

Solution :-

$$① \omega_d = \omega_n \sqrt{1 - \zeta^2} = 5 \sqrt{1 - 0.36} = 5 \sqrt{0.64} = 5(0.8) = \boxed{4}$$

$$② \zeta = \zeta \omega_n = 0.6(5) = \boxed{3}$$

$$③ t_r = \frac{\pi - \beta}{\omega_d} \quad \left[\beta = \tan^{-1} \frac{\sqrt{1 - \zeta^2}}{\zeta} = \tan^{-1} \frac{\omega_d}{\zeta} \right]$$

$$\beta = \tan^{-1} \frac{\omega_d}{\zeta} = \tan^{-1} \frac{4}{3} = 53^\circ = \boxed{0.92 \text{ radian}}$$

$$\therefore t_r = \frac{\pi - 0.92}{4} = \boxed{0.55 \text{ s}}$$

$$④ t_p = \frac{\pi}{\omega_d} = \frac{\pi}{4} = \boxed{0.78 \text{ s}}$$

$$⑤ M_p = e^{-\frac{\zeta \pi}{\sqrt{1 - \zeta^2}}} = e^{-\frac{\pi \zeta}{\omega_d}} = e^{-\frac{3}{4} \pi} = \boxed{0.1}$$

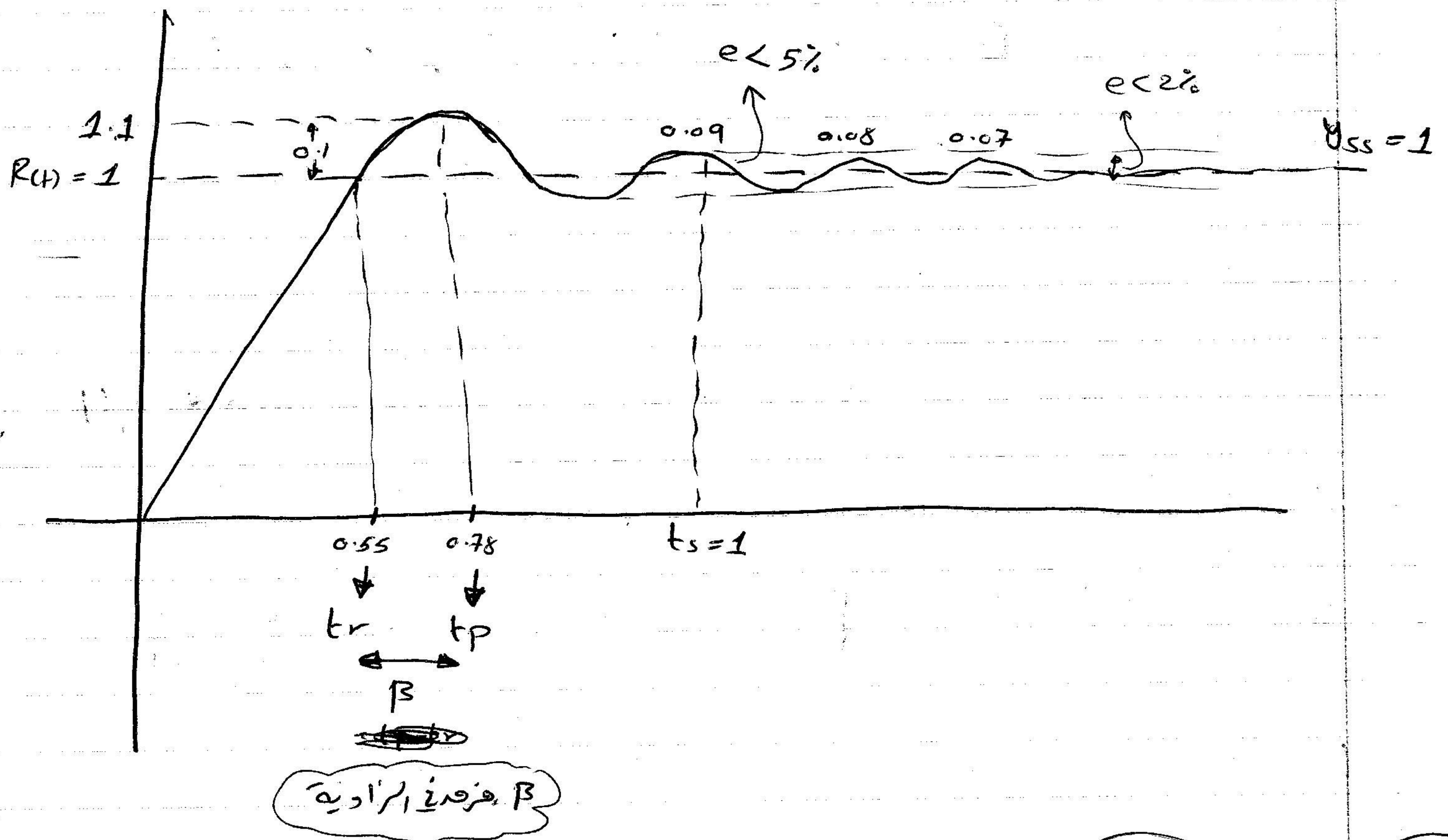
$$M_p\% = 0.1 \times 100\% = \boxed{10\%}$$

$$⑥ t_{s \atop 5\%} = \frac{3}{\zeta} = \frac{3}{3} = \boxed{1 \text{ s}}$$

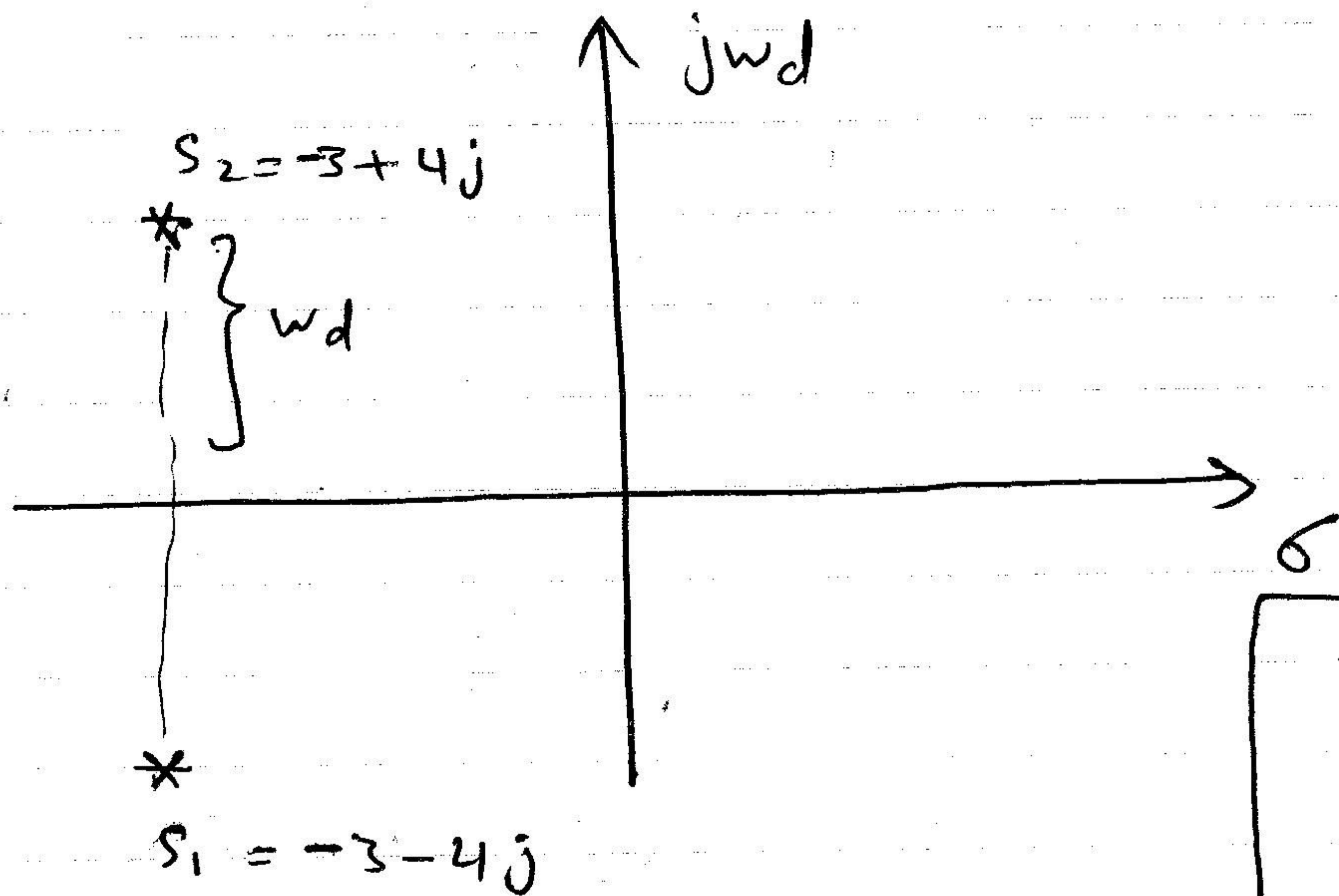
$$t_{s \atop 2\%} = \frac{4}{\zeta} = \frac{4}{3} = \boxed{1.3 \text{ s}}$$

$$⑦ \text{E.D.R} = e^{-\frac{2\zeta \pi}{\sqrt{1 - \zeta^2}}} = [M_p]^2 = \boxed{0.01}$$
$$= [0.1]^2 = \boxed{0.01}$$

⊛ Drawing the response & -



⊛ Roots of the system $s = \sigma + j\omega_d$



This system is stable
Roots are -ve

Roots location (2 roots)

⊛ $G(s)$ for the system $= \frac{k\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} =$

$\frac{25}{s^2 + 6s + 25}$

T.F.

(26)

Ex (H.W #1)

Find ω_n & ζ

So that the PO% = 25%

and Settling time at 5% Criteria = 3 s

Find the other Specs.

$$\textcircled{*} e^{-\zeta\pi/\sqrt{1-\zeta^2}} = 0.25 \Rightarrow \ln e^{-\zeta\pi/\sqrt{1-\zeta^2}} = \ln 0.25$$

$$\Rightarrow \frac{-\zeta\pi}{\sqrt{1-\zeta^2}} = -1.386 \Rightarrow \frac{+3.14\zeta}{\sqrt{1-\zeta^2}} = +1.386$$

$$\Rightarrow \frac{9.869\zeta^2}{1-\zeta^2} = 1.921 \Rightarrow 9.869\zeta^2 = 1.921 - 1.921\zeta^2$$

$$11.79\zeta^2 = 1.921 \Rightarrow \zeta^2 = \frac{1.921}{11.79} \approx 0.16 \Rightarrow \boxed{\zeta = 0.4}$$

$$t_{s|5\%} = \frac{3}{\zeta\omega_n} = 3 \Rightarrow \zeta\omega_n = 1$$

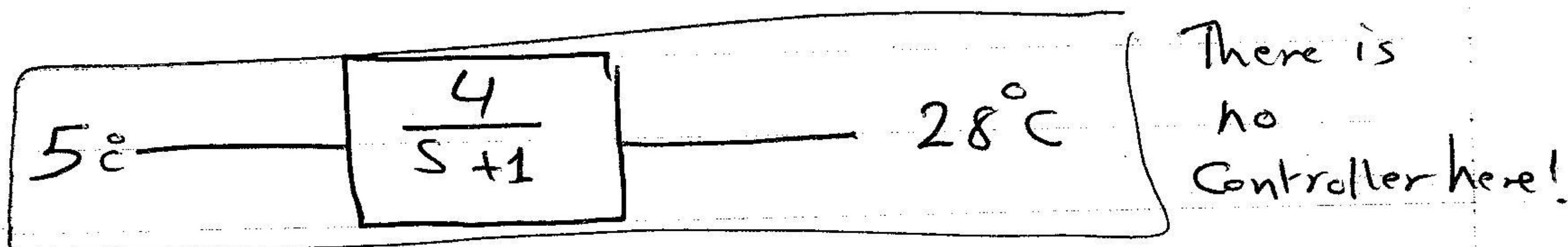
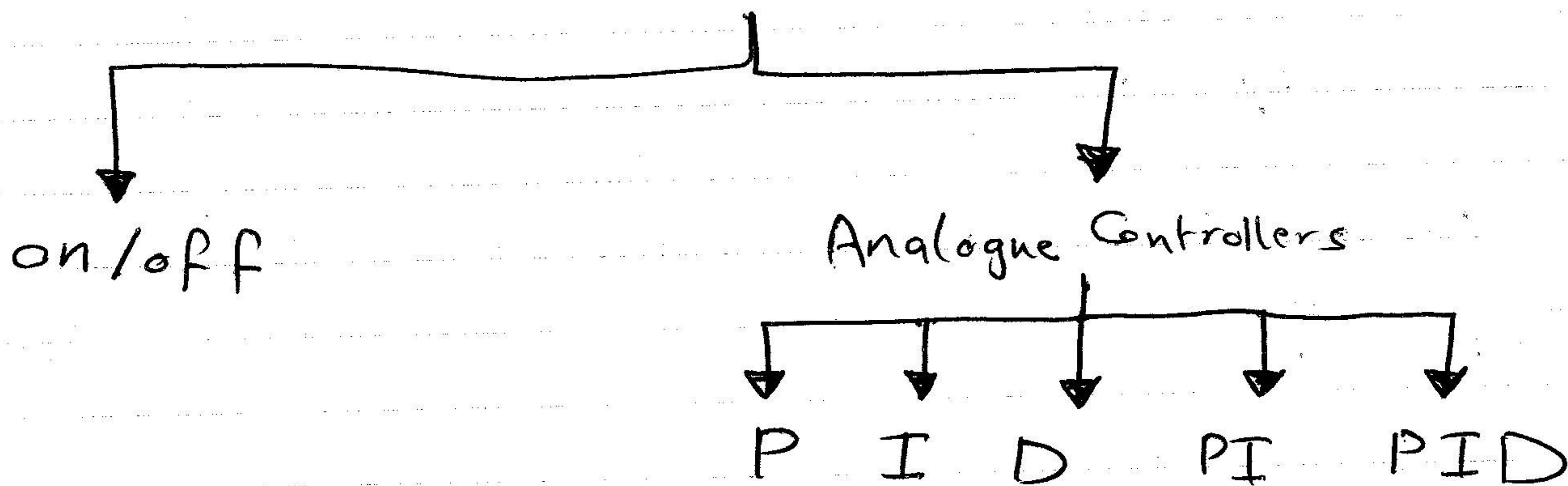
$$\Rightarrow 0.4\omega_n = 1 \Rightarrow \boxed{\omega_n = \frac{1}{0.4} = 2.5}$$

$$\zeta = 0.4$$
$$\omega_n = 2.5$$

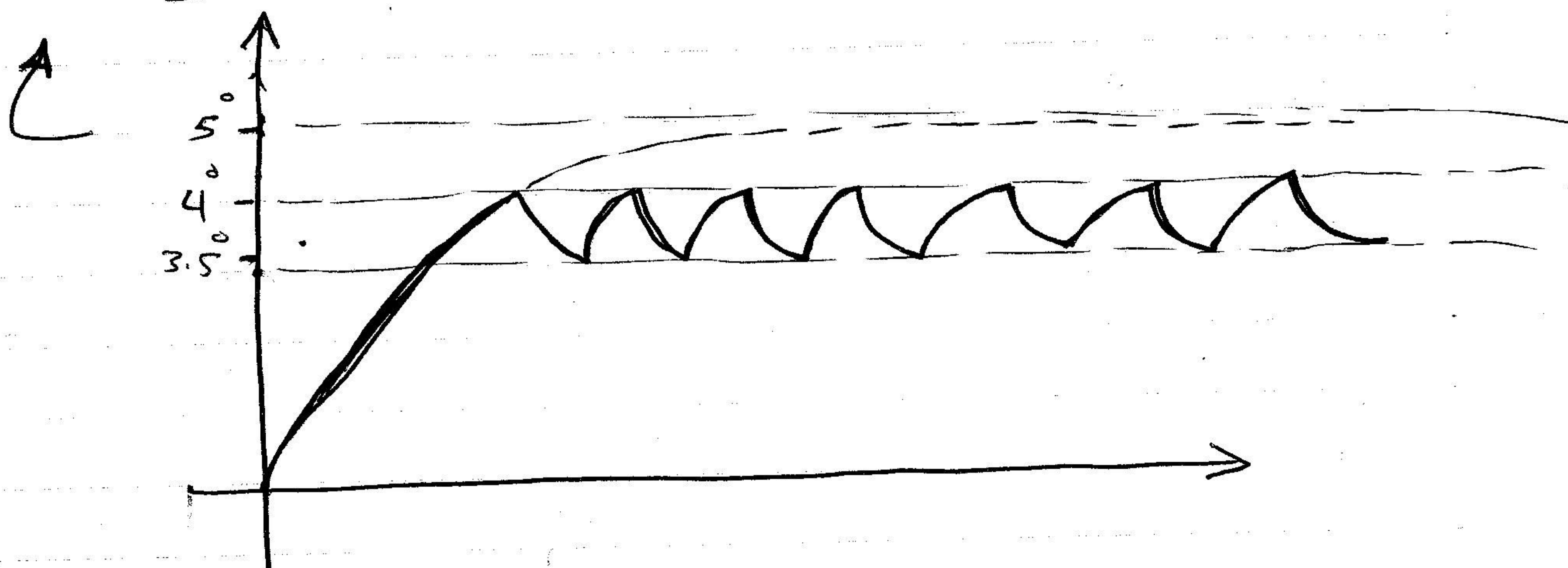
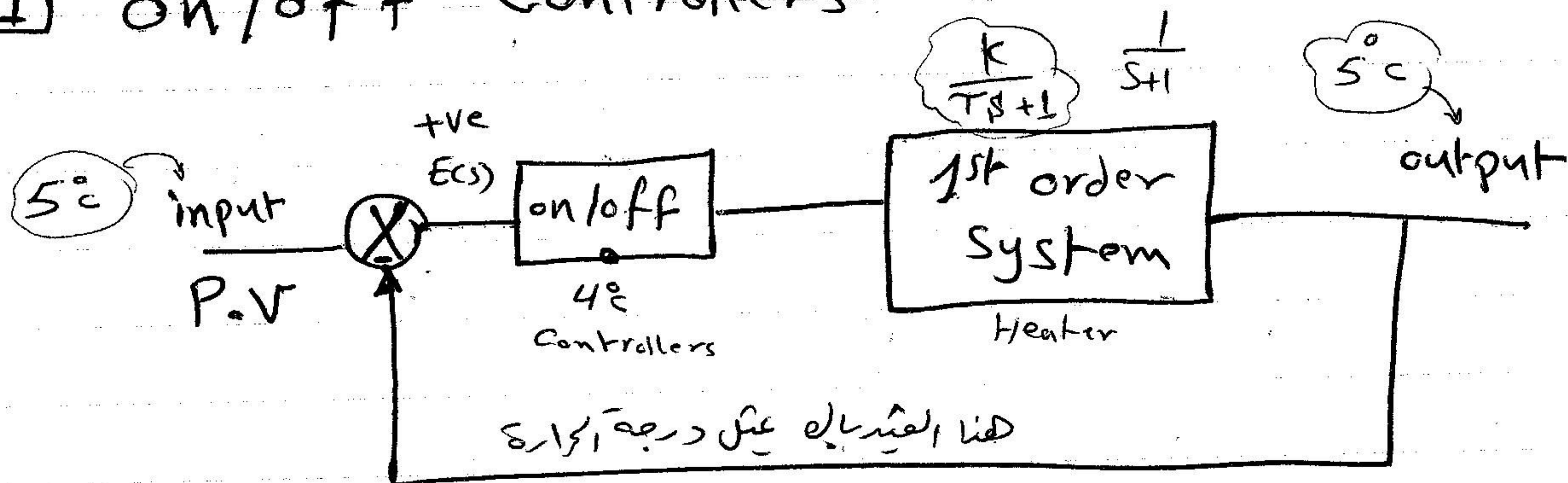
We find ζ and $\omega_n \Rightarrow \therefore$ Now we can find
all the other Specs :-)

CONTROLLERS

Conventional Controllers



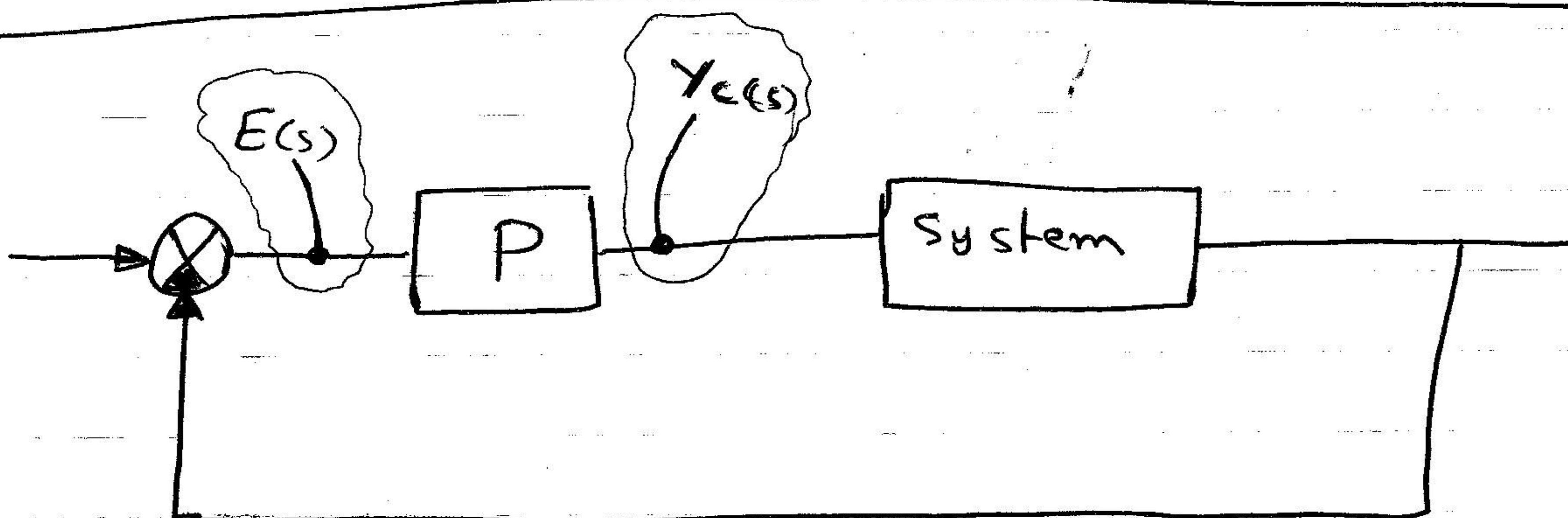
① on/off Controllers



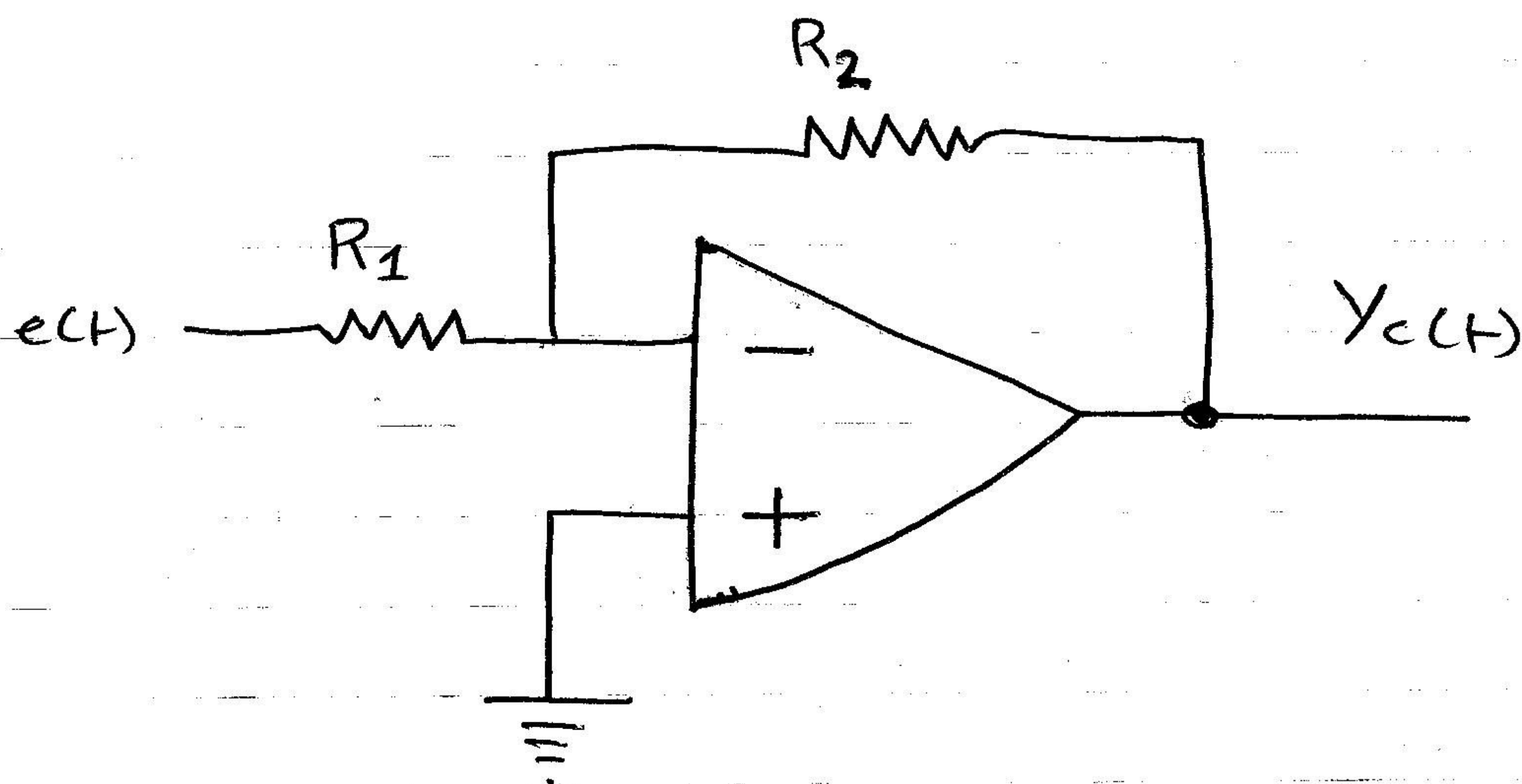
An Example on (on/off) Controllers : Thermo Switch

2 Analogue Controllers

a) Proportional Controller [P] : "الحكم التناسبي"

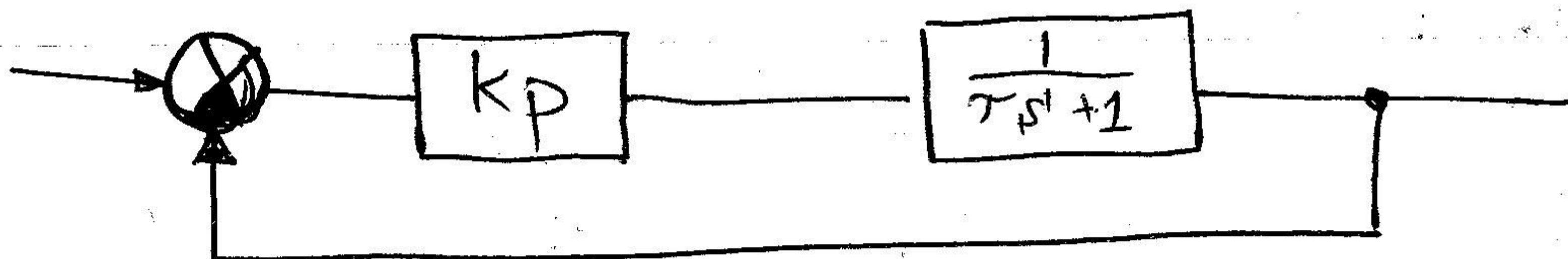


$$Y_c(s) = K_p * E(s)$$



$$Y_c(t) = -\frac{R_2}{R_1} * e(t)$$

$$\Rightarrow K_p = \frac{R_2}{R_1}$$



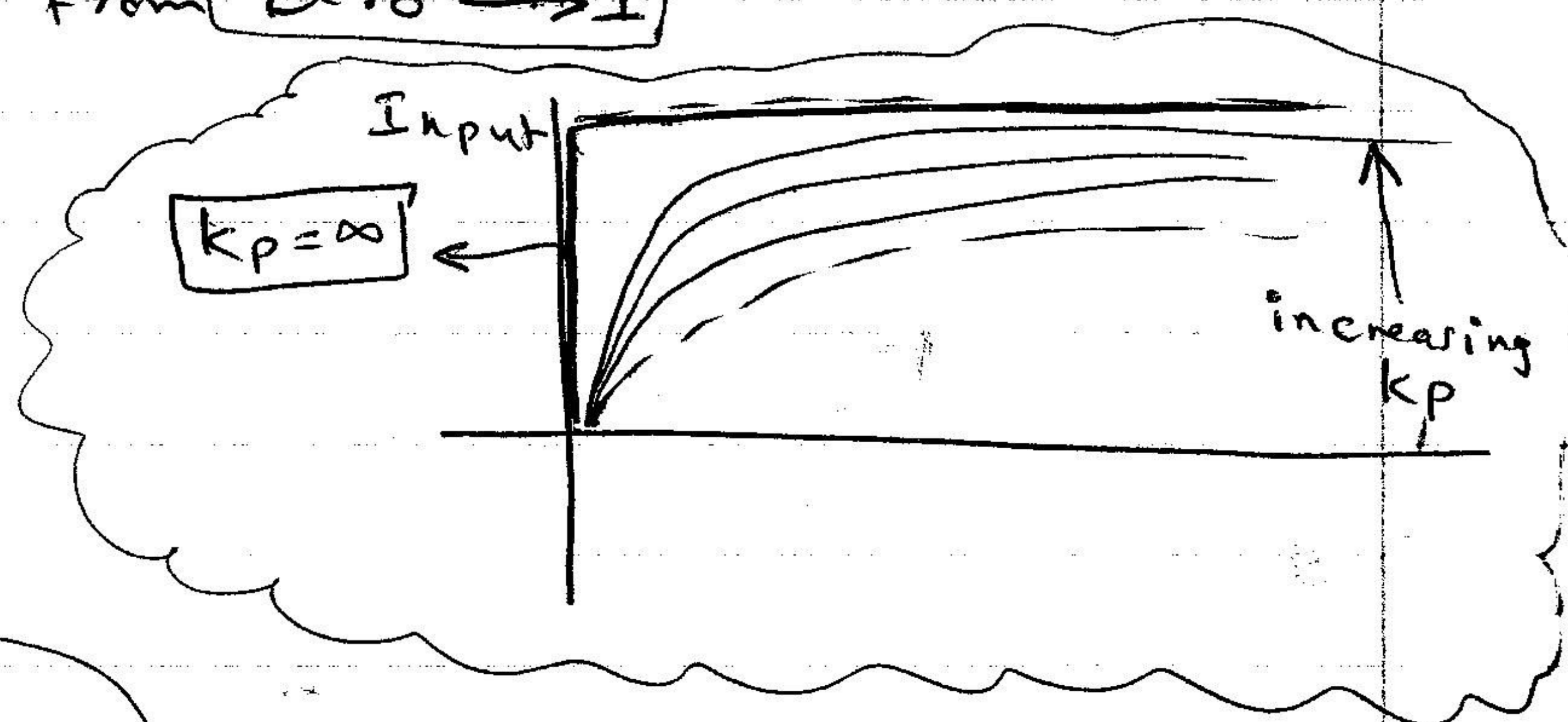
$$= \frac{\frac{K_P}{\gamma s + 1}}{1 + \frac{K_P}{\gamma s + 1}} = \boxed{\frac{K_P}{\gamma s + 1 + K_P}} = \boxed{\frac{\frac{K_P}{1 + K_P}}{\frac{\gamma}{1 + K_P} s + 1}}$$

$$\textcircled{*} \text{ Total gain} = \frac{K_P}{1 + K_P}$$

⊛ if K_P increased from $\text{Zero} \rightarrow \infty$

⊛ The total gain increases from $\text{Zero} \rightarrow 1$

$$\text{Error} = \frac{1}{1 + K_P} R$$



$$* K_P = 10$$

$$\Rightarrow \text{gain} = \frac{10}{1 + 10} = \frac{10}{11} = 0.909$$

$$* K_P = 20$$

$$\Rightarrow \text{gain} = \frac{20}{1 + 20} = \frac{20}{21} = 0.95$$

مميزات التحكم التناسبي

- 1] خفيف الثمن
- 2] سريع الاستجابة
- 3] بسيط وسهل التركيب

مزايا

- 1] لا يغير إشارة الكفاءة

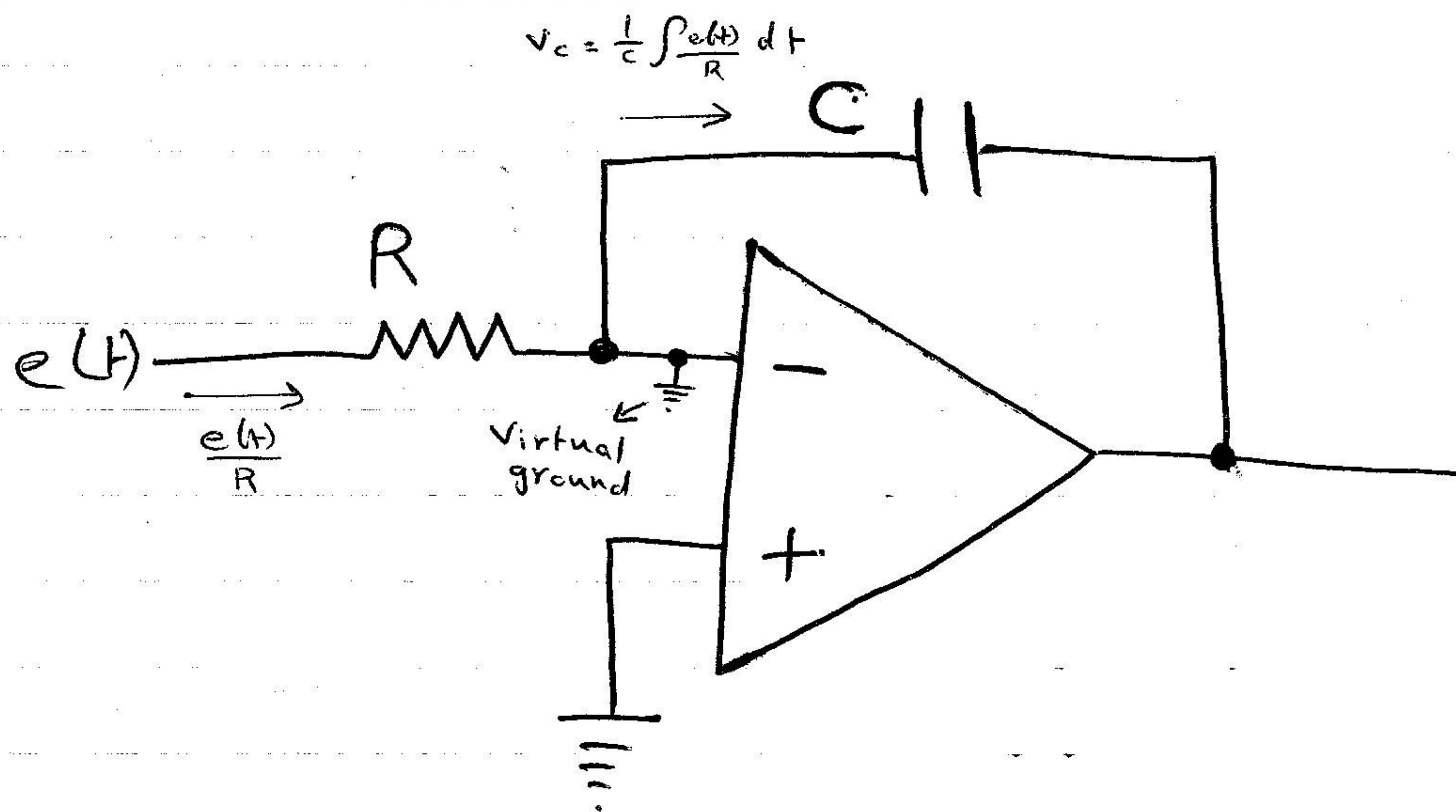
سلبية

ⓑ Integral Controller [I] "التحكم التكامل"

$$y_c(t) = \cancel{k_i} k_i \int e(t) dt$$

$$Y_c(s) = k_i * \frac{1}{s} E(s) \Rightarrow Y_c(s) = \frac{k_i}{s} * E(s)$$

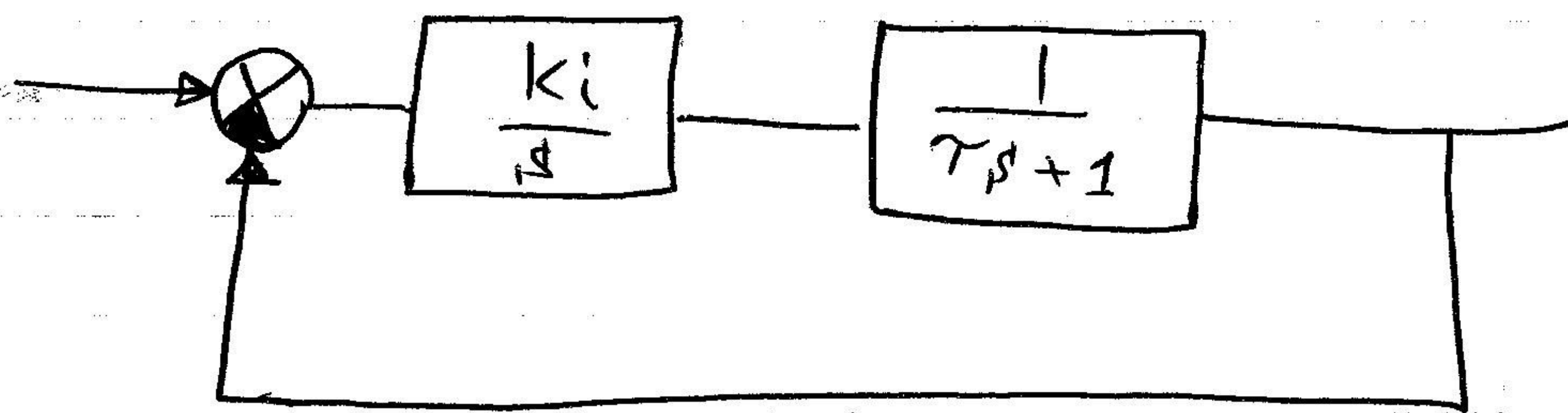
$$T.F. = \frac{Y(s)}{E(s)} = \frac{ki}{s}$$



$$v_c = \frac{1}{C} \int \frac{e(t)}{R} dt = \frac{1}{RC} \int e(t) dt$$

$$\Rightarrow v_o = -\frac{1}{RC} \int e(t) dt$$

$$\therefore ki = \frac{1}{RC}$$



$$\frac{\frac{ki}{s(\tau s + 1)}}{\frac{ki}{s(\tau s + 1)} + 1} = \frac{ki}{ki + s(\tau s + 1)}$$

$$= \frac{ki}{\tau s^2 + s + ki}$$

$$= \frac{\frac{ki}{\tau}}{s^2 + \frac{s}{\tau} + \frac{ki}{\tau}}$$

$$\frac{k\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

→ 2nd order Equation

⇒ total gain = 1

$$\text{Error} = 0$$

مميزات الكاظم المتكامل

- ① يُغيي إشارة الخطأ نهائياً
- ② نسبة الترتيب
- ③ - حيف التشن

- سيئاته
- ① يمكن أن يحدث استعارة في النظام
 - أ. يزيد من قيمة الاستعارة
 - ② يمكن أن يؤثر على استقرار النظام

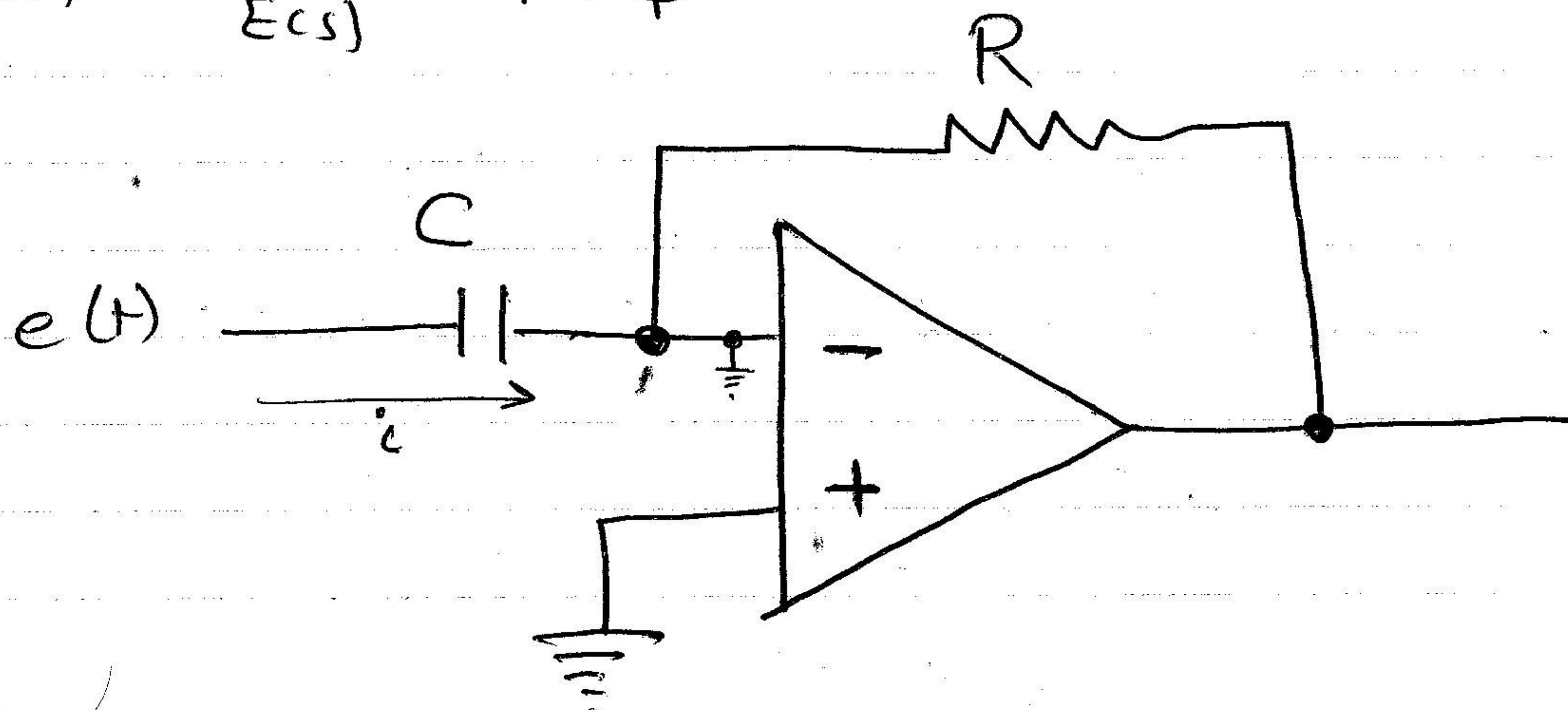
المحاضرة ١٥/٣

☐ Derivative Controller [D] "الكاظم المتفاضلي"

$$y_c = k_d * \frac{de(t)}{dt}$$

$$Y_c(s) = k_d s E(s)$$

$$G_c(s) = \frac{Y_c(s)}{E(s)} = k_d s$$



$$\textcircled{*} V_c = \frac{1}{C} \int i dt$$

$$e(t) = \frac{1}{C} \int i dt$$

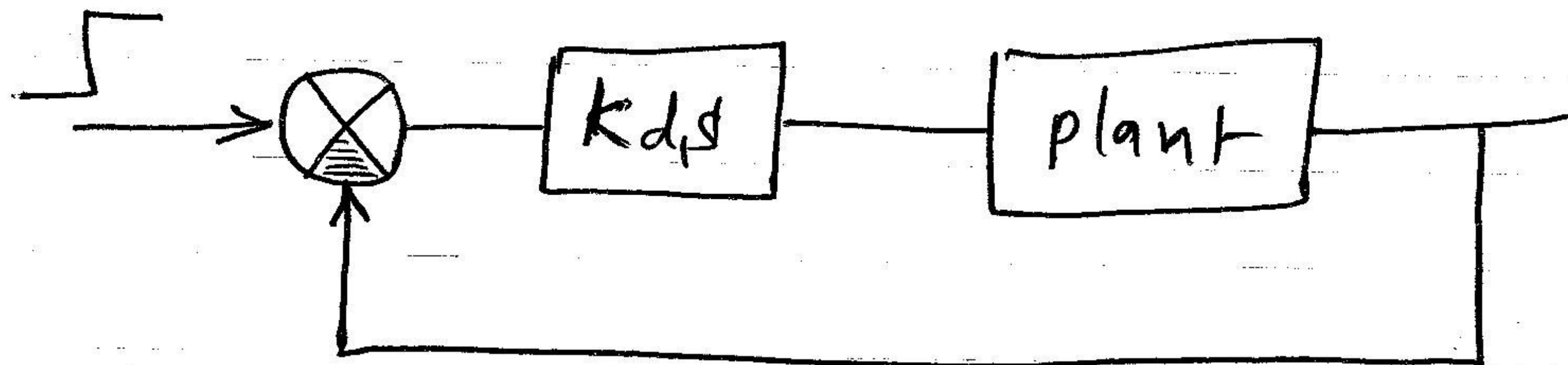
$$C e(t) = \int i dt$$

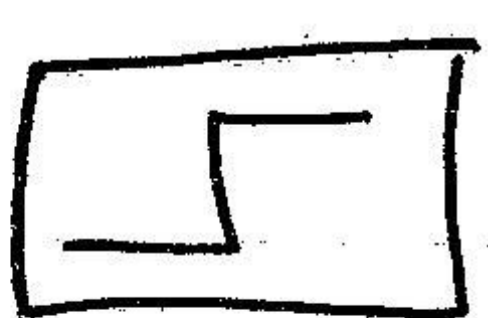
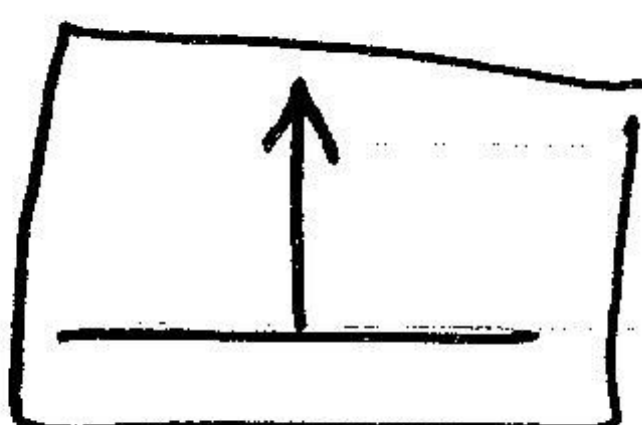
$$C \frac{de(t)}{dt} = i \quad \textcircled{*}$$

$$\textcircled{*} V_R = Ri = RC \frac{de(t)}{dt}$$

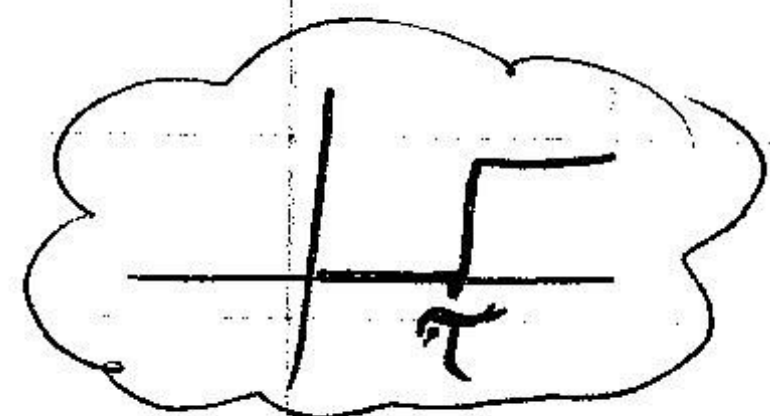
$$\textcircled{*} V_o = -RC \frac{de(t)}{dt}$$

$$\Rightarrow K_d = RC$$



derivative of this signal  is  pulse

take care from this Controller!! :-)



$\textcircled{*}$ D-Controller when added to P-Controller, provides a means of high sensitivity.

“میزان حساسیت بالا”

$\textcircled{*}$ an advantage of using D-Controller is that it responds to the change of actuating error & can provide a significant correction before the magnitude of the error goes large.

and thus, it tends to increase the stability of the system.

Why we do not use **D**-Controller alone?

① it can't eliminate error.

② in the case of ^{unit} step input, this controller give an impulse function with amplitude goes to (∞) !

d PID - Controller

Combines the advantages of P + I + D controllers.

$$y_c(t) = k_p * e(t) + k_i \int e(t) dt + k_d \frac{de(t)}{dt}$$

$$\begin{aligned} Y_c(s) &= k_p E(s) + \frac{k_i}{s} E(s) + k_d s' E(s) \\ &= E(s) \left[k_p + \frac{k_i}{s} + k_d s' \right] \end{aligned}$$

$$\therefore G(s) = \frac{Y_c(s)}{E(s)} = k_p + \frac{k_i}{s} + k_d s'$$

$$\therefore G(s) \text{ (as T.F)} = \frac{k_d s'^2 + k_p s' + k_i}{s'}$$

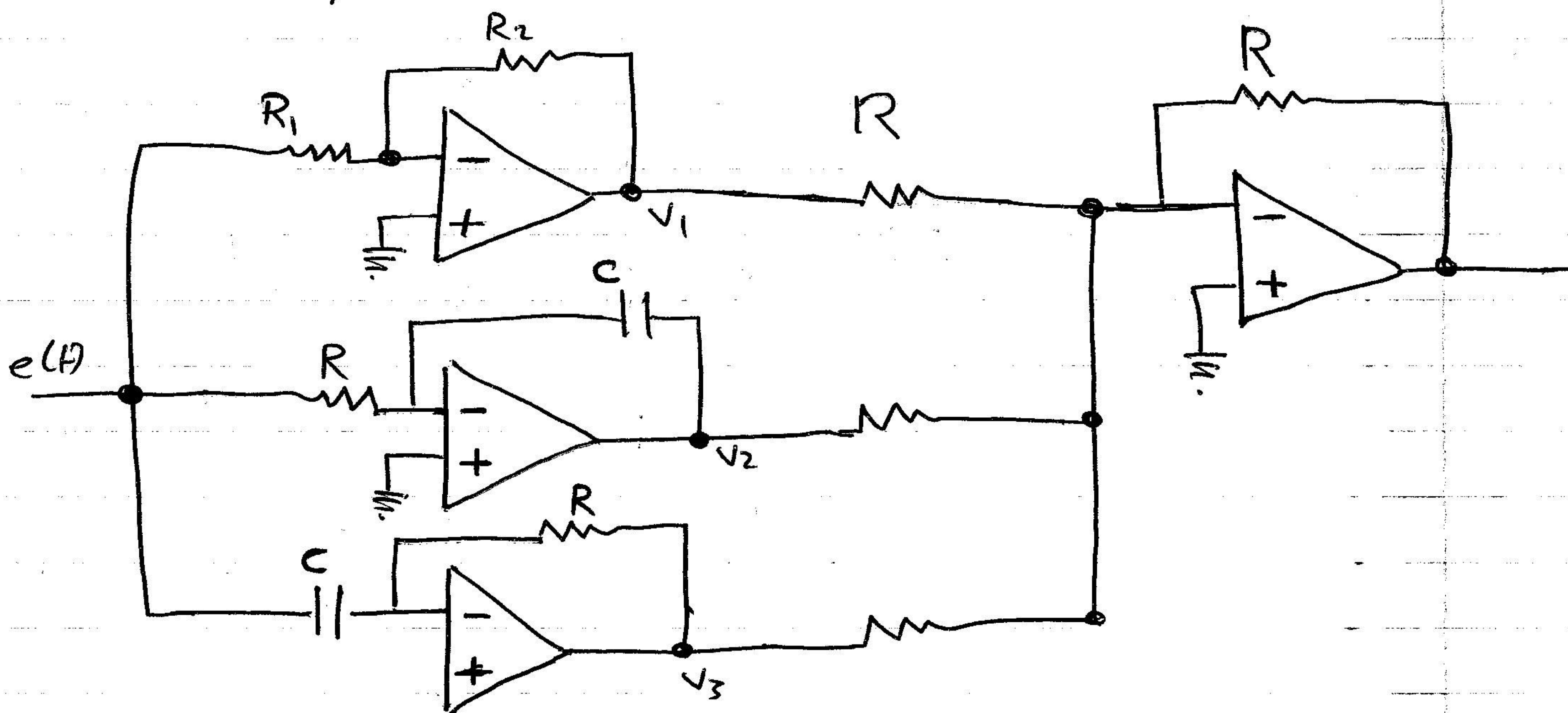
$$G(s) = k_p \left[1 + \frac{1}{T_i s'} + T_d s' \right]$$

Some engineers use this Equation.

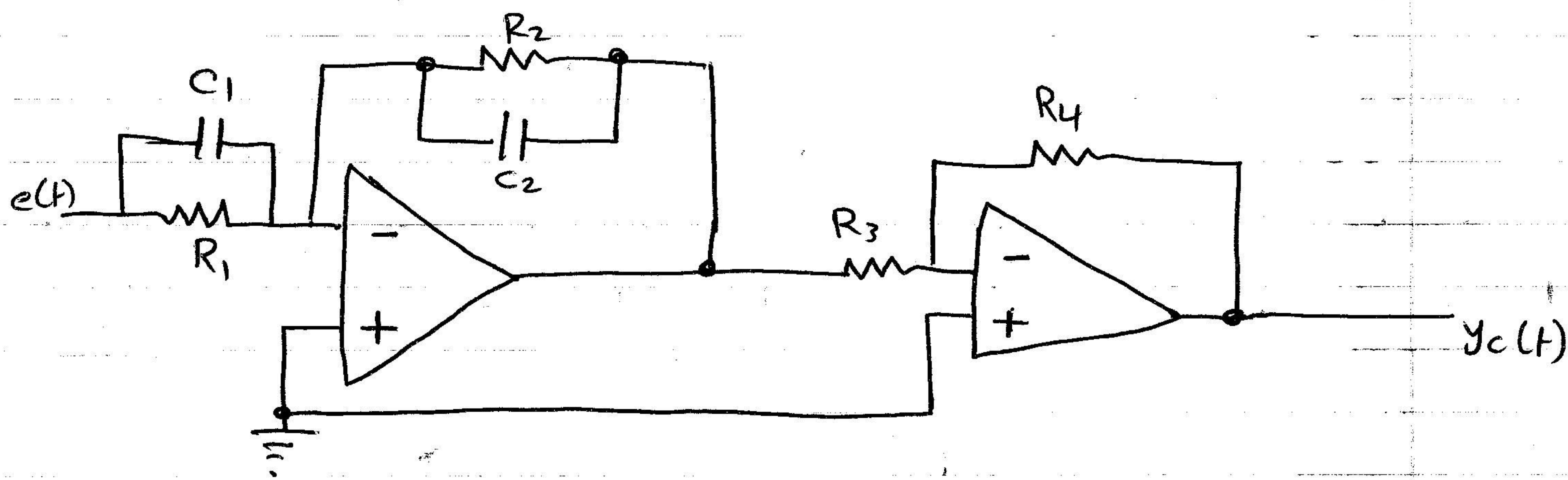
من مساوی های بالا می بینیم: - هر چه ضرایب k_p ، k_i و k_d بزرگتر باشد،

سیستم پایداری بیشتری خواهد داشت.

⊗ PID Equivalent electronic circuit :-



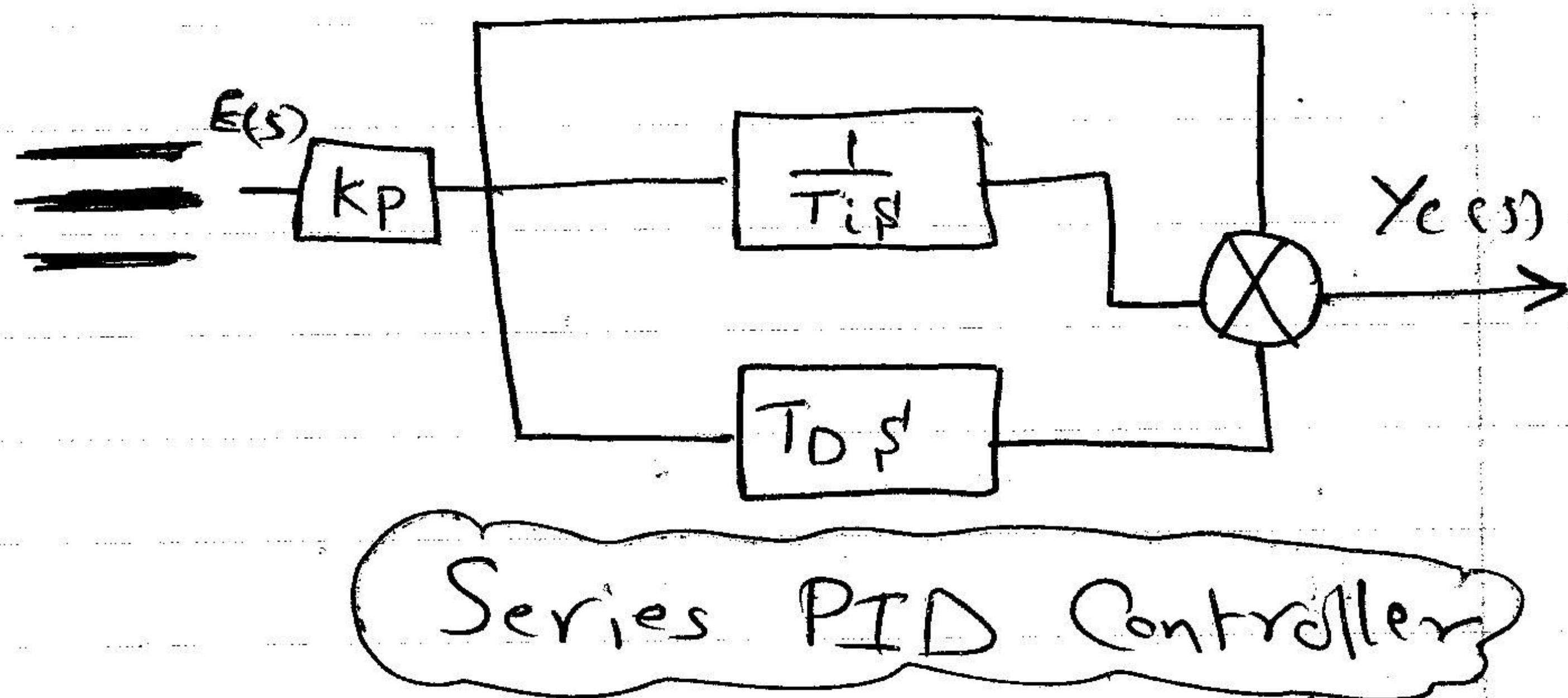
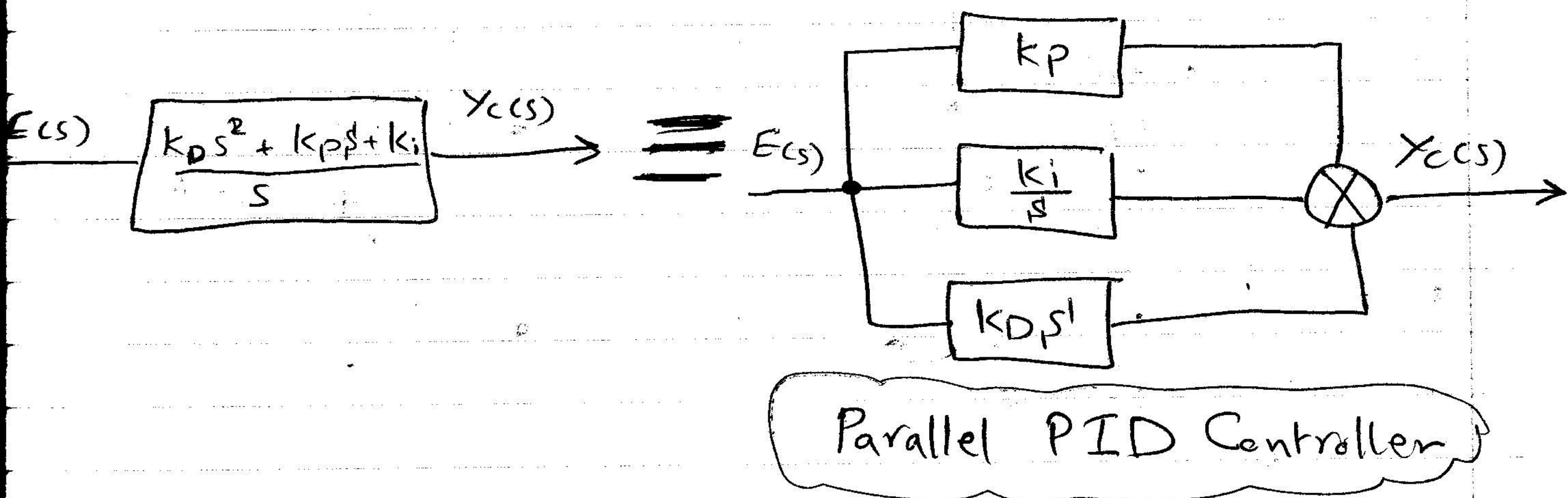
⊗ Another Equivalent CKT for PID-Controller (Ogata) :-



$$\frac{Y_c(s)}{E(s)} = G(s) = \frac{R_4}{R_3} * \frac{R_2}{R_1} \frac{(R_1 C_1 s + 1)(R_2 C_2 s + 1)}{R_2 C_2 s}$$

هنا التكميم غير مستقل، التكميم بأي نوع - يؤثر على الأنواع الأخرى

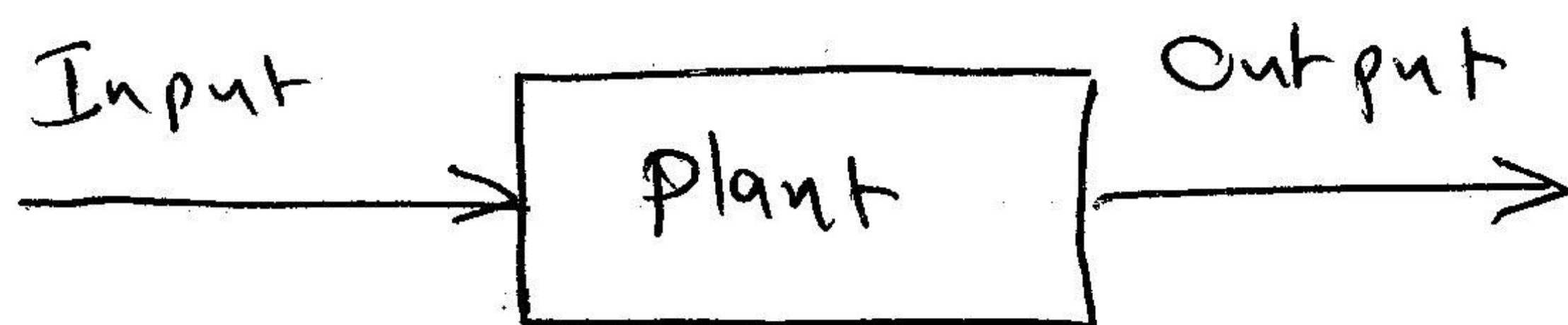
PID Controller



Tuning of PID Controller [Using Ziegler-Nicholas] Methods

[1] The 1st Method (Reaction Line):

In this method, we obtain experimentally the response of the plant to a unit step input (2nd order system).

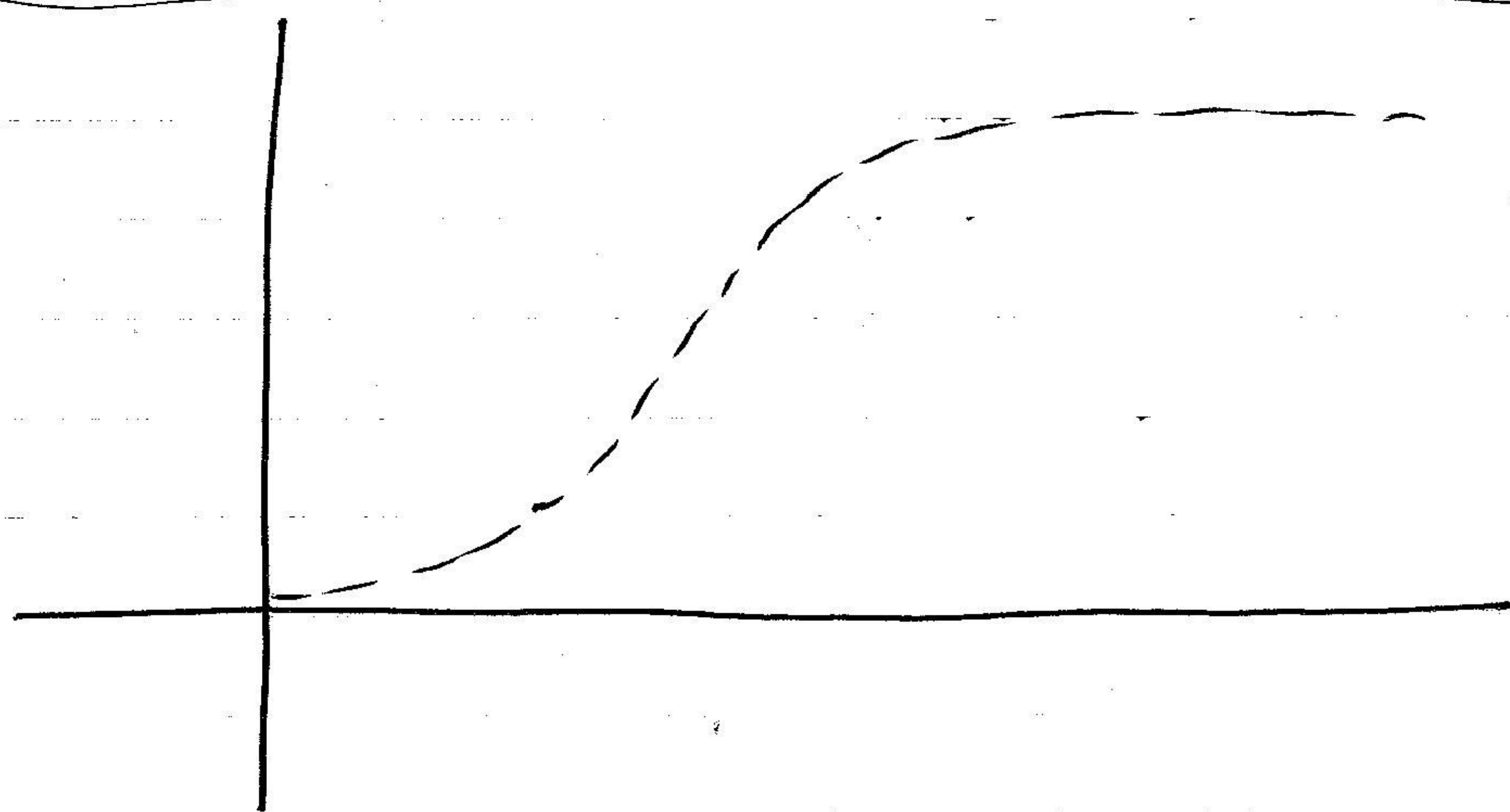


No Feed back
No Controller

* If the plant involves neither integrators nor dominant Complex Conjugate Poles, then the step response curve may look as S-shaped.

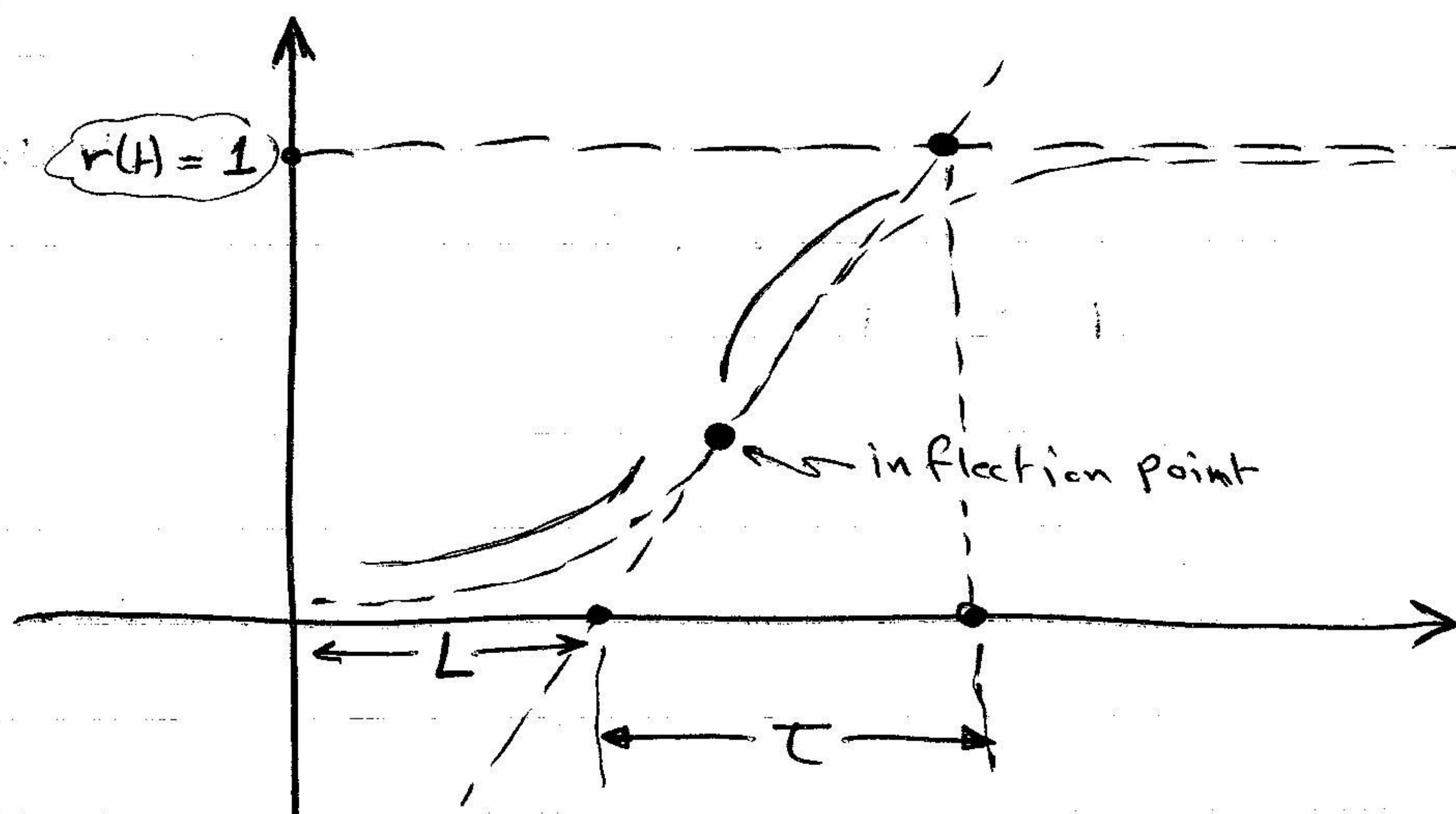
* $\frac{1}{s(s+5)} \propto$ Wrong, because there is an integration $\left(\frac{1}{s}\right)$.

$\frac{1}{s^2+s+1} \propto$ Wrong, because there are Complex Conjugate Poles.



تقریباً و ليس
بالخط
يشبه حرف S

* The delay time & time constant are determined by drawing a tangent line at the inflection point.



L : delay time

T : time constant

Type of Controller	K_p	T_i	T_d
P	$\frac{T}{L}$	∞	Zero
PI	$0.9 \frac{T}{L}$	$\frac{L}{0.3}$	Zero
PID	$1.2 \frac{T}{L}$	$2L$	$0.5L$

Apply to Series ~~PID~~ PID Controller

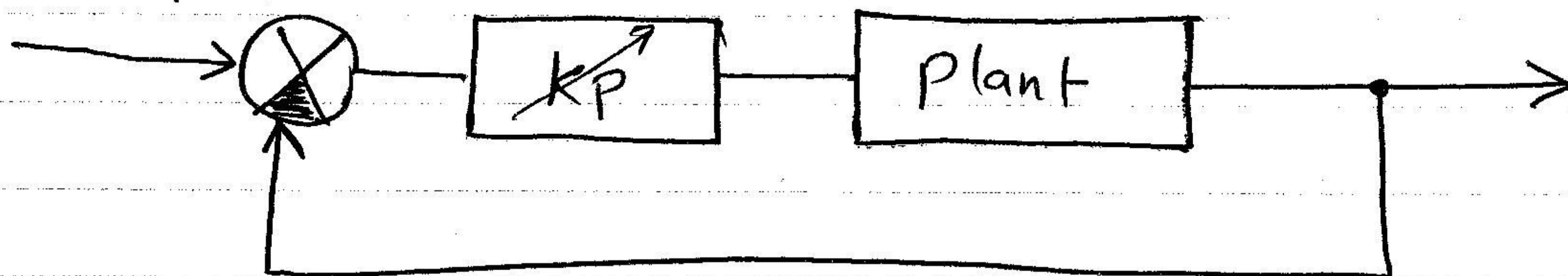
$$\otimes K_i = K_p * \frac{1}{T_i}$$

$$\otimes K_D = K_p * T_D$$

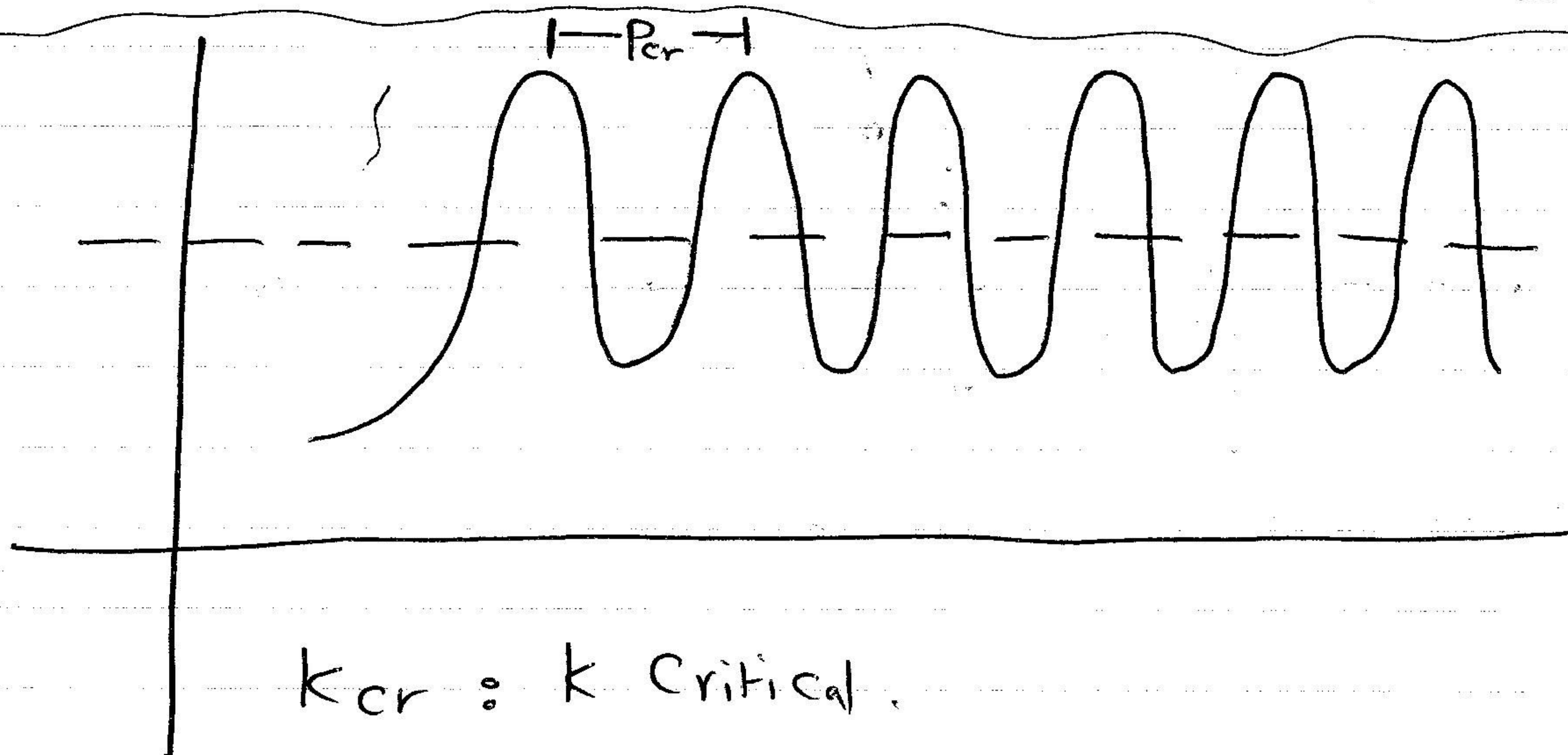
[2] The 2nd Method (Oscillation Rule):

Using the proportional Controller only [P-Controller]

Unit step input



\otimes Increase k from zero to critical value K_{cr} at which the output first exhibits sustained oscillation.



K_{cr} : k Critical.

P_{cr} : Peak Critical.

Type of Controller	K K_p	T_i	T_d
P	$0.5 K_{cr}$	∞	Zero
PI	$0.45 K_{cr}$	$\frac{1}{1.2} P_{cr}$	Zero
PID	$0.6 K_{cr}$	$0.5 P_{cr}$	$0.125 P_{cr}$

11/5 2019

Stability of Linear Feedback System [Dorf ch6]

- A stable system should exhibit a bounded output if the corresponding input is bounded.
- So, a stable system is a dynamic system with a bounded system to a bounded input.
- The stability of a feedback system is directly related to location of the roots of the characteristic equation of the total transfer function $G_T(s) = \frac{\text{output}}{\text{Input}}$

Stability Checking Methods

① Direct Method

In this method, the roots of ch. eq. should be calculated & assigned to the s -plane.

System is said to be "Stable" if all roots located at the left side of s -plane, and one +ve root is enough to make the whole system "Unstable"

Total T.F.
 نص
عین

$$G_T(s) \equiv T(s)$$

e.g. check the stability of the following systems :-

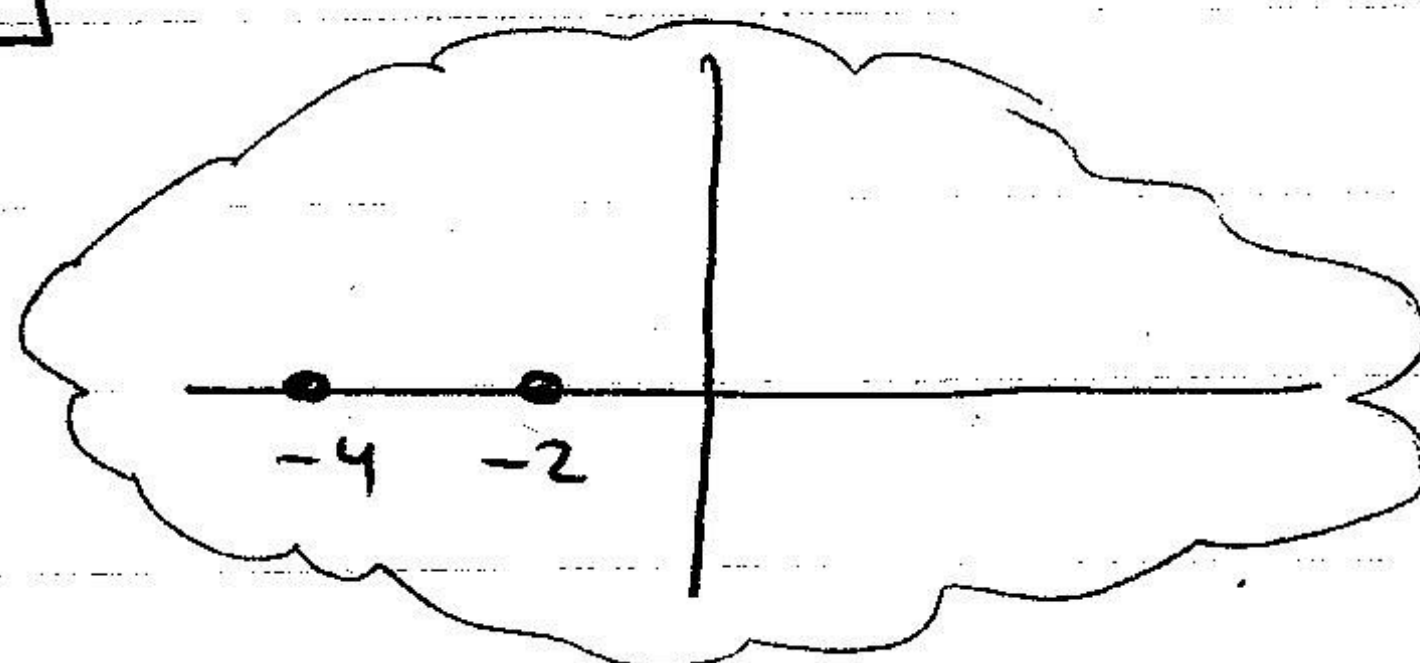
[a] $G_T(s) = \frac{22s}{s^2 + 6s + 8}$

$$\Rightarrow q(s) = s^2 + 6s + 8 = 0$$

$$(s+4)(s+2) = 0$$

$s = -4$ $s = -2$

∴ The system is stable



[b] $Y(s) = \frac{1}{s^2 - 1}$; $X(t) = e^t$

\downarrow

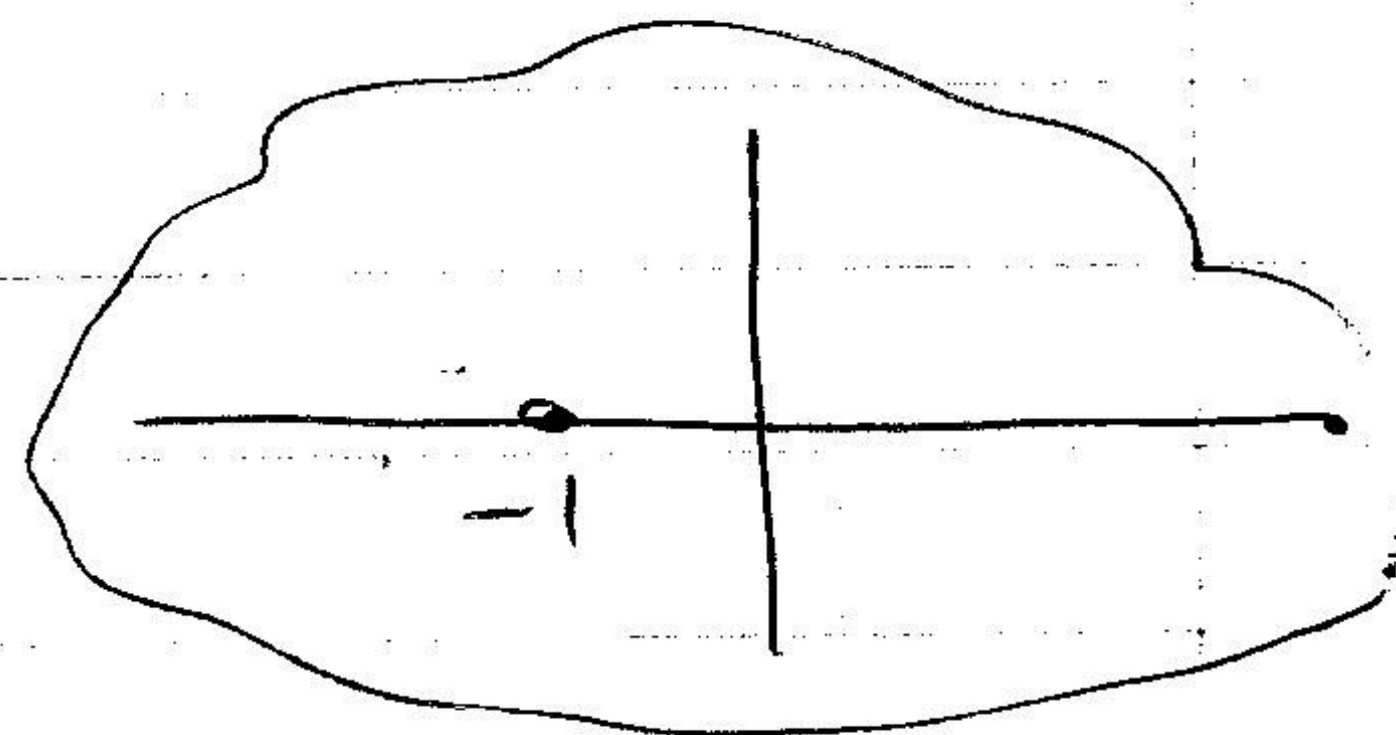
$X(s) = \frac{1}{s-1}$

$$G_T(s) = \frac{Y(s)}{X(s)} = \frac{\frac{1}{s^2 - 1}}{\frac{1}{s-1}} = \frac{1}{(s-1)(s+1)} \cdot \frac{(s-1)}{(s-1)} = \frac{1}{(s-1)(s+1)}$$

$$= \frac{1}{s+1}$$

∴ $q(s) = s+1 = 0 \Rightarrow s = -1$

∴ The system is stable.



[c] $T(s) = \frac{3}{s^3 + 1}$

$$q(s) = s^3 + 1 = 0$$

$$q(s) = s^3 + 1 = 0$$

$$q(s) = (s+1)(s^2 - s + 1) = 0$$

$$\begin{array}{r} s^2 - s + 1 \\ s+1 \overline{) s^3 + 1} \\ \underline{-s^3 + s^2} \\ -s^2 + 1 \\ \underline{+s^2 + s} \\ s+1 \\ \underline{s+1} \\ 0 \end{array}$$

$$s^2 - s + 1 = 0$$

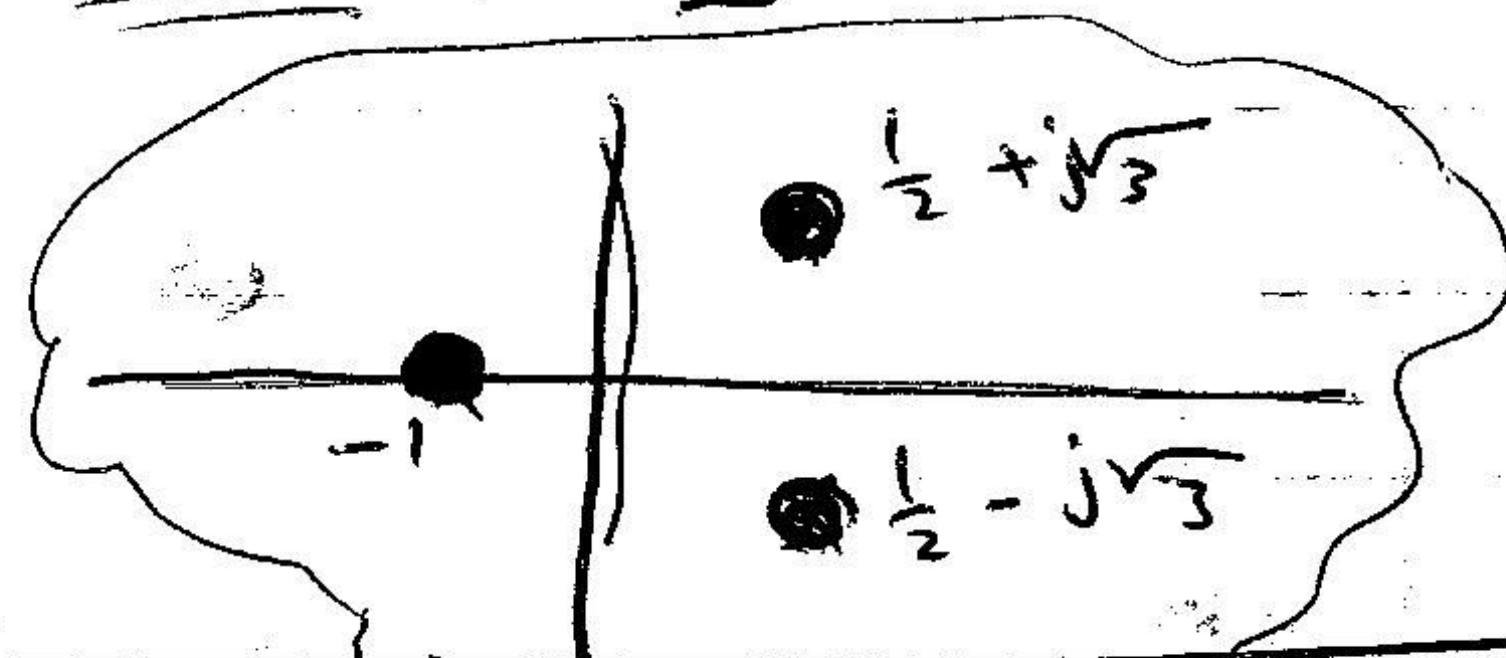
general formula

$$\frac{-b \pm \sqrt{b^2 - 4ac}}{2a}$$

$$\frac{1 \pm \sqrt{1 - 4}}{2} = \frac{1 \pm j\sqrt{3}}{2} \Rightarrow \left[\frac{1}{2} + j\frac{\sqrt{3}}{2}, \frac{1}{2} - j\frac{\sqrt{3}}{2} \right]$$

$$\therefore \text{Total roots} = \left[-1, \frac{1}{2} + j\frac{\sqrt{3}}{2}, \frac{1}{2} - j\frac{\sqrt{3}}{2} \right]$$

\therefore The system is Unstable



2 Response to Impulse function

a linear system is stable if and only if the absolute value of its impulse response $g(t)$ integrated over an infinite range is finite.

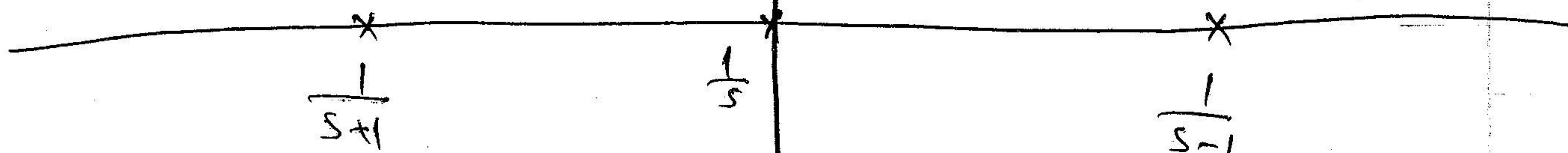
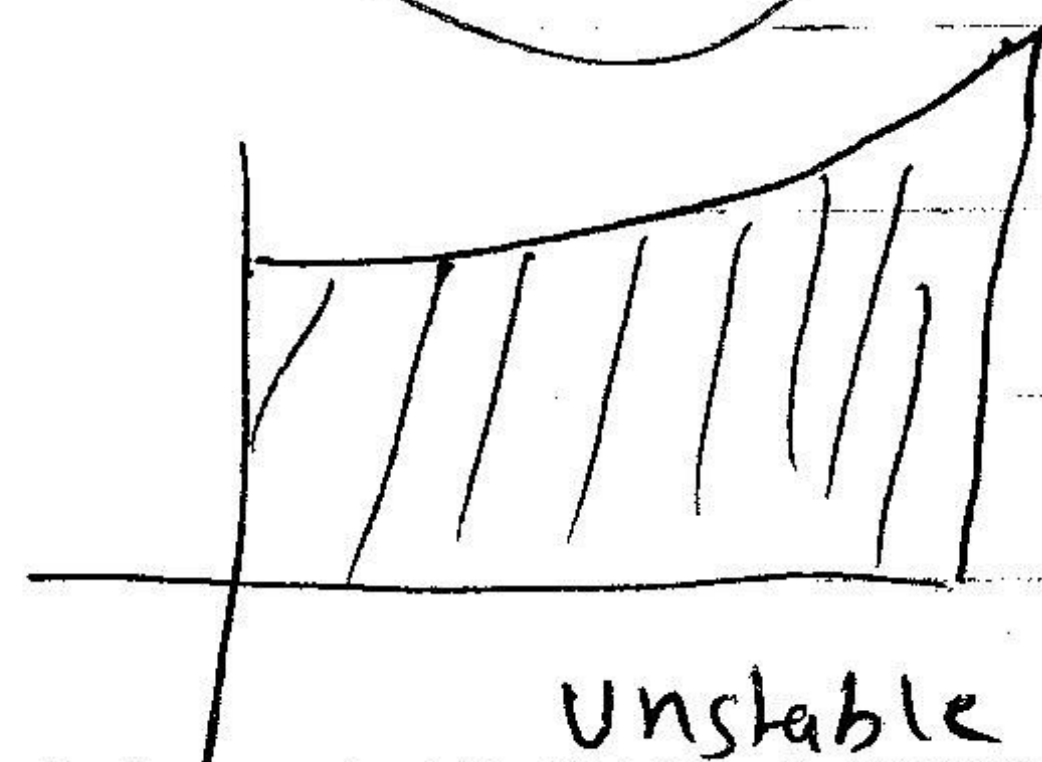
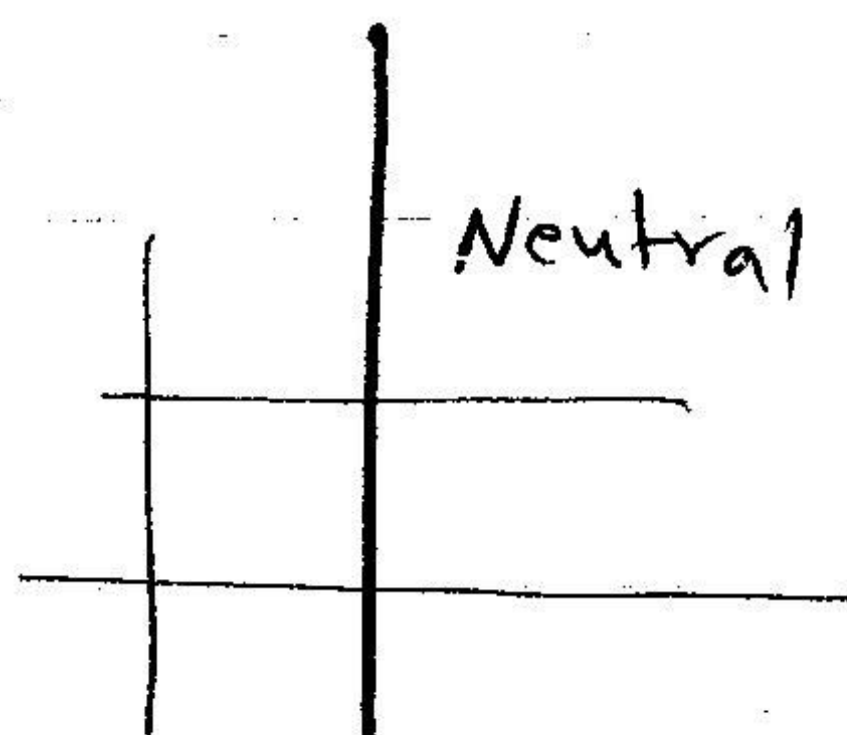
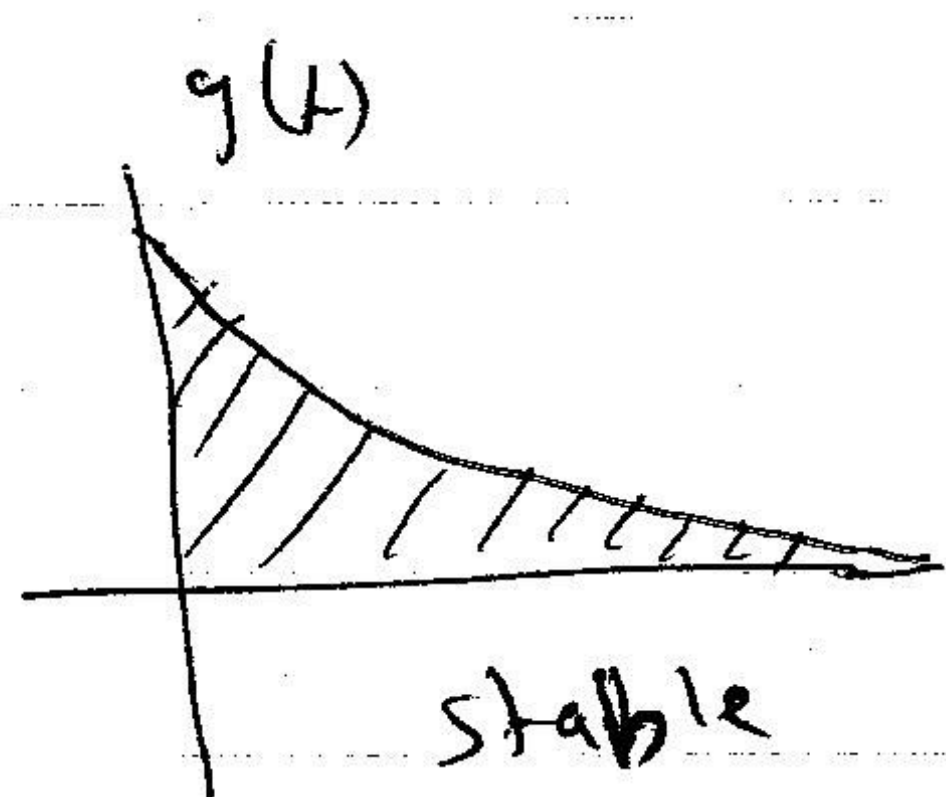


$$\int_0^{\infty} g(t) dt = \text{Finite}$$

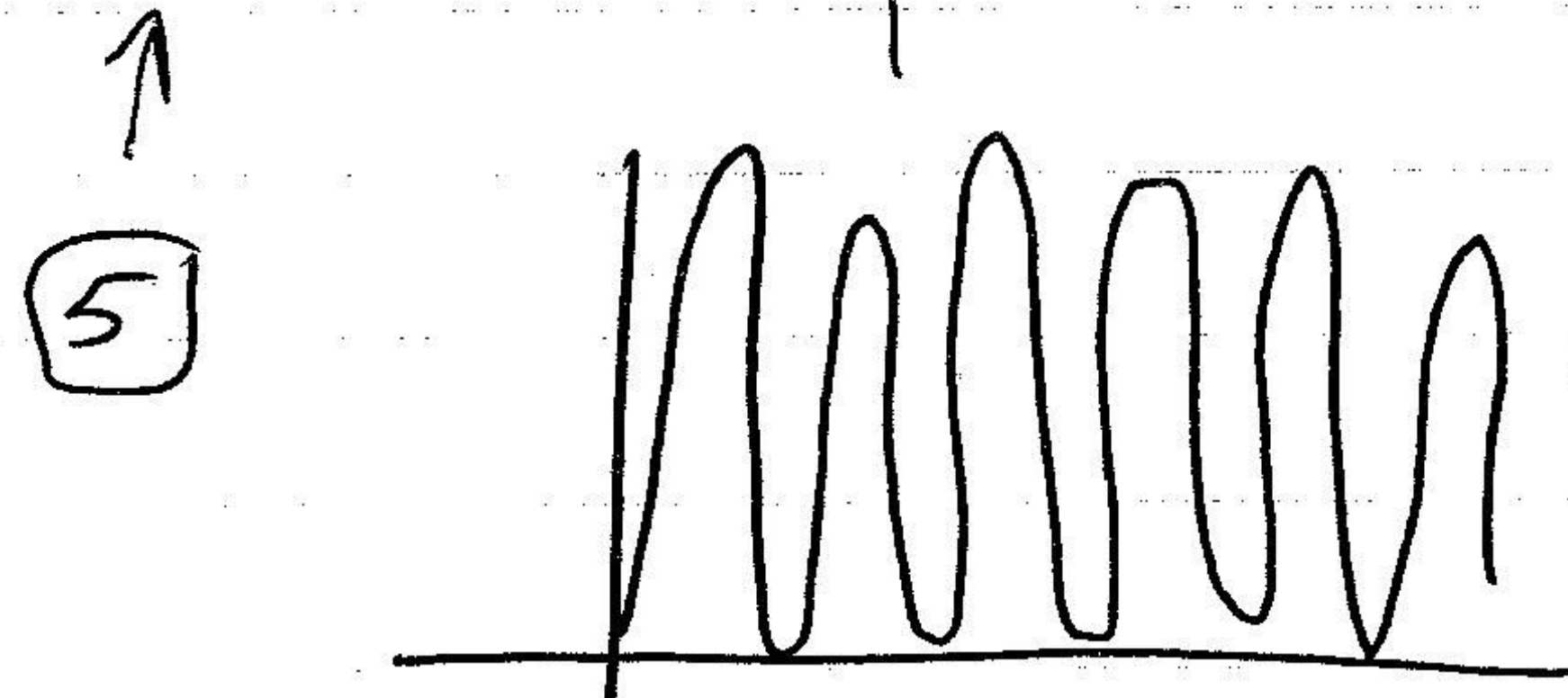
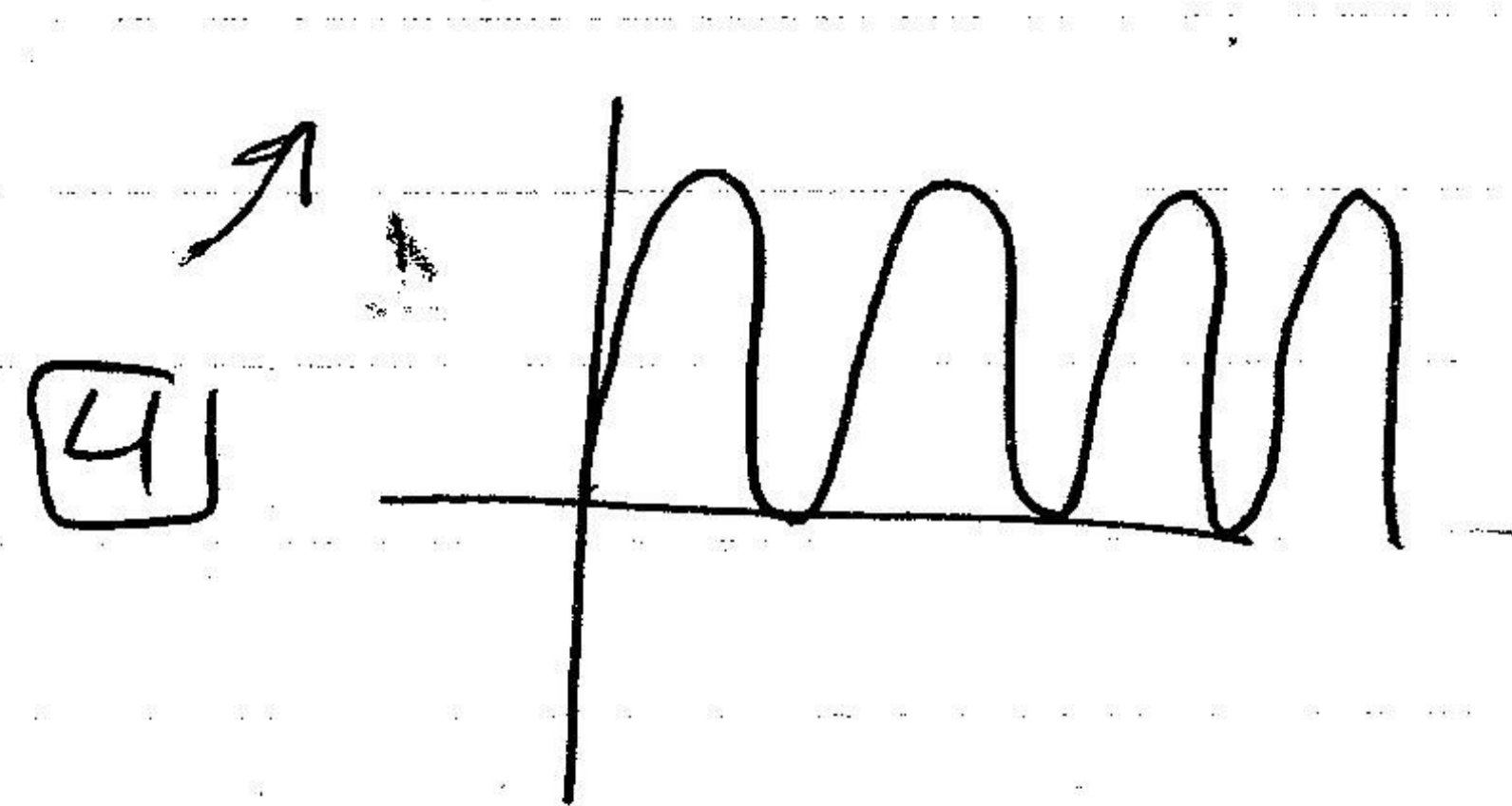
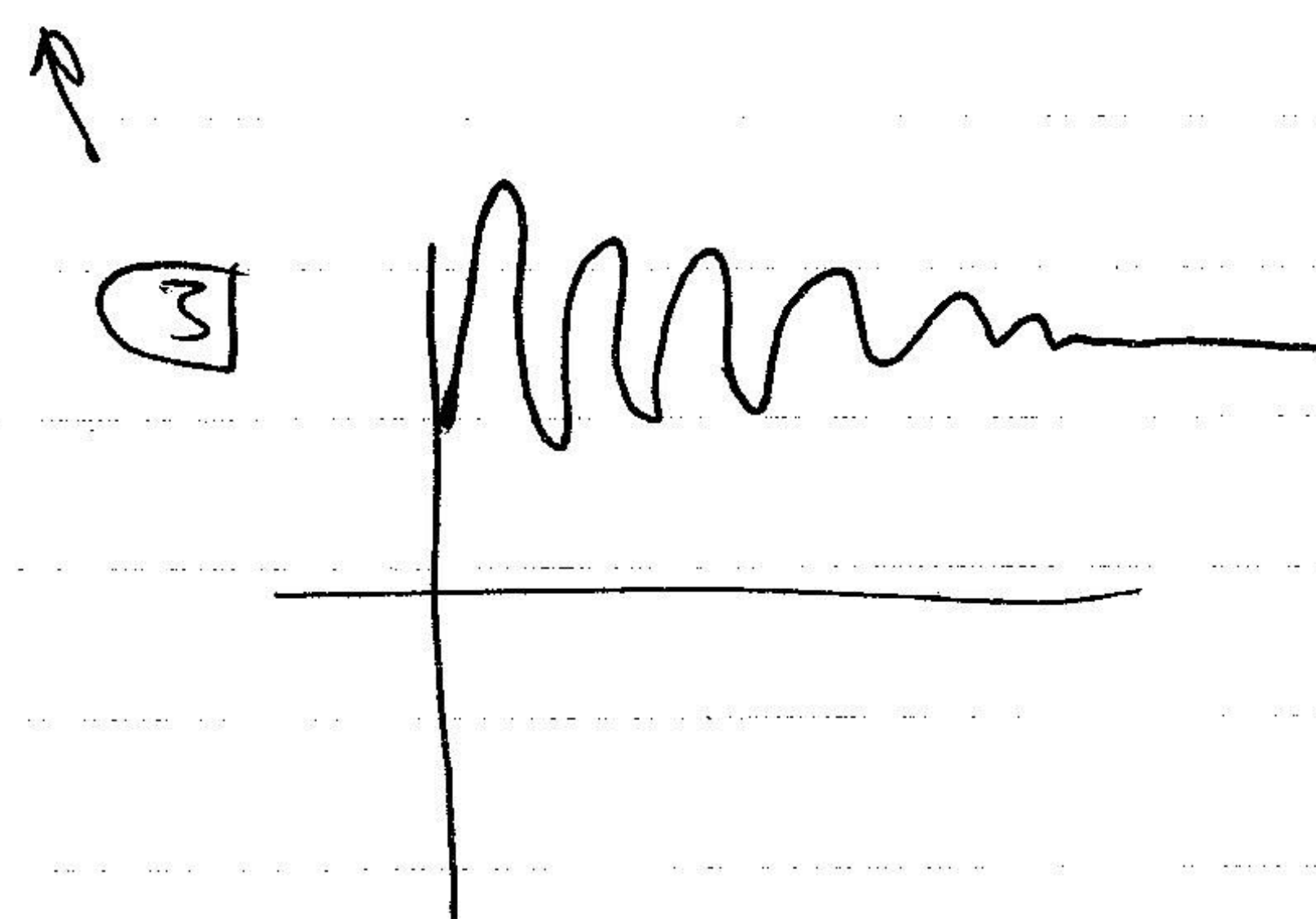
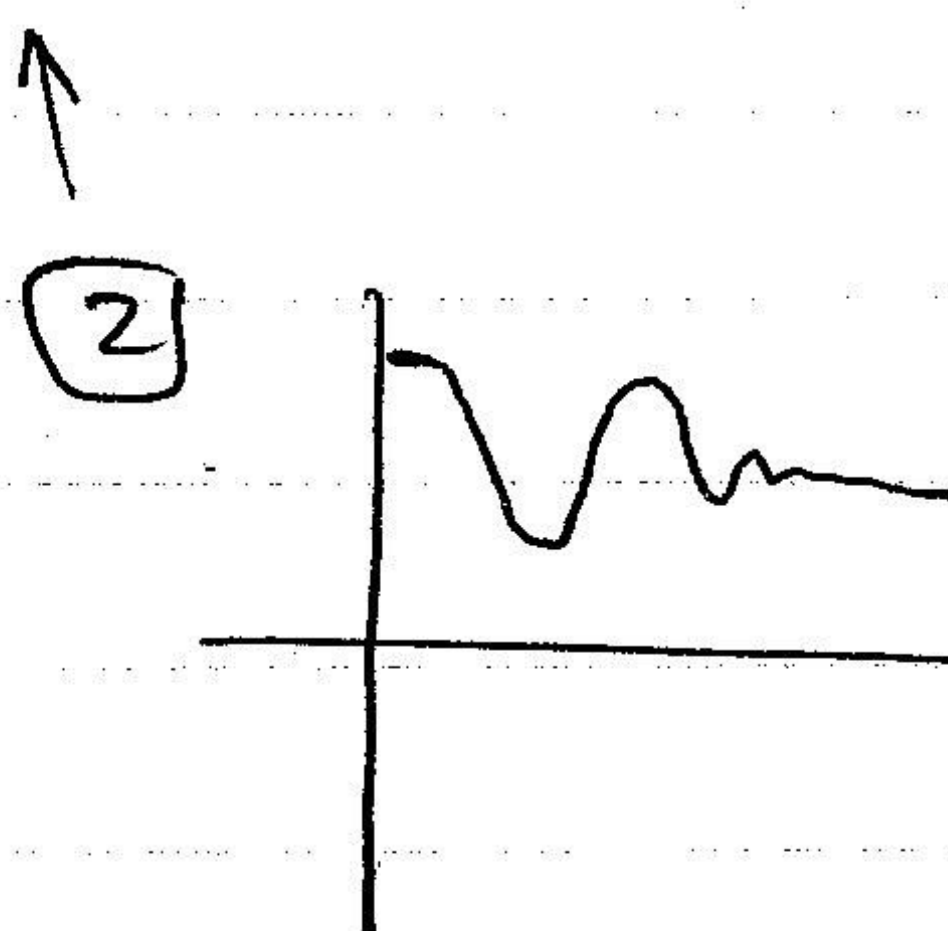
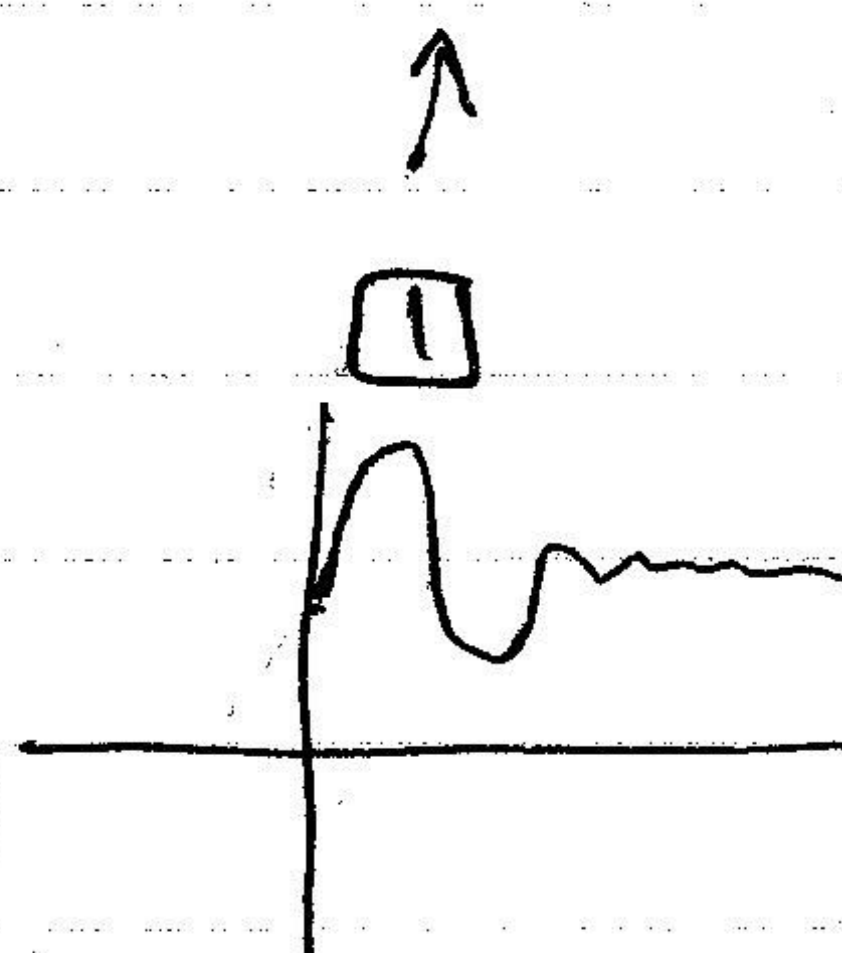
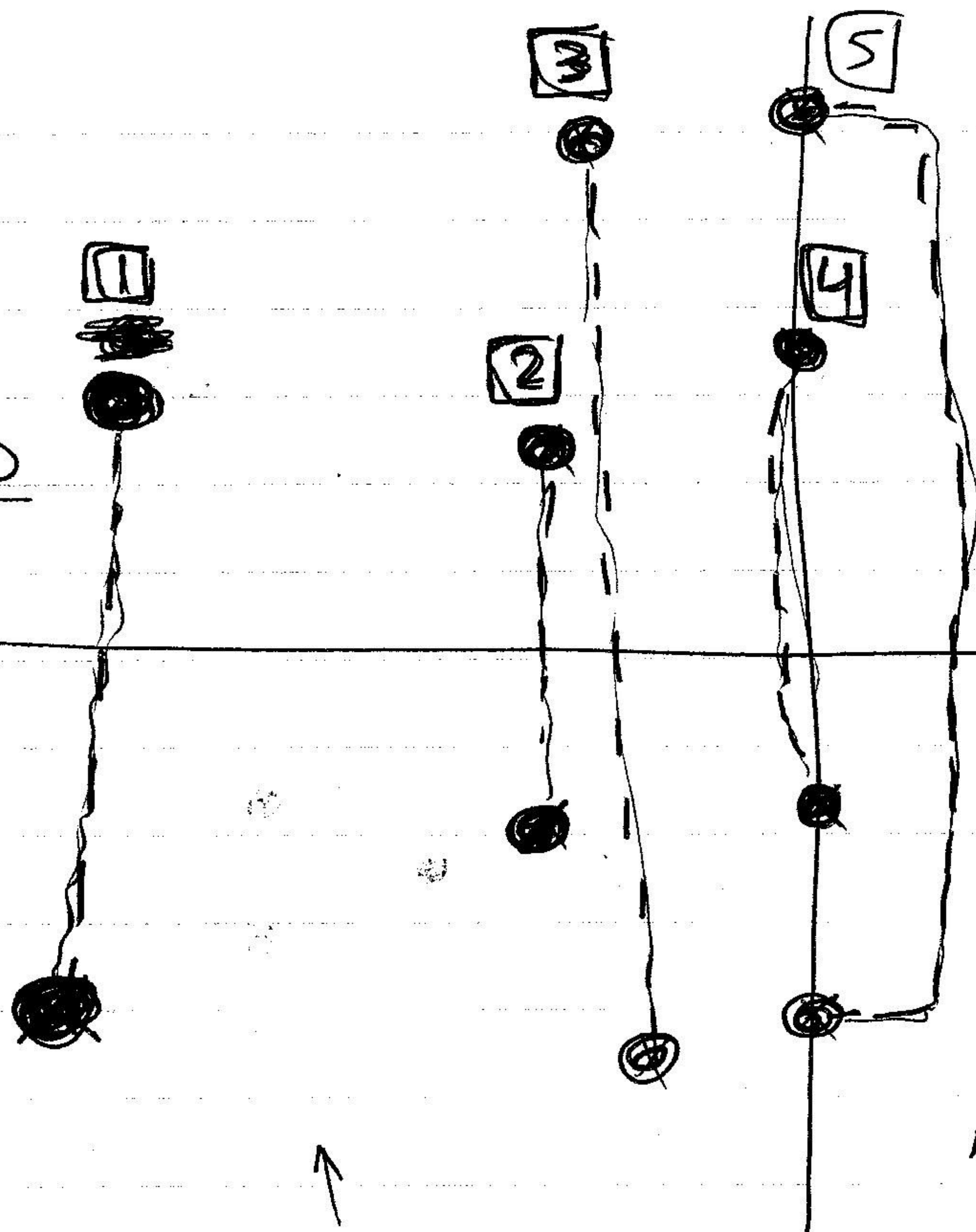
(محدود)

input	Lablace
$\delta(t)$	1
1	$1/s$
t	$1/s^2$
t^2	$2/s^3$
t^3	$6/s^4$

1st order



2nd order:



غ: 5: تیزید و سداد فقط
 4: سدا و سرفه منه

3 Rouths-Hurwitz Stability Criteria :

- This technique allowed ^{to} us to compute the number of roots of the characteristic equation in the right half plane without actually computing the values of roots.
- This gives us a design method for determining values of certain system that will lead to close loop stability.
- This method is based on ordering coefficients of the ch. eq. into an array as follows :-

$$Q(s) = a_n s^n + a_{n-1} s^{n-1} + \dots + a_1 s + a_0$$

s^n	a_n	a_{n-2}	a_{n-4}	—
s^{n-1}	a_{n-1}	a_{n-3}	a_{n-5}	—
s^{n-2}	b_{n-1}	b_{n-3}	b_{n-5}	—
\vdots	c_{n-1}	c_{n-3}	c_{n-5}	—
\vdots				
s				
1	h_{n-1}			

$$b_{n-1} = \frac{a_{n-1} a_{n-2} - a_n a_{n-3}}{a_{n-1}} = \frac{-1}{a_{n-1}} \begin{vmatrix} a_n & a_{n-2} \\ a_{n-1} & a_{n-3} \end{vmatrix}$$

$$b_{n-3} = \frac{a_{n-1} a_{n-4} - a_n a_{n-5}}{a_{n-1}} = \frac{-1}{a_{n-1}} \begin{vmatrix} a_n & a_{n-4} \\ a_{n-1} & a_{n-5} \end{vmatrix} \quad (43)$$

$$C_{n-1} = \frac{-1}{b_{n-1}} \begin{vmatrix} a_{n-1} & a_{n-3} \\ b_{n-1} & b_{n-3} \end{vmatrix}$$

• The Routh's Criteria ~~states~~ States that the no of roots of $q(s)$ with (+ve) real part is equal to the no of change in sign of the first column.

So, the system is stable if there is no change in Sign of the First Column.

ex

2	1	3
4	5	6
7	8	4
1	2	1
2	2	2

Stable
There is no change in sign

2	2	3
2	1	2
5	-1	6
-6	7	2
-1	5	6

Unstable
There is change in sign

2	2	1
1	1	1
-1	2	1
2	2	
1		

Unstable
There is change in sign

-1	5	3
-2	1	2
-1	1	1
-4		

Stable
because there is no change in sign

CASE 1 No element in the 1st Column is Zero

Ex Check the stability of the 2nd order system represented by $q(s) = as^2 + bs + c$

s^2	a	c
s	b	0
1	$-\frac{bc}{b} = \boxed{c}$	

The system is stable when :-

① $a > 0 \quad b > 0 \quad c > 0$

the system is stable for all values of a, b, c is greater than zero (all positive)

or

② the system is stable for all values of a, b, c is less than zero (all negative)

$a < 0 \quad b < 0 \quad c < 0$

Ex $q(s) = a_3 s^3 + a_2 s^2 + a_1 s + a_0$

s^3	a_3	a_1
s^2	a_2	a_0
s^1	$\frac{a_2 a_1 - a_3 a_0}{a_2} \quad 0$	
1	a_0	

① Stable if
 $a_3 > 0$
 $a_2 > 0$
 $a_0 > 0$
 $a_2 a_1 > a_3 a_0$

If ~~stable~~
 ① all +ve $\frac{a_2 a_1 - a_3 a_0}{a_2} > 0$

$a_2 a_1 > a_3 a_0$

② if all -ve

Stable if

$$a_3 < 0$$

$$a_2 < 0$$

$$a_0 < 0$$

$$a_2 a_1 > a_3 a_0$$

$$\frac{a_2 a_1 - a_3 a_0}{a_2} < 0$$

when multiply $a_2 \neq 0$

$a_2 < 0$

so the sign $<$ become $>$

$\therefore a_2 a_1 > a_3 a_0$

Ex $S^3 + S^2 + 2S + 24$

Sol $S^3 + S^2 + 2S + 24 = 0$

$$(S - 1 + j\sqrt{7})(S - 1 - j\sqrt{7})(S + 3) = 0$$

$$S = \underline{\underline{1 - j\sqrt{7}}}$$

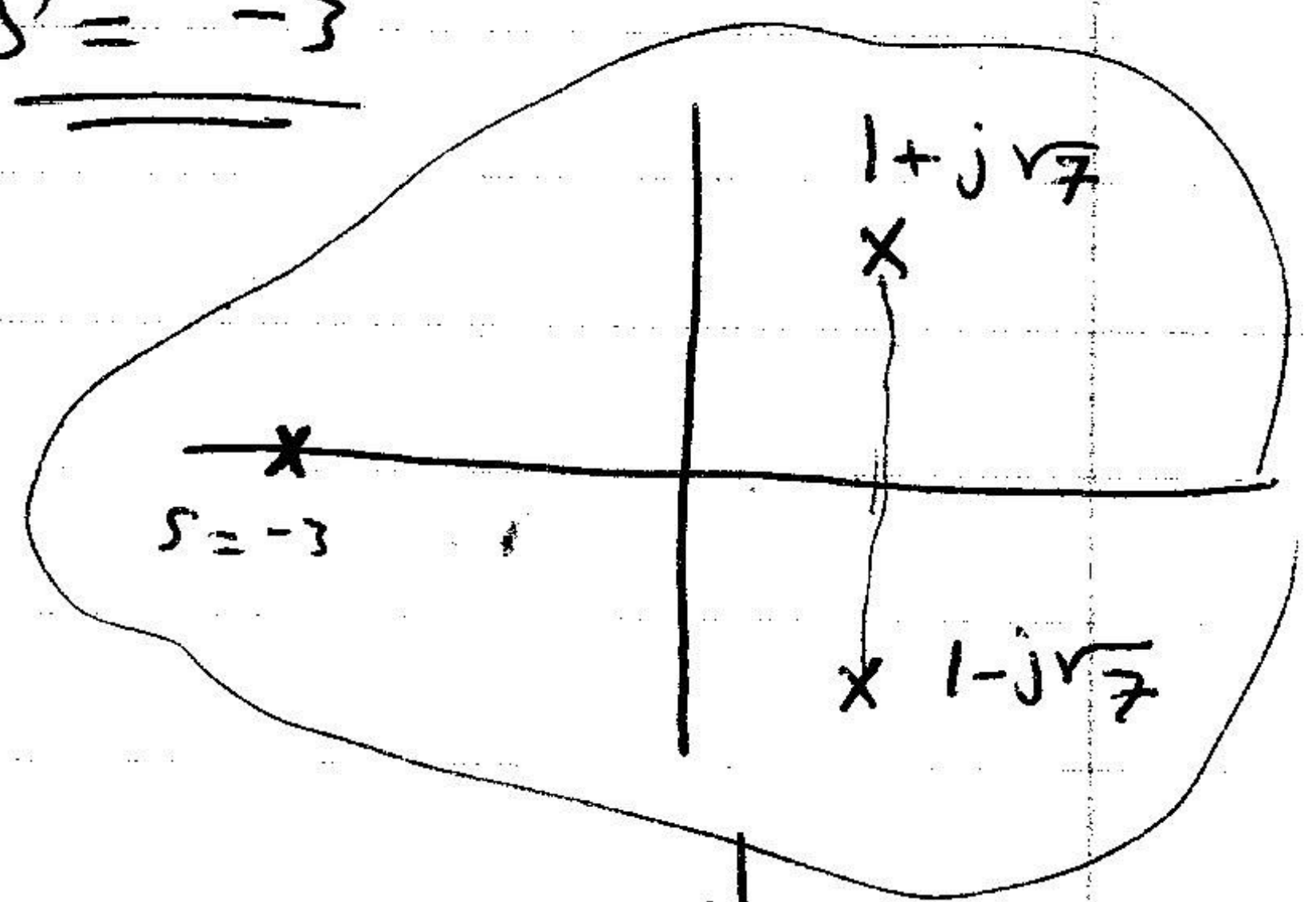
$$S = \underline{\underline{1 + j\sqrt{7}}}$$

$$S = \underline{\underline{-3}}$$

Solve by Routh's:-

S^3	1	2
S^2	1	24
S^1	-22	0
1	24	

\swarrow root
 \searrow root



~~Stable~~
Unstable

\therefore The system is Unstable!

CASE 2 : There is a zero in the 1st Column, but Some other elements of the row containing Zero are non-zero

* If there is only one zero after completing the array it might be replaced by $[\underline{\epsilon}]$

ϵ : is a small positive number approaches to Zero (0^+)

1	2	3
0	2	4
5	6	3

ϵ : Epsilon

EX Consider the following characteristic polynomial :-

$$q(s) = s^5 + 2s^4 + 2s^3 + 4s^2 + 11s + 10$$

s^5	1	2	11
s^4	2	4	10
s^3	$\epsilon^{2 \times 0^+}$	6	0
s^2	$\frac{-12}{\epsilon}$	10	0
s	6	0	
1	10		

Cell 4X1 $\frac{4\epsilon - 12}{\epsilon} = \frac{-12}{\epsilon}$

Cell 4X2 $\frac{10\epsilon - 0}{\epsilon} = 10$

Cell 5X1 $\frac{\frac{-12}{\epsilon} \times 6 - 10\epsilon}{\frac{-12}{\epsilon}} = \frac{-72 - 10\epsilon}{-12} = \frac{-72 - 0}{-12} = 6$

Cell 5X2 $\frac{\frac{-12}{\epsilon} \times 0 - 0 \times \epsilon}{\frac{-12}{\epsilon}} = 0$

(47)

∴ The system is Unstable!

2 +ve roots

2 change in sign

Ex $q(s) = s^4 + s^3 + s^2 + s + k$

s^4	1	1	k
s^3	1	1	0
s^2	ϵ	k	0
s^1	$\frac{\epsilon - k}{\epsilon}$	0	0
1	k		

⊛ To be stable :-

$$\frac{\epsilon - k}{\epsilon} > 0 \Rightarrow \frac{-k}{\epsilon} > 0 \Rightarrow -k > 0 \Rightarrow \boxed{k < 0}$$

$$\boxed{k > 0}$$

∴ The system is Unstable for any value of k except for $k=0$, the system then is said to be marginally stable.

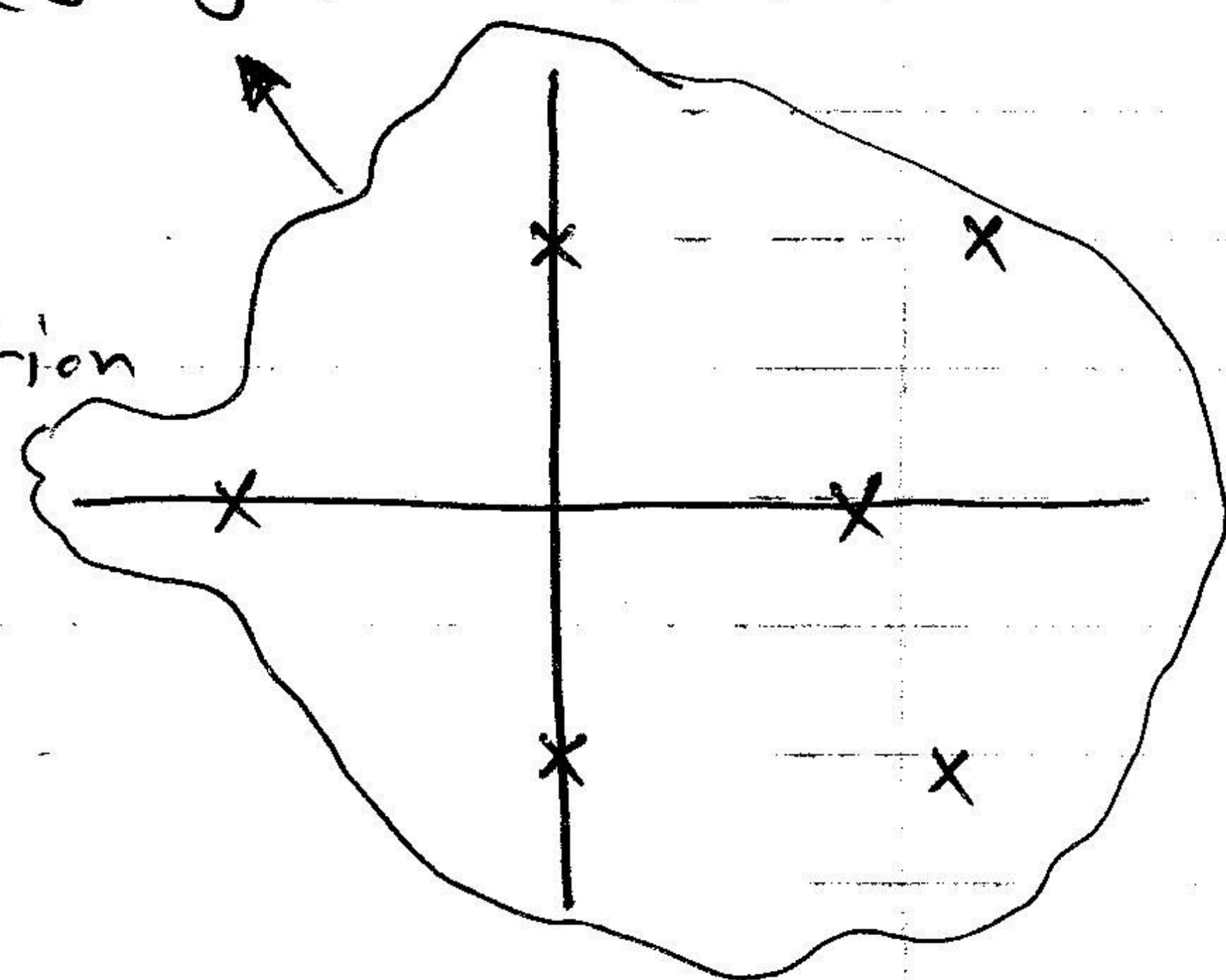
CASE 3 : There is a zero in the 1st Column, and other elements of the row containing zero are zeros

* This condition occurs when polynomial contains singularities that are symmetrical allocated about the origin.

There for, a factors such as :

$$(S+6)(S-6) \quad \text{or} \quad (S+jw)(S-jw) \quad \text{occur.}$$

* by utilizing the auxiliary equation $U(s)$: which is immediately precedes the zero entry in Routh array



Ex : Consider the following polynomial

$$S^3 + 2S^2 + 4S + K$$

S^3	1	4
S^2	2	K
<hr/>		
S^1	$\frac{8-K}{2}$	
1	K	

To be stable :-

$$K > 0$$

$$\frac{8-K}{2} > 0$$

$$\frac{8-K}{2} > 0 \Rightarrow 8-K > 0$$

$$\Rightarrow K < 8$$

$$\Rightarrow \boxed{0 < K < 8}$$

but if $k = 8$:-

$$\frac{8-k}{2} \Rightarrow \frac{8-8}{2} = \frac{0}{2} = \boxed{0}$$

s^3	1	4
s^2	2	k
s^1	0	0
1	k	

The Auxiliary equation (The row above the zeros)

~~Let $s^2 + 2s + k = 0$~~

$$U(s) = 2s^2 + k = 0$$

$$\longrightarrow \begin{array}{c|cc} & 1 & \\ \hline s^2 & 2 & k \end{array}$$

$$\Rightarrow 2s^2 + 8 = 0$$

$$s^2 + 4 = 0 \Rightarrow \boxed{s = \pm 2j}$$

Note :- The Auxiliary equation must be s^2, s^4, s^6

The power of s must be even

Ex :- Consider the characteristic polynomial & find the intersection point with Imagining axis :-

$$q(s) = s^5 + s^4 + 4s^3 + 24s^2 + 3s + 63$$

s^5	1	4	3	0
s^4	1	24	63	0
s^3	-20	-60	0	
s^2	+21	63	0	
s^1	0	0	0	
1				

$$\begin{aligned} & \frac{1}{-20} (-20 \times 24 - -60) \\ &= (-480 + 60) \frac{1}{-20} \\ &= +420 \times \frac{1}{+20} = \boxed{21} \end{aligned}$$

$$\begin{aligned} & \frac{-20 \times 63 - 1 \times 0}{-20} \\ &= \frac{-20 \times 63}{-20} = \boxed{63} \end{aligned}$$

$$\begin{aligned} & \frac{(21)(-60) - (-20)(63)}{21} \\ &= \frac{-1260 - -1260}{21} \\ &= \frac{-1260 + 1260}{21} \\ &= \frac{0}{21} = \boxed{0} \end{aligned}$$

Auxiliary Equation :-

$$21s^2 + 63 = 0$$

$$21s^2 = -63 \Rightarrow s^2 = \frac{-63}{21} = -3$$

$$\therefore s = \pm \sqrt{-3} j$$

CASE 4 : Repeated Roots of the ch. equ. on the (jw) axis

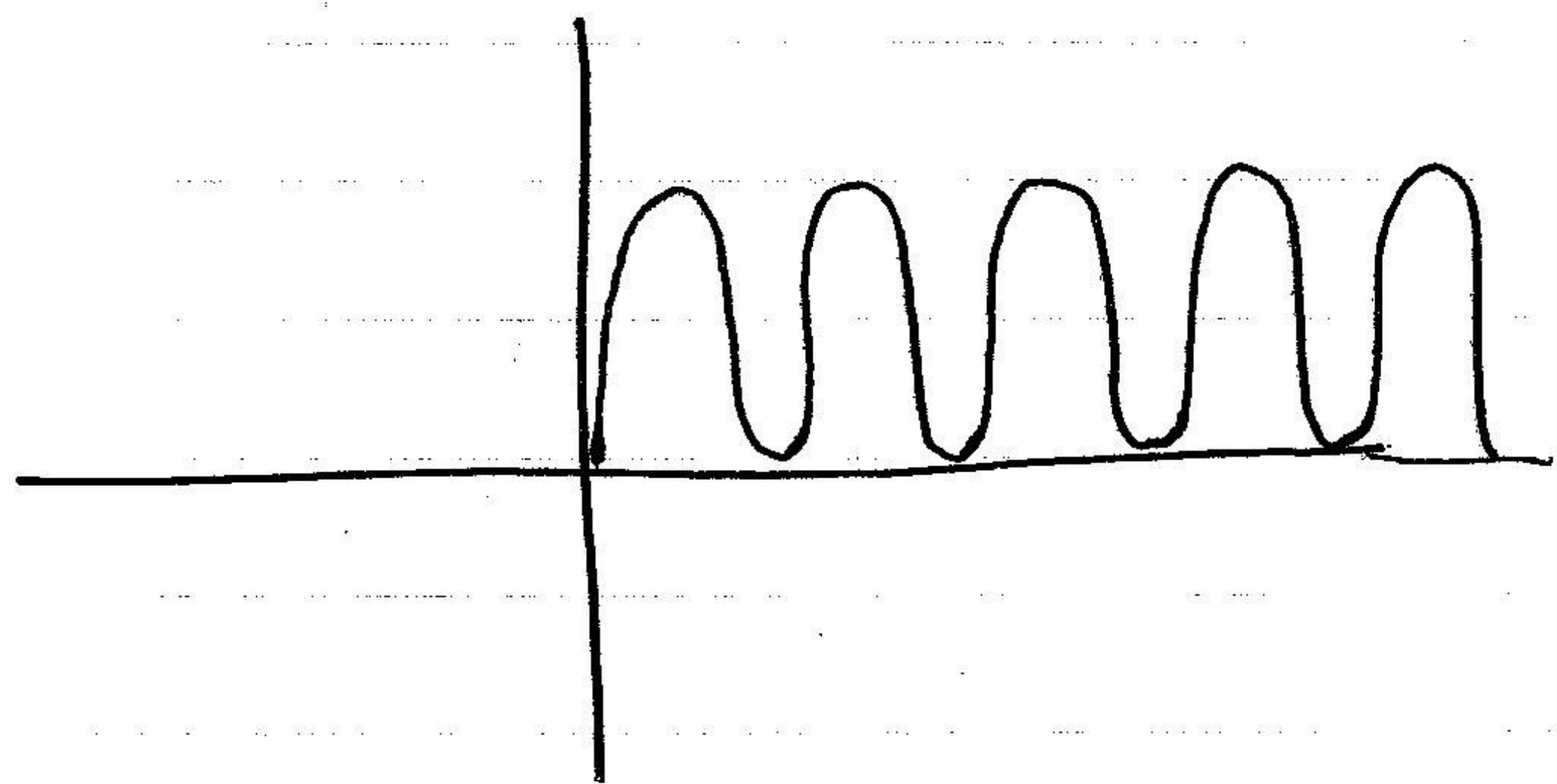
* If the (jw) axis roots are simple, the system is neither stable nor unstable.

* Its instead called marginally stable, since

it has an undamped sinusoidal.

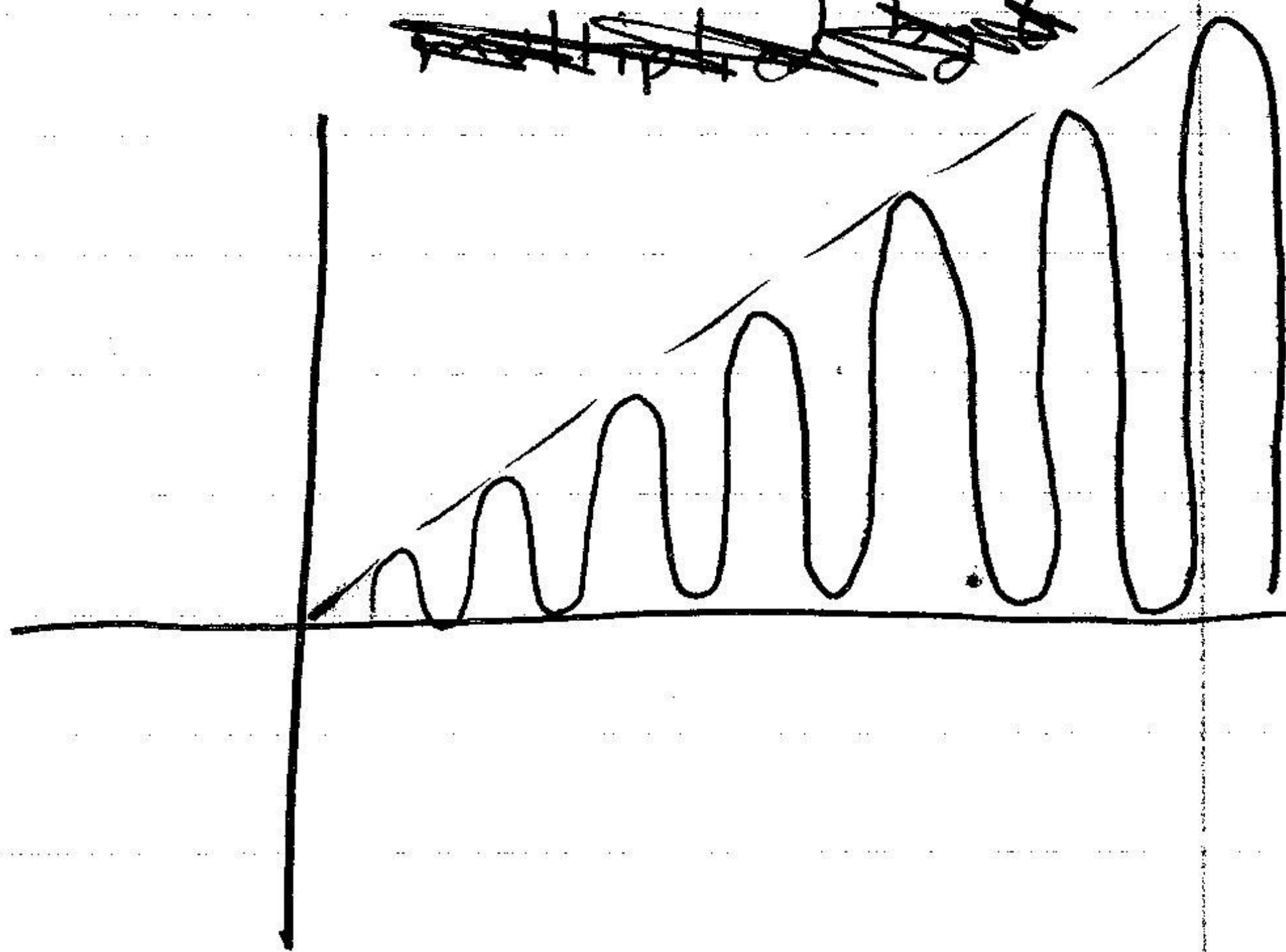
⊛ If the $(j\omega)$ axis roots are repeated, the system response will be Unstable, with a form of $t \sin(\omega t + \theta)$, thus Routh's table cannot reveal this form of instability.

~~the system is stable~~



Undamped Sinusoidal

~~the system is stable~~



when multiply the Undamped Sinusoidal by t

Ex $q(s) = s^5 + s^4 + 2s^3 + 2s^2 + s + 1$

s^5	1	2	1
s^4	1	2	1
s^3	ϵ	ϵ	0
s^2	1	1	0
s^1	ϵ	0	0
1	1		

$$\frac{2\epsilon - \epsilon}{\epsilon} = \frac{\epsilon}{\epsilon} = \boxed{1}$$

$$\frac{\epsilon - 0}{\epsilon} = \boxed{1}$$

∴ In this case, the Routh's table failed to reveal if the system is unstable or not

* So, we must take the Auxiliary equation:-

$$\begin{array}{c|ccc} s^4 & 1 & 2 & 1 \\ \hline s^0 & 1 & 2 & 1 \end{array}$$

$$s^4 + 2s^2 + 1 = 0$$

$$(s^2 + 1)(s^2 + 1) = 0$$

$$\begin{array}{cc} \downarrow & \downarrow \\ s = \pm j & s = \pm j \end{array}$$

∴ The system is Unstable, due to ~~not duplication~~.

Imaginary root duplication

∴ imaginary part is repeated
Zero is repeated \rightarrow the system is Unstable

Ex $q(s) = s^3 + s + 7$

s^3	1	1
s^2	ϵ	7
s^1	$\frac{-7}{\epsilon}$	0
s^0	7	$-\infty$

$$\frac{\epsilon - 7}{\epsilon} \Rightarrow \frac{-7}{\epsilon}$$

the system is unstable

Ex $q(s) = s^2 + s + 1$

s^2	1	1
s	1	0
1	1	

\therefore the system is stable

Ex $q(s) = s^2 + 1$

s^2	1	1
s	ϵ	0
1	1	

$$\frac{(\epsilon)(1) - (1)(0)}{\epsilon} = \frac{\epsilon}{\epsilon} \neq 1$$

Stable
or marginally stable ($\omega > 0$)

Ex $q(s) = s^2 - 1$

s^2	1	-1
s	ϵ	0
1	-1	

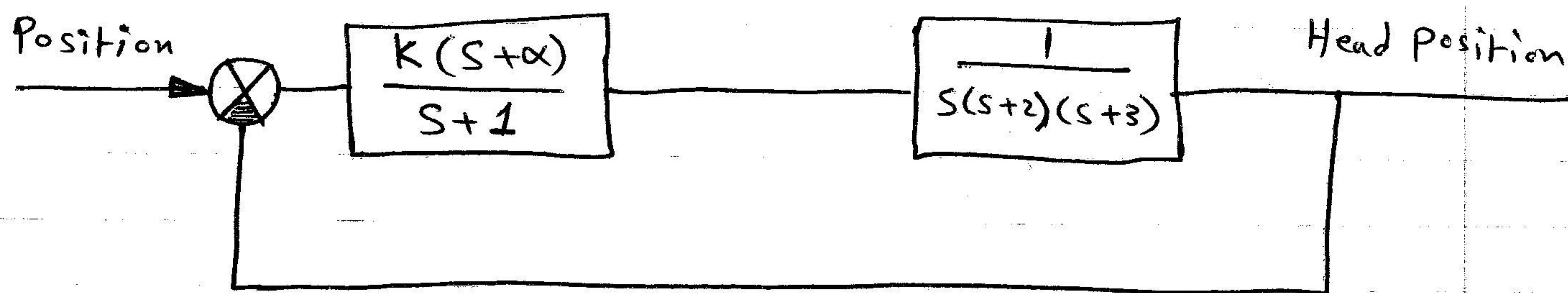
$$\frac{-\epsilon - 0}{\epsilon} = \frac{-\epsilon}{\epsilon} = \boxed{-1}$$

Unstable (there is a change of sign)

Ex: a large welding Robots are used in today's auto plants. the ~~welding~~ welding head is moved to different positions on the auto body, and a rapid accurate response is required.

⊗ a block diagram of the welding head position is shown below.

⊗ We must desire to determine the range of k and α for which the system is stable.



Solution :-

$$T.F. = \frac{K(s+\alpha)}{(s+1)(s)(s+2)(s+3)}$$

$$1 + \frac{K(s+\alpha)}{(s+1)(s)(s+2)(s+3)}$$

$$= \frac{K(s+\alpha)}{(s+1)(s)(s+2)(s+3) + K(s+\alpha)}$$

$$Q(s) [P.L.] = S^4 + 6S^3 + 11S^2 + (K+6)S + K\alpha = 0$$

ملاحظة هامة: - $q(s)$ هو مقام $T.F$

s^4	I	II	$k\alpha$
s^3	6	$k+6$	0
s^2	$\frac{60-k}{6}$	$k\alpha$	0
s^1	B_3	0	0
1	$k\alpha$		

$$\frac{66 - (k+6)}{6}$$

$$= \frac{66 - k - 6}{6}$$

$$= \boxed{\frac{60 - k}{6}}$$

$$\frac{6k\alpha - 0}{6}$$

$$= \boxed{k\alpha}$$

$$\frac{\frac{60-k}{6} (k+6) - 6k\alpha \times \frac{6}{6}}{\frac{60-k}{6}}$$

$$= \boxed{\frac{(60-k)(k+6) - 36k\alpha}{60-k}}$$

$$= \boxed{B_3}$$

CASE 1) > 0

The system is stable when:-

$$\boxed{\frac{60-k}{6} > 0}$$

$$\boxed{B_3 > 0}$$

$$\boxed{k\alpha > 0}$$

$$\textcircled{1} \frac{60-k}{6} > 0$$

$$\Rightarrow 60 - k > 0$$

$$\Rightarrow 60 > k \Rightarrow \boxed{k < 60}$$

$$\textcircled{2} k\alpha > 0$$

$$\therefore \boxed{k > 0}, \boxed{\alpha > 0}$$

$$(3) \quad B_3 > 0$$

$$\frac{(60-k)(k+6) - 36k\alpha}{60-k} > 0$$

$$(60-k)(k+6) - 36k\alpha > 0 \quad \times (-)$$

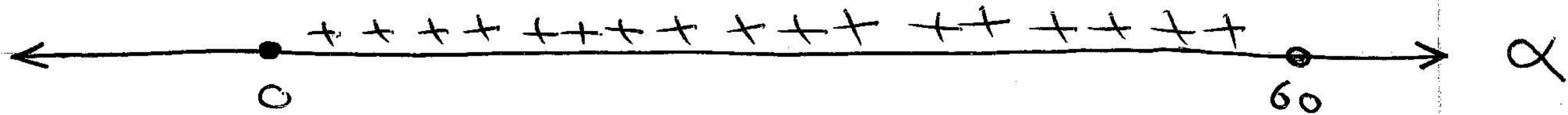
$$(k-60)(k+6) + 36k\alpha \leq 0$$

$$(k-60)(k+6) \leq -36k\alpha$$

$$\frac{(k-60)(k+6)}{-36k} \leq \frac{-36k\alpha}{-36k}$$

عند القيمة سالبة
تفكس إشارة المتباينة

$$\Rightarrow \frac{(60-k)(k+6)}{36k} \geq \alpha$$



∴ The system is stable when :-

$$(1) \quad 0 < k < 60 \quad [k \text{ is positive}]$$

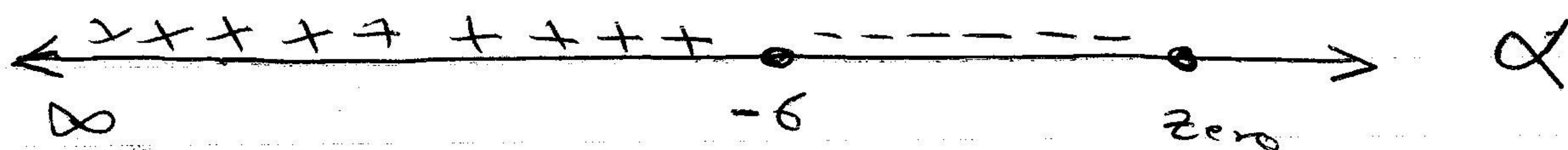
$$(2) \quad \alpha \leq \frac{(60-k)(k+6)}{36k} \quad [\alpha \text{ is positive}]$$

CASE 2 $\alpha < 0$

System is stable when

$$k < 0 \quad \alpha < 0$$

$$\frac{(60-k)(k+6)}{36k} \geq \alpha$$



∴ The system is stable when :-

① $-6 < k < 60$

② $\alpha \leq \frac{(60-k)(k+6)}{36k}$

∴ إلى هنا مادة الفيزياء

Until Here ∴ The 1st Exam

Material ∴

” بالتوفيق للجميع
 “ إن شاء الله (😊)